Variable-Domain Displacement Transfer Functions for Converting Surface Strains into Deflections for Structural Deformed Shape Predictions

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ABSTRACT

Variable-Domain Displacement Transfer Functions were formulated for shape predictions of complex wing structures, for which surface strain-sensing stations must be properly distributed to avoid jointed junctures, and must be increased in the high strain gradient region. Each embedded beam (depth-wise cross section of structure along a surface strain-sensing line) was discretized into small variable domains. Thus, the surface strain distribution can be described with a piecewise linear or a piecewise nonlinear function. Through discretization, the embedded beam curvature equation can be piece-wisely integrated to obtain the Variable-Domain Displacement Transfer Functions (for each embedded beam), which are expressed in terms of geometrical parameters of the embedded beam and the surface strains along the strain-sensing line. By inputting the surface strain data into the Displacement Transfer Functions, slopes and deflections along each embedded beam can be calculated for mapping out overall structural deformed shapes. A long tapered cantilever tubular beam was chosen for shape prediction analysis. The input surface strains were analytically generated from finite-element analysis. The shape prediction accuracies of the Variable-Domain Displacement Transfer Functions were then determined in light of the finite-element generated slopes and deflections, and were found to be comparable to the accuracies of the constant-domain Displacement Transfer Functions.

NOMENCLATURE

\( a \) coefficient of first term of quadratic function, \( \varepsilon(\xi) = a + b \xi + e \xi^2 \), in./in.
\( b \) coefficient of second term of quadratic function, \( \varepsilon(\xi) = a + b \xi + e \xi^2 \), 1/in.
\( c \) depth factor (vertical distance from neutral axis to lower surface of uniform embedded beam), in.
\( c(x) \) depth factor (vertical distance from neutral axis to lower surface of nonuniform embedded beam at \( x \)), in.
\( c_i \) lower depth factor (vertical distance from neutral axis to lower surface of nonuniform embedded beam at \( x = x_i \)), in.
\( \overline{c}_i \) upper depth factor (vertical distance from neutral axis to upper surface of nonuniform embedded beam at \( x = x_i \)), in.
\( c_0 \) value of \( c_i \) at fixed end (embedded beam root), \( x = x_0 = 0 \), in.
\( c_n \) value of \( c_i \) at free end (embedded beam tip), \( x = x_n = l \), in.
DLL design limit load
\( e \) coefficient of third term of quadratic function, \( \varepsilon(\xi) = a + b \xi + e \xi^2 \), 1/in.
\( h_i \) depth of embedded beam at \( x = x_i \).
\( i \) strain-sensing station identification number
\( j \) dummy index
\( k \) dummy index
\( l \) length of embedded beam, in.
\( n \) index for the last span-wise strain-sensing station (or number of strain-sensing domains)
\( R \) radius of curvature, in.
RRF rotated reference frame, aligned with wing’s out-of-plane direction
SPAR Structural Performance And Resizing
\( x, y \) Cartesian coordinates (\( x \) in embedded beam axial direction, \( y \) in lateral direction), in.
\( x_i \) axial coordinate (or symbol) associated with \( i \)-th strain-sensing station, in.
\( y(x) \)  
embedded beam deflection at axial location \( x \), in.

\( y_i \)  
\( \equiv y(x_i) \), embedded beam deflection at axial location, \( x = x_i \), in.

\( (y_i)_{\text{Pred.}} \)  
predicted value of \( y_i \), in.

\( (y_i)_{\text{SPAR}} \)  
value of \( y_i \) calculated from SPAR program, in.

\( \Delta l \)  
\( \equiv (x_i - x_{i-1}) = l/n \), constant-domain length (distance between two adjacent strain-sensing stations \( \{x_{i-1}, x_i\} \), in.

\( (\Delta l)_i \)  
\( \equiv (x_i - x_{i-1}) \), variable-domain length (distance between two adjacent strain-sensing stations \( \{x_{i-1}, x_i\} \), in.

\( \varepsilon(x) \)  
lower surface bending strain at axial location \( x \), in/in

\( \varepsilon_i \)  
\( \equiv \varepsilon(x_i) \), lower surface bending strain at strain-sensing station, \( x_i \), in/in

\( \bar{\varepsilon}_i \)  
upper surface bending strain at strain-sensing cross section, \( x_i \), in/in

\( \theta(x) \)  
embedded-beam slope in reference to \( x \)-system, deg

\( \theta_i \)  
\( \equiv \theta(x_i) \), slope at strain-sensing station, \( x = x_i \), deg

\( (\theta_i)_{\text{Pred.}} \)  
predicted values of \( \theta_i \), deg

\( (\theta_i)_{\text{SPAR}} \)  
SPAR-calculated values of \( \theta_i \), deg

\( \xi \)  
\( \equiv x - x_{i-1} \), local axial coordinate measured from strain-sensing station, \( x_{i-1} \), in.

**INTRODUCTION**

For structural deformed shape predictions using distributed surface strains, the Displacement Transfer Functions (refs. 1–9) are needed to convert the surface strains into out-of-plane deflections for mapping out the overall structural deformed shapes. The surface strain data are to be obtained at the strain-sensing stations discretely distributed along the strain-sensing line lying on the surfaces of a planer structure (for example, an aircraft wing). The depth-wise cross section of the structure along the strain-sensing line can be considered as an embedded beam with width of the strain-sensing line (fig. 1).

In the formulations of earlier Displacement Transfer Functions (refs. 1–9), the embedded beam was first discretized evenly into multiple small domains (constant-domain lengths) with strain-sensing stations located at the domain junctures. Thus, within each small domain, the depth variation of the embedded beam could be described with a linear function, and the surface strain distribution could be described with either linear or nonlinear function. Such a piecewise approach enabled piecewise integrations of the embedded-beam curvature equation (second order differential equation) to yield beam slope and deflection equations in recursive forms. Each set of recursive slope and deflection equations were then combined into a summation-form deflection equation (called the Displacement Transfer Function) for each embedded beam. In the Displacement Transfer Function, the deflections are expressed explicitly in terms of geometrical parameters of the discretized embedded-beam and surface strains along the strain-sensing line. By inputting the surface strains into the Displacement Transfer Function, one can then calculate deflections along the embedded-beam for generating the elastic curve. By using multiple strain-sensing lines, one can then map out all the elastic curves of the multiple embedded beams to construct the overall structural deformed shapes (under bending and torsion) for visual display. The Displacement Transfer Functions combined with a strain-sensing system, thus created a revolutionary new structure-shape-sensing technology, “Method for Real-Time Structure Shape-Sensing,” (U.S. Patent Number 7,520,176, ref. 6), which is very attractive for application to the in-flight deformed shape monitoring of flexible wings and tails of unmanned flight vehicles by the ground-based pilot for maintaining safe flights. In addition, the real-time wing shape monitored could then be input to the aircraft control system for aero-elastic wing shape control.
The shape prediction accuracies of the earlier constant-domain Displacement Transfer Functions (based on piecewise-linear strain representations) (refs. 1–7) were successfully validated analytically using finite-element analyses of different sample structures such as cantilever tubular beams (uniform, tapered, slightly tapered, step-wisely tapered), two-point supported tapered tubular beams, flat panels, and tapered wing boxes (un-swept and swept).

Also, by using piecewise-nonlinear strain representations in the formulations (ref. 8), the shape prediction accuracies could be improved greatly, especially for highly tapered beam structures with rapidly changing strain gradients. The Displacement Transfer Functions were originally formulated for the straight embedded beams. However, with the introduction of empirically established curvature-effect correction terms, those Displacement Transfer Functions could be used for shape predictions of slender curved structures with different arc-angles up to 360 degrees, a complete circle (ref. 10). All the earlier Displacement Transfer Functions (refs. 1–9) were formulated based on uniform discretization of the embedded beam into multiple domains with equal domain lengths.

For the shape predictions of certain long complex wing structures consisting of multiple sections joined together, one must avoid locating the strain-sensing stations right at the jointed junctures, where the strain outputs could be erroneous. Also in the region where the strain gradient changes rapidly, shorter strain-sensing intervals (shorter domains) are needed to improve the shape prediction accuracies. Such situations demand the use of variable-domain lengths for properly locating the strain-sensing stations. Therefore, variable-domain Displacement Transfer Functions are needed for deformed shape calculations of a complex structure for which a non-uniform discretization approach is required.

This technical publication presents the formulations of the variable-domain Displacement Transfer Functions (unevenly distributed strain-sensing stations) by extending the previously developed constant-domain Displacement Transfer Functions (evenly distributed strain-sensing stations) (refs. 1–9) for structural deformed shape predictions.

A long tapered cantilever tubular beam was chosen for the shape prediction accuracy studies of the variable-domain Displacement Transfer Functions. Finite-element analysis was used to analytically calculate the input surface strains, and to generate reference yardsticks (slopes and deflections) for the shape prediction error analysis. The variable-domain Displacement Transfer Functions were found to provide highly accurate shape predictions, similar to the constant-domain Displacement Transfer Functions.

NONUNIFORM AND UNIFORM BEAMS

In the present technical publication, the term “beam” implies the embedded beam, which is defined as the depth-wise cross section of the structure along the strain-sensing line, and different from the isolated classical Euler-Bernoulli beam. The nonuniform beam implies the embedded beam with varying depth. The slightly tapered beam implies the embedded beam with slowly changing depth. The uniform beam implies the embedded beam with constant depth.

BASICS OF CONSTANT-DOMAIN DISPLACEMENT TRANSFER FUNCTIONS

This section reviews the basics of formulations of the earlier Displacement Transfer Functions for constant-domain lengths (refs. 1–9).
Curvature-Strain Equation

Formulations of the earlier constant-domain Displacement Transfer Functions for structure deformed shape predictions stemmed from the integrations of the following geometrically established shifted curvature-strain differential equation for the embedded beam (ref. 9):

\[
\frac{d^2 y}{dx^2} \approx \frac{\varepsilon (x)}{c(x)} = \frac{1}{R}
\]

In equation (1), \(x\) is the undeformed axial coordinate, \(y\) is the lateral deflection, \(\varepsilon (x)\) is bottom-surface bending strain, and \(c(x)\) is the embedded beam depth factor, which is defined as the vertical distance from the neutral axis to the bottom surface of the embedded beam. Equation (1) is the simplified Lagrangian curvature equation, \(1/R(x) = (d^2 y / dx^2) / \sqrt{1 - (dy / dx)^2}\), with the term, \((dy / dx)^2\), neglected. Neglection of the \((dy / dx)^2\) term is equivalent to setting axial displacements to zero (called shifted Lagrangian formulation) (refs. 9, 11, and 12). Keep in mind that equation (1) is referred to as the undeformed \(x\)-coordinate, and is not the simplified form of the classical curvature equation, \(1/R = (d^2 y / dx^2) / \left[ 1 + (dy / dx)^2 \right]^{3/2}\), which is referred to as the deformed \(x\)-coordinate (Eulerian formulation) (refs. 9, 11, and 12).

The seven types of constant-domain Displacement Transfer Functions formulated earlier, based on equation (1), were found to be quite accurate [0.0235–0.3852 percent error range at beam tip] in the shape predictions of a tapered cantilever tubular beam under both small and large bending deformations, even with the free-end deflection reaching as large as 94 percent of the structural length (ref. 9).

Discretizations

To integrate equation (1), the embedded beam of length, \(l\), lying along the strain-sensing line (fig. 1), was first discretized evenly into \(n\) number of small domains, \(x_{i-1} \leq x \leq x_i (i = 1, 2, 3, \ldots, n)\), with constant-domain lengths, \(\Delta l = l / n \equiv (x_i - x_{i-1})\), so that within each small domain, \(x_{i-1} \leq x \leq x_i\), the beam depth factor, \(c(x)\) (vertical distance from neutral axis to lower surface of beam at axial location, \(x\)), could be described with linear functions, and the surface strain, \(\varepsilon (x)\), could be described with either linear or nonlinear function as shown in equations (2)–(4a):

Piecewise-linear depth factor representations (refs. 1–7):

\[
c(x) = c_{i-1} + (c_i - c_{i-1}) \frac{x - x_{i-1}}{\Delta l} ; \quad (x_{i-1} \leq x \leq x_i)
\]

Piecewise-linear strain representations (refs. 1–7):

\[
\varepsilon(x) = \varepsilon_{i-1} + (\varepsilon_i - \varepsilon_{i-1}) \frac{x - x_{i-1}}{\Delta l} ; \quad (x_{i-1} \leq x \leq x_i)
\]
Piecewise-nonlinear strain representations (ref. 8):

\[
\varepsilon(x) = \varepsilon_{i-1} - \frac{3\varepsilon_{i-1} - 4\varepsilon_i + \varepsilon_{i+1}}{2\Delta l}(x-x_{i-1}) + \frac{\varepsilon_{i-1} - 2\varepsilon_i + \varepsilon_{i+1}}{2(\Delta l)^2}(x-x_{i-1})^2 \tag{4a}
\]

\(x_{i-1} \leq x \leq x_i\)

Note that, when \(i = n\), equation (4a) contains a nonexistence (fictitious) strain, \(\varepsilon_{n+1}\), which can be estimated by using the following three-points extrapolation equation:

\[
\varepsilon_{n+1} = \varepsilon_{n-2} - 3\varepsilon_{n-1} + 3\varepsilon_n \tag{4b}
\]

Equation (4b) was obtained from equation (4a) by setting \(i = n - 1\) and setting the term \((x-x_{i-1})\) to \((x_{n+1} - x_{n-1}) = 3\Delta l\) in the region \((x_{n-1} \leq x \leq x_{n+1})\).

The piecewise approach enabled the piecewise integrations of the curvature-strain differential equation (1) to yield closed-form beam slope and deflection equations in recursive forms. The recursive slope and deflection equations were then combined into one deflection equation in dual summation forms (called the constant-domain Displacement Transfer Function), which are expressed in terms of embedded beam geometrical parameters and surface strains obtained at evenly spaced strain-sensing stations (at domain junctures) along the strain-sensing line. By inputting the surface strain data into the Displacement Transfer Function, one can then calculate the out-of-plane deflections along each embedded beam. By using multiple strain sensing lines, one can then map out the overall structural deformed shapes (under bending and torsion) for visual display.

LISTS OF CONSTANT-DOMAIN DISPLACEMENT TRANSFER FUNCTIONS

This section lists previously formulated key Displacement Transfer Functions for constant-domain lengths, \(\Delta l = l/n\) (uniform discretization) (refs. 1–9). Those constant-domain Displacement Transfer Functions can then be used as a basis to formulate the variable-domain lengths Displacement Transfer Functions (nonuniform discretization) presented in the subsequent sections. Depending on the structural types, one can choose a suitable Displacement Transfer Function from the following list for shape predictions.

Based on Piecewise-Linear Strain Representations

The following two sets of slope and deflection equations were formulated using piecewise-linear functions [eqs. (2) and (3)] to describe the distributions of both depth factor and surface strain, \(\{c(x), \varepsilon(x)\}\).

1. Nonuniform Embedded Beams (Linear Strain) \((c_{i-1} \neq c_i)\) (refs. 1, 3)

Slope equation:

\[
\tan \theta_i = \Delta l \left[ \frac{\varepsilon_{i-1} - \varepsilon_i - \varepsilon_i c_i - c_i c_{i+1}}{c_{i-1} - c_i} \log \frac{c_i}{c_{i+1}} \right] + \tan \theta_{i-1} \quad ; \quad (i = 1,2,3,\ldots,n) \tag{5a}
\]
Deflection equations:

a. In recursive form:

\[ y_i = (\Delta l)^2 \left[ \frac{\varepsilon_{i-1} - \varepsilon_i}{2(c_{i-1} - c_i)} - \frac{\varepsilon_{i-1} c_i - \varepsilon_i c_{i-1}}{(c_{i-1} - c_i)^3} \left( c_i \log \frac{c_i}{c_{i-1}} + (c_{i-1} - c_i) \right) \right] + y_{i-1} + \Delta l \tan \theta_{i-1} \quad (5b) \]

\[ (i = 1, 2, 3, \ldots, n) \]

b. In dual summation form:

\[ y_i = (\Delta l)^2 \sum_{j=1}^{i} \left[ \frac{\varepsilon_{j-1} - \varepsilon_j}{2(c_{j-1} - c_j)} - \frac{\varepsilon_{j-1} c_j - \varepsilon_j c_{j-1}}{(c_{j-1} - c_j)^3} \left( c_j \log \frac{c_j}{c_{j-1}} + (c_{j-1} - c_j) \right) \right] \]

\[ + (\Delta l)^2 \sum_{j=1}^{i-1} \left[ (i - j) \left( \frac{\varepsilon_{j-1} - \varepsilon_j}{c_{j-1} - c_j} + \varepsilon_{j-1} c_j - \varepsilon_j c_{j-1} \right) \log \frac{c_j}{c_{j-1}} \right] + y_0 + (i) \Delta l \tan \theta_0 \]

\[ \text{Contributions from deflection terms} \]

\[ \text{Contribution from slope terms} \]

\[ (i = 1, 2, 3, \ldots, n) \]

Equations (5) are called the Constant-Domain Nonuniform Displacement Transfer Functions for a nonuniform embedded beam based on piecewise-linear strain representations. The term “Displacement Transfer Function” was used because equation (5c) transforms the surface strains into out-of-plane deflections for plotting the deformed shape of the embedded beam.

Note that equations (5) are not applicable to the limit case of uniform beam \((c_{i-1} = c_i = c)\) because of mathematical indeterminacy \((0/0)\) caused by \(\log(c_i/c_{i-1}) \to 0\) and \((c_i - c_{i-1}) \to 0\). However, by expanding the logarithmic terms in equations (5) in series forms in the neighborhood of \((c_{i-1} / c_i) \to 1\), the factor \((c_i - c_{i-1})\) in the denominators can be eliminated (refs. 1, 3). The resulting Log-expanded equations (6) listed below are for slightly nonuniform embedded beams \([(c_{i-1} / c_i) \to 1]\), and applicable to the limit case of uniform beams \((c_{i-1} = c_i = c)\).

2. Slightly Nonuniform Embedded Beams (Linear Strain) \([(c_{i-1} / c_i) \to 1]\) (refs. 1, 3)

Log-expanded slope equation:

\[ \tan \theta_i = \frac{\Delta l}{2c_{i-1}} \left[ 2 - \frac{c_i}{c_{i-1}} \right] \varepsilon_{i-1} + \varepsilon_i \quad + \tan \theta_{i-1} \quad ; \quad (i = 1, 2, 3, \ldots, n) \quad (6a) \]

Log-expanded deflection equations:

a. In recursive form:

\[ y_i = \frac{(\Delta l)^2}{6c_{i-1}} \left[ 3 - \frac{c_i}{c_{i-1}} \right] \varepsilon_{i-1} + \varepsilon_i \quad + y_{i-1} + \Delta l \tan \theta_{i-1} \quad ; \quad (i = 1, 2, 3, \ldots, n) \quad (6b) \]
b. In dual summation form:

\[
y_j = \frac{(\Delta l)^2}{6} \sum_{j=1}^{i} \left\{ \frac{1}{c_{j-1}} \left[ \frac{3 - c_j}{c_{j-1}} \right] e_{j-1} + e_j + \varepsilon_0 \right\} + \frac{(\Delta l)^2}{2} \sum_{j=1}^{i-1} \left\{ \frac{(i-j)}{c_{j-1}} \left[ \frac{2 - c_j}{c_{j-1}} \right] e_{j-1} + e_j + (i)\Delta l \tan \theta_0 \right\}
\]

Equations (6) are called the Constant-Domain Log-expanded Displacement Transfer Functions for slightly nonlinear (including uniform) embedded beams based on piecewise-linear strain representations.

**Based on Piecewise-Nonlinear Strain Representations**

The following two sets of slope and deflection equations were formulated by using the piecewise-linear function [eq. (2)] to describe the actual distribution of the depth factor, \( c_i(x) \), and the piecewise-nonlinear function [eq. (4)] to describe the actual distribution of the surface strain, \( \varepsilon(x) \).

1. **Nonuniform Beams (Nonlinear Strain) \( (c_i \neq c_j) \) (ref. 8)**

   Improved slope equation:
   \[
   \tan \theta_i = \frac{\Delta l}{2(c_i - c_{i-1})^3} \left[ (2c_i - c_{i-1})(c_i\varepsilon_{i-1} - 2c_{i-1}\varepsilon_i) + c_i c_{i-1} \varepsilon_{i+1} \right] \log_e \frac{c_i}{c_{i-1}} - \frac{\Delta l}{4(c_i - c_{i-1})^2} \left[ (5c_i - 3c_{i-1})\varepsilon_{i-1} - 2(3c_i - c_{i-1})\varepsilon_i + (c_i + c_{i-1})\varepsilon_{i+1} \right] \tan \theta_{i-1}
   \]
   \( (i = 1,2,3,\ldots,n) \)

   Improved deflection equations:
   a. In recursive form:
   \[
   y_i = \frac{(\Delta l)^2}{2(c_i - c_{i-1})^3} \left[ (2c_i - c_{i-1})(c_i\varepsilon_{i-1} - 2c_{i-1}\varepsilon_i) + c_i c_{i-1} \varepsilon_{i+1} \right] \left[ c_i \log_e \frac{c_i}{c_{i-1}} - (c_i - c_{i-1}) \right] - \frac{(\Delta l)^2}{12(c_i - c_{i-1})^2} \left[ (8c_i - 5c_{i-1})\varepsilon_{i-1} - 2(5c_i - 2c_{i-1})\varepsilon_i + (2c_i + c_{i-1})\varepsilon_{i+1} \right] + y_{i-1} + \Delta l \tan \theta_{i-1}
   \]
   \( (i = 1,2,3,\ldots,n) \)
b. In dual summation form:

\[
y_i = (\Delta l)^2 \sum_{j=1}^{i-1} \left\{ \frac{1}{2(c_j - c_{j-1})^3} \left[ (2c_j - c_{j-1})(c_j e_{j-1} - 2c_{j-1} e_j) + c_j c_{j-1} e_{j+1} \right] \times \right.
\]
\[
\times \left[ c_j \log_e \frac{c_j}{c_{j-1}} - (c_j - c_{j-1}) \right] - \frac{1}{12(c_j - c_{j-1})^2} \left[ (8c_j - 5c_{j-1}) e_{j-1} - 2(5c_j - 2c_{j-1}) e_j + (2c_j + c_{j-1}) e_{j+1} \right] \right.
\]
\[
+ (\Delta l)^2 \sum_{j=1}^{i-1} (i - j) \left\{ \frac{1}{2(c_j - c_{j-1})^3} \left[ (2c_j - c_{j-1})(c_j e_{j-1} - 2c_{j-1} e_j) + c_j c_{j-1} e_{j+1} \right] \log_e \frac{c_j}{c_{j-1}} \right.
\]
\[
- \frac{1}{4(c_j - c_{j-1})^2} \left[ (5c_j - 3c_{j-1}) e_{j-1} - 2(3c_j - c_{j-1}) e_j + (c_j + c_{j-1}) e_{j+1} \right] \right\}
\]
\[
+ (i) \Delta l \tan \theta_i
\]

Equations (7) are called the Constant-Domain Improved Displacement Transfer Functions for nonuniform embedded beams based on piecewise-nonlinear strain representations.

Equations (7) are not applicable to the limit case of uniform beam \((c_{i-1} = c_i = c)\) because of mathematical indeterminacy (that is, 0/0). To handle the limit case of uniform beams \((c_{i-1} = c_i = c)\), the logarithmic terms in equations (7) could be expanded in series forms in the neighborhood of \((c_{i-1} / c_i) \rightarrow 1\) (ref. 8), and obtain the following Log-expanded equations (8a), (8b), and (8c) for slightly nonuniform beams, and which are applicable to the limit case of uniform embedded beams \((c_{i-1} = c_i = c)\).

2. Slightly Nonuniform Embedded Beams (Nonlinear Strain) \([(c_{i-1} / c_i) \rightarrow 1]\) (ref. 8)

Log-expanded slope equation:

\[
\tan \theta_i = \frac{\Delta l}{12 c_{i-1}^3} \left[ 5c_{i-1}^3 + 4(c_i - c_{i-1})^2 \right] e_{i-1} - 8c_{i-1}(c_i - 2c_{i-1}) e_i + c_{i-1}(2c_i - 3c_{i-1}) e_{i+1} \right] + \tan \theta_{i-1}
\]

\((i = 1, 2, 3, ..., n)\)

Log-expanded deflection equations:

a. In recursive form:

\[
y_i = \frac{(\Delta l)^2}{24 c_{i-1}^4} \left[ 7c_{i-1}^3 + (c_i - c_{i-1})(8c_i c_{i-1} - 3c_i^2) - 3(c_i^3 - c_{i-1}^3) \right] e_{i-1}
\]
\[
+ 2c_{i-1}^3 \left[ (3c_i^2 + (c_i - c_{i-1})(3c_i - 8c_{i-1}) \right] e_i - c_{i-1} \left[ c_i^2 + 2(c_i - c_{i-1})^2 \right] e_{i+1} \right] + y_{i-1} + \Delta l \tan \theta_{i-1}
\]

\((i = 1, 2, 3, ..., n)\)

b. In dual summation form:
Equations (8) are called the Constant-Domain Log-expanded Displacement Transfer Functions for slightly nonuniform (including uniform) embedded beams based on piecewise-nonlinear strain representations.

**FORMULATION OF VARIABLE-DOMAIN DISPLACEMENT TRANSFER FUNCTIONS**

This section describes the formulations of the Displacement Transfer Functions for variable-domain lengths (nonuniform discretization cases). Let the embedded beam be discretized into \( n \) domains with variable-domain length, \((\Delta l)_i \equiv x_i - x_{i-1}\) \((i = 1, 2, ..., n)\) \((\Delta l \) in figure 1 replaced with \((\Delta l)_i\)], then the beam depth factor, \(c(x)\), within each small domain, \(x_{i-1} \leq x \leq x_i\), between the two adjacent strain-sensing stations, \(\{x_{i-1}, x_i\}\), can be represented with the piecewise-linear function, and the surface strain, \(\varepsilon(x)\), represented with either the piecewise-linear or the piecewise-nonlinear function (quadratic function).

Piecewise-linear depth factor representation as shown in equation (9):

\[
c(x) = c_{i-1} + (c_i - c_{i-1}) \frac{x - x_{i-1}}{(\Delta l)_i} \quad ; \quad (x_{i-1} \leq x \leq x_i)
\]

Piecewise-linear strain representation as shown in equation (10):

\[
\varepsilon(x) = \varepsilon_{i-1} + (\varepsilon_i - \varepsilon_{i-1}) \frac{x - x_{i-1}}{(\Delta l)_i} \quad ; \quad (x_{i-1} \leq x \leq x_i)
\]
Piecewise-nonlinear strain representations (fig. 2) (see Appendix A for derivation):

\[
\varepsilon(x) = \varepsilon_{i-1} - \frac{(\Delta l)_{i+1} [2(\Delta l)_i + (\Delta l)_{i+1}] \varepsilon_{i-1} - [(\Delta l)_i + (\Delta l)_{i+1}]^2 \varepsilon_i + (\Delta l)^2 \varepsilon_{i+1}}{(\Delta l)_i ((\Delta l)_i + (\Delta l)_{i+1})} (x - x_{i-1})
\]

\[
\quad + \frac{(\Delta l)_{i+1} \varepsilon_{i-1} - [(\Delta l)_i + (\Delta l)_{i+1}] \varepsilon_i + (\Delta l)_i \varepsilon_{i+1}}{(\Delta l)_i ((\Delta l)_i + (\Delta l)_{i+1})} (x - x_{i-1})^2
\]

\[
(x_{i-1} \leq x \leq x_i)
\]

When \( i = n \), equation (11) contains a nonexistence (fictitious) strain, \( \varepsilon_{n+1} \). By setting \( i = n - 1 \) and setting the term \((x - x_{i-1})\) to \((x_{n+1} - x_{n-1})\) \(= [(\Delta l)_{n-1} + (\Delta l)_n + (\Delta l)_{n+1}] \) in the region \((x_{n-1} \leq x \leq x_{n+1})\), equation (11) becomes the following three-points extrapolation formula for the calculation of the fictitious strain, \( \varepsilon_{n+1} \):

\[
\varepsilon_{n+1} = \varepsilon_{n-1} - \frac{(\Delta l)_n [2(\Delta l)_{n-1} + (\Delta l)_n] \varepsilon_{n-2} - [(\Delta l)_{n-1} + (\Delta l)_n]^2 \varepsilon_{n-1} + (\Delta l)^2 \varepsilon_n}{(\Delta l)_{n-1} (\Delta l)_n [(\Delta l)_{n-1} + (\Delta l)_n]} \times
\]

\[
\quad \times [(\Delta l)_{n-1} + (\Delta l)_n + (\Delta l)_{n+1}]
\]

\[
\quad + \frac{(\Delta l)_n \varepsilon_{n-2} - [(\Delta l)_{n-1} + (\Delta l)_n] \varepsilon_{n-1} + (\Delta l)_{n-1} \varepsilon_n}{(\Delta l)_{n-1} (\Delta l)_n [(\Delta l)_{n-1} + (\Delta l)_n]} [(\Delta l)_{n-1} + (\Delta l)_n + (\Delta l)_{n+1}]
\]

\[
(12)
\]

For a special case of constant-domain lengths, \( (\Delta l)_1 = (\Delta l)_2 = (\Delta l)_3 = \ldots = (\Delta l) \), it is easy to show that equations [(11), and (12)] will degenerate respectively into equations [(4a), and (4b)] (see Appendix A).

**LISTS OF VARIABLE-DOMAIN DISPLACEMENT TRANSFER FUNCTIONS**

The variable-domain slope and deflection equations in recursive forms can be obtained from the constant-domain recursion slope and deflection equations [(5a), and (5b)–(8a), and (8b)] by simply changing the constant-domain length, \( \Delta l(= l/n) \), into variable-domain lengths, \( (\Delta l)_i(= x_i - x_{i-1}) \). However, the derivations of the variable-domain Displacement Transfer Functions (recursive slope and deflection equations combined into dual summation forms) require considerable mathematical manipulations (see Appendix B).

**Based on Piecewise-Linear Strain Representations**

The following two sets of variable-domain slope and deflection equations were formulated by using piecewise-linear functions to describe the piecewise variations of the depth factor and surface strain, \( \{ \varepsilon(x) \} \), [eqs. (9), and (10)].
1. Nonuniform Beams (Linear Strain) \([(\Delta l)_{i-1} \neq (\Delta l)_i; (c_{i-1} \neq c_i)\)]

Slope equation [see eq. (5a)]:

\[
\tan \theta_i = (\Delta l) \left[ \frac{\varepsilon_{i-1} - \varepsilon_i + c_{i-1} \varepsilon_i c_{i-1} - c_i \log \frac{c_i}{c_{i-1}}}{c_{i-1} - c_i} \right] + \tan \theta_{i-1} \\
(i = 1, 2, 3, \ldots, n)
\]

Deflection equations:

a. In recursive form [see eq. (5b)]:

\[
y_i = (\Delta l)_i^2 \left[ \frac{\varepsilon_{i-1} - \varepsilon_i}{2(c_{i-1} - c_i)} - \frac{\varepsilon_{i-1} c_i - \varepsilon_i c_{i-1}}{(c_{i-1} - c_i)^3} \left( c_i \log \frac{c_i}{c_{i-1}} + (c_{i-1} - c_i) \right) \right] + y_{i-1} + (\Delta l)_i \tan \theta_{i-1} \\
(i = 1, 2, 3, \ldots, n)
\]

b. In dual summation form (see Appendix B for derivation):

\[
y_i = \sum_{j=1}^{i} \left( \Delta l \right)_j^2 \left[ \frac{\varepsilon_{j-1} - \varepsilon_j}{2(c_{j-1} - c_j)} - \frac{\varepsilon_{j-1} c_j - \varepsilon_j c_{j-1}}{(c_{j-1} - c_j)^3} \left( c_j \log \frac{c_j}{c_{j-1}} + (c_{j-1} - c_j) \right) \right] + y_0 \\
+ \sum_{j=1}^{i} \left( \Delta l \right)_j \left[ \delta \left( \frac{\varepsilon_{j-1} - \varepsilon_j + c_{j-1} \varepsilon_j c_{j-1} - c_j \log \frac{c_j}{c_{j-1}}}{c_{j-1} - c_j} \right) \right] + \sum_{j=1}^{i} \tan \theta_0 \\
(i = 1, 2, 3, \ldots, n)
\]

Equations (13) are called the Variable-Domain Nonuniform Displacement Transfer Functions for nonuniform embedded beams based on piecewise-linear strain representations. For constant-domain lengths, \((\Delta l)_1 = (\Delta l)_2 = (\Delta l)_3 = \ldots = (\Delta l)\), equation (13c) degenerates into equation (5c). Equations (13) are not applicable to the limit case of uniform beam \((c_{i-1} = c_i = c)\) because of mathematical indeterminacy (that is, \(0/0\)). However, by expanding the logarithmic terms in equations (13) in series forms in the neighborhood of \((c_{i-1} / c_i) \to 1\) (refs. 1, and 3), one can obtain the following Log-expanded equations (14) for slightly nonuniform embedded beams \([(c_{i-1} / c_i) \to 1]\), which are applicable to the limit case of uniform beams \((c_{i-1} = c_i = c)\).

2. Slightly Nonuniform Embedded Beams (Linear Strain) \([(\Delta l)_{i-1} \neq (\Delta l)_i; (c_i / c_{i-1}) \to 1] \)

Slope equations:

\[
\tan \theta_i = \frac{(\Delta l)_i}{2c_{i-1}} \left( 2 - \frac{c_i}{c_{i-1}} \right) \varepsilon_{i-1} + \varepsilon_i \right) + \tan \theta_{i-1} \\
(i = 1, 2, 3, \ldots, n)
\]
Deflection equations:
a. In recursive form:
\[ y_i = \frac{(\Delta l)^2_i}{6c_{i-1}} \left[ \left( 3 - \frac{c_j}{c_{i-1}} \right) e_{j-1} + \varepsilon_i \right] + y_{i-1} + \Delta l \tan \theta_{i-1} \quad ; \quad (i = 1, 2, 3, \ldots, n) \]

b. In dual summation form:
\[ y_i = \frac{1}{6} \sum_{j=1}^{i} \left\{ \frac{(\Delta l)^2_j}{c_{j-1}} \left[ \left( 3 - \frac{c_j}{c_{j-1}} \right) e_{j-1} + \varepsilon_j \right] \right\} + y_0 \\
+ \frac{1}{2} \sum_{j=1}^{i} \left[ \frac{1}{\sum_{k=j+1}^{n} (\Delta l)^2_k} \left[ 2 - \frac{c_j}{c_{j-1}} \right] e_{j-1} + \varepsilon_j \right] + \left[ \sum_{j=1}^{i} (\Delta l)^2_j \right] \tan \theta_i \\
\] 
\[(i = 1, 2, 3, \ldots, n)\]

Equations (14) are the Variable-Domain Log-expanded Displacement Transfer Function for slightly nonuniform (including uniform) embedded beams based on piecewise-linear strain representations. The derivation of equation (14c) is similar to that of equation (13c) (see Appendix B). For constant-domain lengths, \((\Delta l)_1 = (\Delta l)_2 = (\Delta l)_3 = \ldots = (\Delta l)\), equation (14c) degenerates into equation (6c). It must be mentioned that the shape prediction accuracy of equation (14a), (14b), and (14c) was experimentally validated (see section: Experimental Validations of Shape-Prediction Accuracies).

**Based on Piecewise-Nonlinear Strain Representations**

The following two sets of slope and deflection equations were formulated by using the piecewise-nonlinear function to describe the variation of the surface strain, \(\varepsilon(x)\) [eq. (11)], but the piecewise-linear function was used to describe the variation of the depth factor, \(c(x)\) [eq. (9)]. For simplifications, let the coefficients of the second and third terms of equation (11) be represented respectively with symbols \(A_i\) and \(B_i\) as shown in equations (15) and (16) respectively:

\[ A_i = -\frac{(\Delta l)_{i+1} [2(\Delta l)_i + (\Delta l)_{i+1}] e_{i-1} - [(\Delta l)_i + (\Delta l)_{i+1}]^2 e_i + (\Delta l)^2_i e_{i+1}}{(\Delta l)_i (\Delta l)_{i+1} [(\Delta l)_i + (\Delta l)_{i+1}]} \]

\[ B_i = \frac{(\Delta l)_{i+1} e_{i-1} - [(\Delta l)_i + (\Delta l)_{i+1}] e_i + (\Delta l)_i e_{i+1}}{(\Delta l)_i (\Delta l)_{i+1} [(\Delta l)_i + (\Delta l)_{i+1}]} \]

Then, the improved slope and deflection equations can be written as (see Appendix C for derivations):
1. Nonuniform Embedded Beams (Nonlinear Strain) [(Δl) \neq (Δl)_i; (c_{i-1} \neq c_i)]

Improved slope equation:

\[
\tan \theta = \frac{(Δl)_i}{(c_i - c_{i-1})} \left[ \left( c_i - c_{i-1} \right)^2 e_{i-1} - A_i (Δl)_i (c_i - c_{i-1}) + B_i (Δl)_i^2 \right] \log \frac{c_i}{c_{i-1}} + \frac{(Δl)_i}{2(c_i - c_{i-1})} \left[ 2A_i (Δl)_i (c_i - c_{i-1}) + B_i (Δl)_i^2 (c_i - 3c_{i-1}) \right] + \tan \theta_{i-1}
\]

\[(i = 1, 2, 3, \ldots, n)\]

Improved deflection equations:

a. In recursive form:

\[
y_i = \frac{(Δl)_i^2}{(c_i - c_{i-1})^4} \left[ \left( c_i - c_{i-1} \right)^2 e_{i-1} - A_i (Δl)_i (c_i - c_{i-1})c_{i-1} + B_i (Δl)_i^2 c_{i-1}^2 \right] \left[ c_i \log \frac{c_i}{c_{i-1}} - (c_i - c_{i-1}) \right] + \frac{(Δl)_i^2}{6(c_i - c_{i-1})^2} \left[ 3A_i (Δl)_i (c_i - c_{i-1}) + B_i (Δl)_i^2 (c_i - 4c_{i-1}) \right] + y_{i-1} + (Δl)_i \tan \theta_{i-1}
\]

\[(i = 1, 2, 3, \ldots, n)\]

b. In dual summation form:

\[
y_i = \sum_{j=1}^{i} \left[ \frac{(Δl)_j^2}{(c_j - c_{j-1})^4} \left[ \left( c_j - c_{j-1} \right)^2 e_{j-1} - A_j (Δl)_j (c_j - c_{j-1})c_{j-1} + B_j (Δl)_j^2 c_{j-1}^2 \right] \left[ c_j \log \frac{c_j}{c_{j-1}} - (c_j - c_{j-1}) \right] \right]
\]

\[
+ \sum_{j=1}^{i} \sum_{k=j}^{i} \frac{(Δl)_k}{2(c_j - c_{j-1})^2} \left[ 2A_j (Δl)_j (c_j - c_{j-1}) + B_j (Δl)_j^2 (c_j - 3c_{j-1}) \right] \tan \theta_{j-1}
\]

\[(i = 1, 2, 3, \ldots, n)\]

Equations (17) are called the Variable-Domain Improved Displacement Transfer Functions for nonuniform embedded beams based on piecewise-nonlinear strain representations. Derivation of equation (17c) is similar to that of equation (13c) (see Appendix B). For the constant-domain lengths, \((Δl)_1 = (Δl)_2 = (Δl)_3 = \ldots = Δl\), equations (17) will degenerate into equations (7) (see Appendix C).
Equations (17) are not applicable to the limit case of uniform embedded beam \((c_{i-1} = c_i = c)\) because of mathematical indeterminacy (that is, \(0/0\)). Therefore, the logarithmic terms in equations (17) were expanded in series forms in the neighborhood of \((c_{i-1}/c_i) \to 1\), to yield the following Log-expanded equations (18) for slightly nonuniform embedded beams \([(c_{i-1}/c_i) \to 1]\), applicable to the limit case of uniform beams \((c_{i-1} = c_i = c)\) (see derivations in Appendix D).

2. Slightly Nonuniform Embedded Beams (Nonlinear Strain) \([(\Delta l)_{i-1} \neq (\Delta l)_i; (c_i / c_{i-1}) \to 1]\)

Log-expanded slope equation:

\[
\tan \theta_i = \frac{(\Delta l)_i}{6c_{i-1}^3} \left[ \left( 6c_{i-1}^3 + (c_i - c_{i-1})(2c_i - 5c_{i-1}) \right) e_{i-1} \right] + \tan \theta_{i-1}
\]

\[
\text{(18a)}
\]

\((i = 1, 2, 3, \ldots, n)\)

Log-expanded deflection equations:

a. In recursive form:

\[
y_i = \frac{(\Delta l)^2_i}{12c_{i-1}^2} \left[ \left( 6c_{i-1}^3 + c_{i-1}(10c_i - 3c_{i-1})(c_i - c_{i-1}) - 3(c_i^3 - c_{i-1}^3) \right) e_{i-1} \right]
\]

\[
+ A_i (\Delta l)_i c_{i-1} \left[ 2c_{i-1}^2 + (3c_i - 4c_{i-1})(c_i - c_{i-1}) \right]
\]

\[
\left[ + B_i (\Delta l)^2_i c_{i-1}^2 \left[ c_{i-1} - 3(c_i - c_{i-1}) \right] \right] + y_{i-1} + (\Delta l)_i \tan \theta_{i-1}
\]

\[
\text{(18b)}
\]

\((i = 1, 2, 3, \ldots, n)\)

b. In dual summation form:

\[
y_i = \frac{1}{12} \sum_{j=1}^{i} \left( \frac{(\Delta l)^2_j}{c_j^{i-1}} \right) \left[ \left( 6c_{j-1}^3 + c_{j-1}(10c_j - 3c_{j-1})(c_j - c_{j-1}) - 3(c_j^3 - c_{j-1}^3) \right) e_{j-1} \right]
\]

\[
+ A_i (\Delta l)_i c_{i-1} \left[ 2c_{i-1}^2 + c_{i-1}(c_j - c_{j-1})(3c_j - 4c_{j-1}) \right]
\]

\[
\left[ + B_i (\Delta l)^2_i c_{i-1}^2 \left[ c_{i-1} - 3(c_j - c_{j-1}) \right] \right] + y_0
\]

\[
\text{(18c)}
\]

\((i = 1, 2, 3, \ldots, n)\)

Equation (18) are called the Variable-Domain Log-expanded Displacement Transfer Functions for slightly nonuniform (including uniform) embedded beams based on piecewise-nonlinear strain representations. Derivation of equation (18c) is similar to that of equation (13c) (see Appendix B). For constant-domain lengths, \((\Delta l)_1 = (\Delta l)_2 = (\Delta l)_3 = \ldots = \Delta l\), equation (18c) degenerates into equation (8c) (see Appendices D, and E).
CHARACTERISTICS OF DISPLACEMENT TRANSFER FUNCTIONS

The Displacement Transfer Functions listed above are purely geometrical in nature, relating surface strains and depth factors to out-of-plane deflections, and contain no material/structural properties. Thus, in the shape predictions using the Displacement Transfer Functions, no knowledge of material/structural properties is required. In fact, the effects of material/structural properties are felt only by the surface strains and influence their outputs.

Another key characteristic of the Displacement Transfer Functions is that the deflection, \( y_i \), at the strain-sensing station, \( x_i \), is expressed in terms of the inboard surface strains, \( (\varepsilon_0, \varepsilon_1, \varepsilon_2, \ldots, \varepsilon_i) \), obtained at all the inboard strain-sensing stations, \( (x_0, x_1, x_2, \ldots, x_i) \), including the strain, \( \varepsilon_i \), at the current strain-sensing station, \( x_i \), where \( y_i \) is calculated. The outboard surface strains, \( (\varepsilon_{i+1}, \varepsilon_{i+2}, \varepsilon_{i+3}, \ldots, \varepsilon_n) \), are not needed in the calculations of \( y_i \).

DETERMINATIONS OF DEPTH FACTORS

To use the Displacement Transfer Functions to convert surface strains into deflections, the depth factors, \( c_i (i = 1, 2, 3, \ldots, n) \), of the embedded beam are needed in the deflection calculations. To determine the depth factors, \( c_i \) (that is, locating the neutral axis), one can use two strain-sensing lines, each of which are lying respectively on the lower and upper surfaces of the embedded beam. The lower and upper surface strains, \( \{\varepsilon_i, \bar{\varepsilon}_i\} (i = 1, 2, 3, \ldots, n) \), can then be used to determine the lower and upper depth factors, \( \{c_i, \bar{c}_i\} \).

As shown in figure 3, \( R \) is the radius of curvature of the neutral axis of the embedded beam under upward bending; \( A_0 B_0 \) is the undeformed segment length along the neutral axis; \( CD \) is the elongation of the lower surface from un-deformed length, \( AB (= A_0 B_0) \) to deformed length, \( AC \); \( -\Delta D \) is the contraction of the upper surface from undeformed length, \( AB (= A_0 B_0) \) to deformed length, \( AC \). Then, through geometrical similarities of the curved triangles, \( \Delta OAC, \Delta B_0 BC, \Delta B_0 \bar{B} \bar{C} \) (fig. 3), the lower and upper surface strains, \( \{\varepsilon_i, \bar{\varepsilon}_i\} \), can be related respectively to the lower and upper depth factors, \( \{c_i, \bar{c}_i\} \), as:

\[
\varepsilon_i = \frac{BC}{AB} = \frac{BC}{A_0 B_0} = \frac{c_i}{R} \quad \text{(tension)} \quad ; \quad \bar{\varepsilon}_i = \frac{-\overline{BC}}{AB} = \frac{-\overline{BC}}{A_0 B_0} = -\frac{\bar{c}_i}{R} \quad \text{(compression)} \quad (19)
\]

Keep in mind that in upward or downward bending, the signs of the lower and upper surface strains, \( \{\varepsilon_i, \bar{\varepsilon}_i\} \), are always opposite. By eliminating \( R \) in equation (19), the two depth factors, \( \{c_i, \bar{c}_i\} \), can be related as:

\[
\bar{c}_i = -\frac{\bar{\varepsilon}_i}{\varepsilon_i} c_i \quad ; \quad (i = 1, 2, 3, \ldots, n) \quad (20)
\]
The depth of the embedded beam, \( h_i \), at strain-sensing station, \( x_i \), is given by:

\[
h_i = c_i + \bar{c}_i; \quad (i = 1,2,3,\ldots,n) \tag{21}
\]

In view of equations (20) and (21), one can establish the depth factor equations in the forms:

\[
c_i = \frac{\varepsilon_i}{\varepsilon_i - \bar{\varepsilon}_i} h_i; \quad \bar{c}_i = -\frac{\bar{\varepsilon}_i}{\varepsilon_i - \bar{\varepsilon}_i} h_i; \quad (i = 1,2,3,\ldots,n) \tag{22}
\]

Equation (22) can also be used for downward bending, and is applicable to both solid beam cross sections, and hollow beam cross sections (for example, aircraft wing cross sections).

If \( \{ \varepsilon_i, \bar{\varepsilon}_i \} \) have the same magnitudes (opposite signs), then equation (22) gives \( c_i = \bar{c}_i = h_i/2 \), indicating that the neutral axis is located at the beam half depth. Equation (22) was used with great success in the shape predictions of both Ikhana (General Atomics Aeronautical Systems Inc., San Diego, California) composite wings (66-ft wingspan) (ref. 3) and Global Observer (AeroVironment Inc., Monrovia, California) composite wings (175-ft wingspan) (ref. 13), for which the locations of neutral axes were unknown. Therefore, equation (22) had to be used to calculate the depth factors, \( \{ c_i, \bar{c}_i \} \), for each embedded beam using the associated surface strains, \( \{ \varepsilon_i, \bar{\varepsilon}_i \} \).

**STRUCTURE USED FOR SHAPE PREDICTION ACCURACY STUDIES**

To compare the shape-prediction accuracies of the newly formulated variable-domain Displacement Transfer Function with the earlier constant-domain Displacement Transfer Functions, a sample structure was needed. The structure chosen for the prediction accuracy analysis was an aluminum tapered cantilever tubular beam with geometries listed in table 1.

<table>
<thead>
<tr>
<th>( l ), in.</th>
<th>( t ), in.</th>
<th>( c_0 ), in.</th>
<th>( c_n ), in.</th>
<th>( \alpha = \tan^{-1}[(c_0 - c_n)/l] ), deg.</th>
</tr>
</thead>
<tbody>
<tr>
<td>(Length)</td>
<td>(Wall thickness)</td>
<td>(Root depth factor)</td>
<td>(Tip depth factor)</td>
<td>(Taper angle)</td>
</tr>
<tr>
<td>300</td>
<td>0.02296</td>
<td>4</td>
<td>1</td>
<td>0.5729</td>
</tr>
</tbody>
</table>

The embedded beam (depth-wise structural cross section along the strain-sensing line) shown in figure 1 can also represent the embedded beam of the tapered cantilever tubular beam. The embedded beam for the tubular beam is defined as the span-wise vertical cross section along the strain-sensing line which is coincidental with the bottom generatrix (a straight line for generating tubular surface through sidewise circular motion). Because the depth factor, \( c_i(i = 1,2,3,\ldots,n) \) (fig. 1), is the known local radius, only one strain-sensing line is needed to obtain surface strains, \( \varepsilon_i(i = 1,2,3,\ldots,n) \) (fig. 1) for shape predictions. An upward point load of \( P = 300 \text{ lb} \) (or \( P = 600 \text{ lb} \)) was applied at the beam free end.

In the present technical publication, sixteen \( (n = 16) \) strain-sensing domains (distance between two adjacent strain-sensing stations) were used for the constant-domain cases with equal domain lengths of \( \Delta l = l/n = 300/16 = 18.75 \text{ in.} \). To create variable-domain cases, the strain-sensing stations at \( i = 4, 8, \text{ and } 12 \) of constant-domain cases were removed to double the domain length to \( (\Delta l)_i = 2 \times 18.75 = 37.5 \text{ in.} \) in those three regions. The rest of the domain lengths remain unchanged [that is, \( (\Delta l)_i = 18.75 \text{ in.} \)].
ANALYTICAL SHAPE PREDICTIONS

The shape prediction analysis in the present technical publication report is called analytical shape prediction analysis. Namely, instead of using experimentally measured surface strains, the Structural Performance And Resizing (SPAR) finite-element computer program (ref. 14) was used to analytically calculate the surface strains for input to the Displacement Transfer Functions for shape calculations. Also, for estimations of shape prediction errors of different Displacement Transfer Functions, the SPAR-generated slopes and deflections were used as reference yardsticks.

FINITE ELEMENT ANALYSIS

The purpose of the finite-element analysis is to generate input surface strains, and to calculate the slopes and deflections, which were used as the reference yardsticks in the shape prediction error analysis.

Finite-Element Model

The SPAR finite-element model generated for the tapered cantilever tubular beam has the following numbers of nodes and elements:

- 3673 nodes = 3636 nodes for the tube wall (101 axial nodes × 36 circumferential nodes) + 37 nodes on the beam-tip disk
- 3636 four-node elements = 3600 elements for the tube wall + 36 elements for beam-tip disk outer region
- 36 three-node elements for beam-tip disk central region

The upward point load of \( P = 300 \text{ lb} \) (or \( P = 600 \text{ lb} \)) was applied at the beam-tip disk central node of the SPAR model.

Figure 4 shows the undeformed and deformed shapes of the SPAR model generated for the tapered cantilever tubular beam subjected to beam-tip load of \( P = 300 \text{ lb} \) (or \( P = 600 \text{ lb} \)). Note that for the loading case of \( P = 300 \text{ lb} \), the beam-tip deflection reached \( y_n \approx 141.6 \text{ in.} \), and the beam-tip slope reached as large as \( \theta_n \approx 48 \text{ deg} \). For the doubled loading case of \( P = 600 \text{ lb} \), the beam-tip deflection was doubled to \( y_n \approx 283.2 \text{ in.} \) (because of linear elasticity), however, the beam-tip slope was not doubled, but only increased by 1.375 times to \( \theta_n \approx 66 \text{ deg} \) (=1.375×48 deg).

It must be mentioned that the SPAR model has 100 elements in the axial direction (fig. 3). For \( n = 16 \) strain-sensing domains, the even number of strain sensing stations can be either coincidental with SPAR nodes or at the midpoints between two adjacent SPAR axial nodes. However, the odd numbers of strain-sensing stations are not precisely at the SPAR nodes. Therefore, the shape prediction errors were based on the actual prediction errors at the even numbers of strain-sensing stations.

Analytical Surface Strains

In the present paper, the surface strains, \( \varepsilon_i \ (i = 0,1,2,3,\ldots,n) \), for inputs to the Displacement Transfer Functions were calculated analytically (not actually measured) by using the SPAR program. Namely, finite elements were used to simulate actual strain gages. The surface bending strains, \( \varepsilon_i \ (i = 0,1,2,3,\ldots,n) \), at the \( i \)-th strain-sensing stations (fig. 1) were generated by converting the nodal or element axial stresses into axial strains, \( \varepsilon_i \) (bending strains), through stress-strain relationship.
It must be mentioned that the surface strain, $\varepsilon_i$, can also be obtained from the span-wise nodal displacement differentials of the SPAR elements. However, it was found that in the beam-tip regions of the highly bent tapered beams (large deformations), the axial strains became negative because the output nodal displacements were the projected displacements along the original beam axis, and not along the deformed beam axis to reflect true span-wise displacements. Therefore, this displacement method was not used.

Figure 5 shows the plots of SPAR-generated surface strains for the two loading cases, $P = \{300, 600\}$ lb. Note that the strain level doubled when the applied load was increased from $P = 300$ lb to $P = 600$ lb because of linear elasticity used in the SPAR program. Note that the surface strains for the two loading cases increased practically linearly from the fixed ends and reached their respective peaks in the outboard region, and then decreased rapidly toward the beam tip (because of decreasing depth factor), and finally reached small nonzero values at the beam tip. Theoretically the beam-tip strains should be zero. Because of finite length, the beam-tip finite element (simulated strain gage), which has one end at the beam tip, but the other end is slightly off from the beam tip, gave small nonzero strains. By reducing the element sizes, the small nonzero beam-tip strains could be reduced. Keep in mind that if an actual strain gage is installed at the beam tip, the strain output could also be nonzero because of finite length of the strain gage.

**Criteria of Prediction Errors**

The SPAR-calculated surface strain data of figure 5 were then input to the Displacement Transfer Functions (constant-domain and variable-domain cases) to calculate the slopes and deflections. The shape prediction errors of the Displacement Transfer Functions were estimated by comparison with the SPAR-calculated slopes and deflections, which were used as reference yardsticks. The prediction errors are then defined as the percent of normalized differences (signs neglected) between the predicted slopes and deflections, $\{(\theta)_{\text{Pred.}}, (y_i)_{\text{Pred.}}\}$, and the corresponding SPAR-generated slopes and deflections, $\{(\theta)_{\text{SPAR}}, (y_i)_{\text{SPAR}}\}$. Namely,

$$\text{Slope (} \theta_i \text{) prediction error} = \left| \frac{(\theta)_{\text{Pred.}} - (\theta)_{\text{SPAR}}}{(\theta)_{\text{SPAR}}} \right| \times 100\% \quad ; \quad (i = 1, 2, 3, \ldots, n) \quad (23)$$

$$\text{Deflection (} y_i \text{) prediction error} = \left| \frac{(y_i)_{\text{Pred.}} - (y_i)_{\text{SPAR}}}{(y_i)_{\text{SPAR}}} \right| \times 100\% \quad ; \quad (i = 1, 2, 3, \ldots, n) \quad (24)$$

**SELECTED DISPLACEMENT TRANSFER FUNCTIONS**

Because the cantilever tubular beam is highly tapered, the following four Displacement Transfer Functions for nonuniform beams were chosen in the prediction accuracy analysis.

Based on piecewise-linear strain representations:

1. Equations (5) — constant-domain Nonuniform Displacement Transfer Functions formulated for nonuniform beams based on piecewise-linear strain representations.
2. Equations (13) — variable-domain Nonuniform Displacement Transfer Functions formulated for nonuniform beams based on piecewise-linear strain representations.

Based on piecewise-nonlinear strain representations:

3. Equations (7) — constant-domain Improved Displacement Transfer Functions formulated for nonuniform beams based on piecewise-nonlinear strain representations.
4. Equations (17) — variable-domain Improved Displacement Transfer Functions formulated for nonuniform beams based on piecewise-nonlinear strain representations.
The above four Displacement Transfer Functions were used to calculate both slopes and deflections for comparison with the corresponding SPAR-calculated values.

**NUMERICAL RESULTS**

The SPAR-generated surface strain data of figure 5 were then input into equations (5a), (5b), (5c), (13a), (13b), and (13c) (piecewise-linear strain representations), and equations (7a), (7b), (7c), (17a), (17b), and (17c) (piecewise-nonlinear strain representations) for the calculations of both slopes and deflections. The slope and deflection prediction errors were then calculated respectively from the prediction error equations (23), and (24).

**Calculated Slope Data**

The slopes, \( \theta_{\text{f}} \), calculated from the slope equations (5a), and (13a) (based on piecewise-linear strain representations) and slope equations (7a), and (17a) (based on piecewise-nonlinear strain representations for the two loading cases, \( P = \{300, 600\} \) lb, are listed below. The percent slope prediction errors were calculated from the slope prediction error equation (23).

*For Piecewise-Linear Strain Representations:*

The slope prediction errors based on piecewise-linear strain representations are listed in tables 2 and 3 for loading cases of \( P = \{300, 600\} \) lb.
Table 2. Comparisons of slopes, $\theta_i$, calculated from SPAR and from constant and variable-domain slope equations [(5a), (13a)] (piecewise-linear strain representations); tapered cantilever tubular beam ($l = 300$ in., $c_0 = 4$ in., $c_n = 1$ in.) subjected to tip load of $P = 300$ lb; $n = 16$, $\Delta l = 18.75$ in., fine ($\Delta l_f = 18.75$ in., coarse ($\Delta l_c = 37.50$ in.

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* 30-percent increase from constant-domain case.
Table 3. Comparisons of slopes, $\theta_i$, calculated from SPAR and from constant and variable-domain non-uniform slope equations [(5a), (13a)] (piecewise-linear strain representations); long tapered cantilever tubular beam ($l = 300$ in., $c_0 = 4$ in., $c_n = 1$ in.) subjected to tip load of $P = 600$ lb. $n = 16$, $\Delta l = 18.75$ in.; fine ($\Delta l_f = 18.75$ in.), coarse ($\Delta l_c = 37.50$ in.).

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* 30-percent increase from constant-domain case.

Note from tables 2, and 3 (piecewise-linear strain representations) that, by removing three strain-sensing stations at $i = 4, 8, 12$ to change the constant-domain cases into variable-domain cases, the beam-tip slope ($\theta_n$) prediction errors increased from 0.3559 percent to 0.4625 percent (30-percent increase) for loading case of $P = 300$ lb (table 2), and from 0.1954 percent to 0.2540 percent (also a 30-percent increase) for loading cases of $P = 600$ lb (table 3). However, the slope prediction errors are still in the negligible levels. Note also that by increasing the load from $P = 300$ lb to $P = 600$ lb, the beam-tip slope ($\theta_n$) prediction errors decreased from 0.3559 percent (table 2) down to 0.1954% (table 3) (45-percent reduction) for the constant-domain cases, and decreased from 0.4625 percent (table 2) down to 0.2540 percent (table 3) (also a 45-percent reduction) for the variable-domain cases.

**For Piecewise-Nonlinear Strain Representations:**

The slope prediction errors based on piecewise-nonlinear strain representations are listed in tables 4 and 5 for loading cases of $P = \{300, 600\}$ lb.
Table 4. Comparisons of slopes, $\theta_i$, calculated from SPAR and from constant and variable-domain slope equations [(7a), (17a)] (piecewise-nonlinear strain representations); tapered cantilever tubular beam ( $l = 300$ in., $c_0 = 4$ in., $c_n = 1$ in.) subjected to tip load of $P = 300$ lb; $n = 16$, $\Delta l = 18.75$ in., fine $(\Delta l)_i = 18.75$ in., coarse $(\Delta l)_i = 37.50$ in.

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* 41-percent reduction from constant-domain case.
Table 5. Comparisons of slopes, $\theta_i$, calculated from SPAR and from constant and variable-domain slope equations [(7a), (17a)] (piecewise-nonlinear strain representations); tapered cantilever tubular beam ($l = 300$ in., $c_0 = 4$ in., $c_n = 1$ in.) subjected to tip load of $P = 600$ lb; $n = 16$, $\Delta l = 18.75$ in., fine $(\Delta l)_f = 18.75$ in., coarse $(\Delta l)_c = 37.50$ in.

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</tbody>
</table>

* 41-percent deduction from constant-domain case.

Note from tables 4, and 5 (piecewise-nonlinear strain representations) that, by removing three strain-sensing stations at $i = 4, 8, 12$ to change the constant-domain cases into variable-domain cases, the beam-tip slope ($\theta_n$) prediction errors decreased from 0.0479 percent down to 0.0283 percent (41-percent reduction) for the loading case of $P = 300$ lb (table 4), and from 0.0263 percent down to 0.0154 percent (also a 41-percent reduction) for loading case of $P = 600$ lb (table 5) [remember, 30-percent increase for stepwise linear strain representations (tables 2, and 3)].

Also note that by increasing the load from $P = 300$ lb to $P = 600$ lb, the beam-tip slope ($\theta_n$) prediction errors decreased from 0.0479 percent (table 4) down to 0.0263 percent (table 5) (45-percent reduction) for the constant-domain cases, and decreased from 0.0283 percent (table 4) down to 0.0154 percent (table 5) (also a 45-percent reduction) for the variable-domain cases.

It is important to mention that by changing the piecewise-linear strain representations (tables 2, and 3) into piecewise-nonlinear strain representations (tables 4, and 5), the beam-tip slope ($\theta_n$) prediction errors for both loading cases, $P = \{300, 600\}$ lb, were reduced by as large as 87 percent for the constant-domain cases, and by 94 percent for the variable-domain cases. Note from tables 4 and 5 that nonlinear strain representations reducing the number of domains (that is, increasing domain lengths) could somewhat improve the beam-tip slope ($\theta_n$) prediction accuracies. Such domain-length-dependent shape prediction
accuracies of the Displacement Transfer Functions were also observed in the experimental shape-prediction accuracy studies (ref. 13).

**Calculated Slope Curves**

Figures 6 and 7 respectively show the slope ($\theta_i$) curves generated using the slope data of tables 2 and 3 (piecewise-linear strain representations), and tables 4 and 5 (piecewise-nonlinear strain representations). Note from figures 6 and 7 that the slope-curves are slightly s-shaped, and the values of $\theta_i$ did not double when the load was increased from $P = 300$ lb to $P = 600$ lb. Namely, $\theta_i$ is not a linear function of $P$. Because of infinitesimal slope prediction errors, the predicted slope curves for all the constant and variable-domain cases pictorially fell on top of the corresponding SPAR slope curves, showing remarkable accuracies of the Displacement Transfer Functions formulated based on piecewise approaches.

**Calculated Deflection Data**

The deflections, $y_i$, calculated from the deflection equations (5b), (5c), (13b), and (13c) (based on piecewise-linear strain representations) and deflection equations (7b), (7c), (17b), and (17c) (based on piecewise-nonlinear strain representations) for the two loading cases, $P = \{300, 600\}$ lb are listed below. The percent deflection prediction errors were calculated from the deflection prediction error equation (24).

*For Piecewise-Linear Strain Representations:*

The deflections, $y_i$, calculated from the deflection equations (5b), (5c) (13b), and (13c)] (based on piecewise-linear strain representations) for the loading cases, $P = \{300, 600\}$ lb are listed respectively in tables 6, and 7.
Table 6. Comparisons of deflections, $y_i$, calculated from SPAR and from constant and variable-domain deflection equations [(5b, 5c), (13b, 13c)] (piecewise-linear strain representations); tapered cantilever tubular beam ($l = 300$ in., $c_0 = 4$ in., $c_n = 1$ in.) subjected to tip load of $P = 300$ lb, $n = 16$, $\Delta l = 18.75$ in., fine ($\Delta l)_f = 18.75$ in., coarse ($\Delta l)_c = 37.50$ in.

<table>
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<th>Percent error</th>
<th>Variable domains [eqs. (13b,13c)]</th>
<th>Percent error</th>
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* 53-percent increase from constant-domain case.
Table 7. Comparisons of deflections, \( y_i \), calculated from SPAR and from constant and variable-domain deflection equations [(5b, 5c), (13b, 13c)] (piecewise-linear strain representations); tapered cantilever tubular beam \((l = 300 \text{ in.}, c_0 = 4 \text{ in.}, c_n = 1 \text{ in.})\) subjected to tip load of \( P = 600 \text{ lb}, n = 16, \Delta l = 18.75 \text{ in.}\), fine \((\Delta l)_f = 18.75 \text{ in.}\), coarse \((\Delta l)_c = 37.50 \text{ in.}\).

<table>
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<tr>
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<th>SPAR ( y_i ), in.</th>
<th>Constant domain [eqs. (5b,5c)]</th>
<th>Variable domains [eqs. (13b,13c)]</th>
<th>Percent error</th>
<th>Percent error</th>
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* 53-percent increase from constant-domain case.

Note from tables 6, and 7 (piecewise-linear strain representations) that deflections \( y_i \) were doubled when the load was doubled from \( P = 300 \text{ lb} \) to \( P = 600 \text{ lb} \) because \( y_i \) are linear functions of \( P \). Note also that the \( y_i \)-prediction percent errors under the two loading cases, \( P = \{300, 600\} \text{ lb} \), (tables 6, and 7) are identical for both constant and variable-domain cases, and are independent of the loading magnitude of \( P \).

As shown in tables 6, and 7, by removing three strain-sensing stations at \( i = 4, 8, 12 \), to change the constant-domain cases to variable-domain cases, the beam-tip deflection \( y_n \) prediction errors increased from 0.2090 percent (constant-domain cases) to 0.3199 percent (variable-domain cases) (53-percent increase) for both loading cases, \( P = \{300, 600\} \text{ lb} \).

For Piecewise-Nonlinear Strain Representations:

The deflections, \( y_i \), calculated from the deflection equations (7b), (7c) (17b), and (17c) (based on piecewise-nonlinear strain representations) for the loading cases, \( P = \{300, 600\} \text{ lb} \) are listed respectively in tables 8, and 9.
Table 8. Comparisons of deflections, $y_i$, calculated from SPAR and from constant and variable-domain deflection equations [(7b, 7c), (17b, 17c)] (piecewise-nonlinear strain representations); tapered cantilever tubular beam ($l = 300$ in., $c_0 = 4$ in., $c_n = 1$ in.) subjected to tip load of $P = 300$ lb, $n = 16$, fine $(\Delta l)_i = 18.75$ in., coarse $(\Delta l)_i = 37.50$ in.

<table>
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<th>Constant domain [eqs. (7b,7c)] $y_i$, in.</th>
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* 156-percent increase from constant-domain case.
Table 9. Comparisons of deflections, \( y_i \), calculated from SPAR and from constant and variable-domain deflection equations [(7b, 7c), (17b, 17c)] (piecewise-nonlinear strain representations); tapered cantilever tubular beam (\( l = 300 \text{ in.}, c_0 = 4 \text{ in.}, c_n = 1 \text{ in.} \)) subjected to tip load of \( P = 600 \text{ lb}, n = 16 \), fine \( (\Delta l)_f = 18.75 \text{ in.}, \) coarse \( (\Delta l)_c = 37.50 \text{ in.} \).

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<th>Percent error ( y_i ), in.</th>
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* 156-percent increase from constant-domain case.

Note from tables 8 and 9 (piecewise-nonlinear strain representations) that deflections \( (y_i) \) were doubled when the load was doubled from \( P = 300 \text{ lb} \) to \( P = 600 \text{ lb} \) because \( y_i \) are linear functions of \( P \). It is important to note that the \( y_i \)-prediction errors are identical under the two loading cases, \( P = \{300, 600\} \text{ lb} \), (tables 8 and 9) for both constant and variable-domain cases and are, therefore, invariant to the loading magnitude of \( P \).

By removing three strain-sensing stations at \( i = 4, 8, 12 \), to convert the constant-domain cases to variable-domain cases, the beam-tip deflection \( (y_n) \) prediction errors increased from 0.0235 percent (constant-domain cases) to 0.0601 percent (156-percent increase) for both loading cases, \( P = \{300, 600\} \text{ lb} \). [Remember, there is only a 53-percent increase for piecewise-linear strain representations (tables 6, and 7)].

**Calculated Deflection Curves**

Figures 8 and 9 respectively show the deflection \( (y_i) \) curves generated using the deflection data of tables 6, and 7 (piecewise-linear strain representations) and tables 8, and 9 (piecewise-nonlinear strain representations). Note from figures 8 and 9 that the \( y_i \)-curves are shallow bow-shaped, and the values of \( y_i \) were doubled when the load was increased from \( P = 300 \text{ lb} \) to \( P = 600 \text{ lb} \). Namely, \( y_i \) is a linear
function of $P$. Because of infinitesimal deflection prediction errors, the predicted deflection curves for all
the constant and variable-domain cases practically fell on top of the corresponding SPAR deflection curves,
showing amazing high accuracies of the Displacement Transfer Functions formulated based on piecewise
formulations.

SHAPE PREDICTION ERROR ANALYSIS

This section compares the shape prediction errors of constant and variable-domain Displacement
Transfer Functions formulated using linear or nonlinear strain representations. The slope and deflection
prediction errors were calculated respectively from the shape prediction error equations (23) and (24).

Slope Prediction Error Curves

Figures 10 and 11 respectively show the slope ($\theta_l$) prediction error curves for loading cases of
$P = \{300, 600\}$ lb based on the slope data listed in tables 2 and 3 (piecewise-linear strain representations).
The beam-tip slope ($\theta_n$) prediction errors (in percent) are also indicated in figures 10 and 11. Those
$\theta_l$-error curves were plotted by using only the actual prediction error data at the even strain-sensing stations
which are either coincidental with SPAR nodes or at the midpoints between two adjacent SPAR axial nodes.
The larger prediction errors at the odd strain-sensing stations (tables 2 and 3) cannot represent the true
prediction errors because the odd strain-sensing stations are not precisely at the SPAR nodes and, therefore,
were not plotted in figures 10 and 11.

The constant-domain cases and variable-domain cases are shown respectively with solid curves (with
solid circular symbols), and dashed curves (with open square symbols). The larger errors near the inboard
regions are due to small numbers divided by other small numbers. The variable-domain cases have slightly
larger errors in the outboard regions (figs. 10 and 11).

Note from figures 10 and 11 that, doubling the load from $P = 300$ lb to $P = 600$ lb, the $\theta_l$-prediction
errors in the outboard regions came down considerably (45-percent error reduction at the beam tip for both
constant-domain and variable-domain cases).

Figures 12 and 13 respectively show the predicted slope ($\theta_l$) error curves for loading cases of
$P = \{300, 600\}$ lb based on the slope data listed in tables 4 and 5 (piecewise-nonlinear strain
representations). The constant-domain cases and variable-domain cases are shown respectively with solid
curves (with solid circular symbols), and dashed curves (with open square symbols). The amounts of the
beam-tip slope ($\theta_n$) prediction errors are also shown in figures 12 and 13.

Note from figures 12 and 13 that, doubling the load from $P = 300$ lb to $P = 600$ lb, the $\theta_l$-prediction
errors in the outboard regions came down considerably, and at the beam tip, the prediction errors also
decreased by 45 percent for both constant-domain and variable-domain cases.

It is very important to mention that by changing the piecewise-linear strain representations (figs. 10 and
11) into piecewise-nonlinear strain representations (figs. 12 and 13), the beam-tip slope ($\theta_n$) prediction
errors for both loading cases, $P = \{300, 600\}$ lb could be reduced remarkably. Namely, reducing from
{$0.3559$ percent, $0.1954$ percent} (figs. 10 and 11) down respectively to {$0.0479$ percent, $0.0263$ percent}
(87-percent error reduction) (figs. 12 and 13) for the constant-domain cases, and reducing from
{$0.4625$ percent, $0.2540$ percent} (figs. 10 and 11) down respectively to {$0.0283$ percent, $0.0154$ percent}
(94-percent error reduction) (figs. 12 and 13) for the variable-domain cases.
Deflection Prediction Error Curves

Figures 14 and 15 respectively show the predicted deflection ($y_i$) error curves based on the deflection error data at even strain-sensing stations listed in tables 6 and 7 (piecewise-linear strain representations) and tables 8 and 9 (piecewise-nonlinear strain representations). The predicted deflection ($y_i$) error curves are identical for both loading cases, $P = \{300, 600\}$ lb. The constant-domain cases and variable-domain cases are shown respectively with solid curves (with solid circular symbols), and dashed curves (with open square symbols). Again, the large errors (figs. 14 and 15) near the fixed end are due to a small number divided by another small number.

As shown in figures 14 and 15, changing from piecewise-linear strain representations into piecewise-nonlinear strain representations, the $y_i$-prediction errors could be reduced considerably in the outboard region. For example, the beam-tip deflections ($y_n$) prediction errors for both loading cases, $P = \{300, 600\}$ lb could be reduced from 0.2090 percent down to 0.0235 percent (89 percent error reduction) for the constant-domain cases, and from 0.3199 percent down to 0.0601 percent (81 percent error deduction) for the variable-domain cases.

EXPERIMENTAL VALIDATIONS OF SHAPE-PREDICTION ACCURACIES

In the process of writing the present report, large-scale ground loads tests of the Global Observer wing (175-ft. wingspan) were carried out at NASA Dryden (now Armstrong) Flight Research Center Flight Loads Laboratory (Edwards, California) (ref. 14). The experimentally measured strains obtained from the ground tests were then used to validate the Displacement Transfer Functions accuracies. The Global Observer wing is slightly tapered and consisted of several sections joined together. Therefore, the strain-sensing stations had to be properly distributed in order to avoid positioning them at the connection junctures, and thus creating the variable-domain case. The surface strains were measured by using four fiber-optic strain-sensing lines, with two lines on the wing upper surface, and another two lines on the wing lower surface. The whole wing was loaded from 0 percent up to 100 percent Design Limit Load (DLL) at which the wing-tip slope exceeded 20 degrees, and wing-tip deflection reached nearly 14 ft. The variable-domain Displacement Transfer Function (14c), developed for the slightly nonuniform beam, was used for the wing shape calculations. Using the measured surface strain data, the wing deflections were calculated from equation (14c), and were compared with the photogrammetry data, which were used as yardsticks to estimate the shape prediction errors.

Figures 16, and 17 (duplications of figures 11 and 12 of ref. 14) respectively show the plots of the predicted and measured wing deflections along the strain-sensing lines Number 3 and Number 4. Note that the strain data points are not uniformly distributed because of variable-domains. Amazingly, the measured data points practically fell on top of the corresponding predicted deflection curves for the whole range of loading levels (0–100-percent DLL). The wing-tip deflection prediction errors based on Number 3 and Number 4 strain-sensing lines are respectively in the small ranges of (0.09–1.72 percent) and (0.25–1.45 percent) at 100-percent DLL, depending on the strain-sensing domain distances used (ref. 14).

In this ground test, it was found that by increasing the domain length from $(\Delta l)_i = 1$ in. to $(\Delta l)_i = 12$ in., the averaged wing-tip deflection prediction errors at 100-percent DLL could be reduced from 1.92 percent down to 0.27 percent (a remarkable 86-percent error reduction). This particular case certainly violates the principle that reducing the strain-sensing domains (that is, increasing the number of strain-sensing stations) can improve the shape prediction accuracies. Because of noisy strain data, increasing the number of strain-sensing stations caused more errors to be accumulated (fig. 10 of ref. 14).
This large-scale ground test results thus reinforced the previous finite-element analytical validation results (refs. 1–9), and give confidence in the shape prediction accuracies of the Variable-Domains Displacement Transfer Functions.

**FACTORS AFFECTING SHAPE PREDICTION ACCURACIES**

The surprising high accuracies of analytical deformed shape predictions could be attributed to the piecewise (or recursive) formulations of the Displacement Transfer Functions. The major factors affecting the shape prediction accuracies of the Displacement Transfer Functions can be listed as:

1. Domain lengths (that is, intervals of strain-sensing stations along each strain-sensing line),
2. Accuracies of piecewise-linear representations of depth-factor distributions,
3. Accuracies of piecewise-linear or piecewise-nonlinear representations of strain distributions,
4. Accuracies of the input surface strains,
5. Accuracies of depth factors determined from surface strains.

**CONCLUDING REMARKS**

The variable-domain Displacement Transfer Functions were formulated through piecewise integrations of the shifted curvature-strain differential equation. Using a long tapered cantilever tubular beam, the shape prediction accuracies of the variable-domain Displacement Transfer Functions were compared with the shape prediction accuracies of the constant-domain Displacement Transfer Functions formulated earlier.

Finite-element analysis was used to analytically generate surface strains. Also, the finite-element calculated slopes and deflections were used as reference yardsticks in the shape prediction error estimations. Some key results are listed below:

1. The slope and deflection equations in recursive forms developed for the constant-domain cases could be converted to those for the variable-domain cases by simply replacing the constant-domain lengths with variable-domain lengths.
2. The variable-domain Displacement Transfer Functions are highly accurate like the constant-domain Displacement Transfer Functions in shape predictions.
3. The shape prediction errors (slopes and deflections) are in the negligible ranges, with beam-tip slope ($\theta_n$) prediction errors in the ranges of {$(0.0263–0.3559 \, \text{percent})$ and $(0.0154–0.4625 \, \text{percent})$} respectively for the constant-domain and variable-domain cases; and the beam-tip deflection ($y_n$) prediction errors in the ranges of {$(0.0253–0.2090 \, \text{percent})$ and $(0.0601–0.3199 \, \text{percent})$} respectively for the constant-domain and variable-domain cases.
4. Based on either piecewise-linear or piecewise-nonlinear strain representations, the deflection prediction errors of the constant-domain and variable-domain Displacement Transfer Functions are invariant to the loading magnitudes. However, the slope prediction errors decreased with increasing loading magnitude.
5. For piecewise-linear strain representations, changing from constant-domain cases into variable-domain cases, the beam-tip slope and deflection {$(\theta_n, y_n)$} prediction errors increased by {30 percent and 53 percent} respectively, regardless of loading magnitude.
6. For piecewise-nonlinear strain representations, changing from constant-domain cases into variable-domain cases, regardless of loading magnitude, the beam-tip slope ($\theta_n$) prediction errors decreased by 41 percent, however, the beam-tip deflection ($y_n$) prediction errors increased by 156 percent.
7. By changing from the piecewise-linear strain representations to piecewise-nonlinear strain representations, the beam-tip slope ($\theta_n$) prediction errors could be reduced by 87 percent for the constant-domain cases, and reduced by 94 percent for the variable-domain cases.
8. By changing from the piecewise-linear strain representations to piecewise-nonlinear strain representations, the beam-tip deflection ($y_n$) prediction errors could be reduced by 89 percent for the constant-domain cases, and reduced by 81 percent for the variable-domain cases.
Figure 1. Embedded beam (cross section of structure, including a tubular beam) along strain-sensing line with evenly distributed strain-sensing stations.

Figure 2. Piecewise linear and piecewise nonlinear representations of variable domain surface strain distribution.

Piecewise linear strain representation:

$$
\varepsilon(x) = \varepsilon_{i-1} + (\varepsilon_i - \varepsilon_{i-1}) \frac{x-x_{i-1}}{(\Delta l)} \quad (x_{i-1} \leq x \leq x_i)
$$

Piecewise nonlinear strain representation:

$$
\varepsilon(x) = \varepsilon_{i-1} - \frac{(\Delta l)_{i+1} \left[ 2(\Delta l)_i + (\Delta l)_{i+1} \right] \varepsilon_{i-1} - \left[ (\Delta l)_i + (\Delta l)_{i+1} \right]^2 \varepsilon_i + (\Delta l)_i^2 \varepsilon_{i+1}}{(\Delta l)_i (\Delta l)_{i+1} \left[ (\Delta l)_i + (\Delta l)_{i+1} \right]} (x-x_{i-1})
$$

$$
+ \frac{(\Delta l)_{i+1} \varepsilon_{i-1} - \left[ (\Delta l)_i + (\Delta l)_{i+1} \right] \varepsilon_i + (\Delta l)_i \varepsilon_{i+1}}{(\Delta l)_i (\Delta l)_{i+1} \left[ (\Delta l)_i + (\Delta l)_{i+1} \right]} (x-x_{i-1})^2 \quad (x_{i-1} \leq x \leq x_i)
$$
Figure 3. Segment of deformed embedded beam used for geometrically relate lower and upper surface strains, \( \{ \varepsilon_i, \bar{\varepsilon}_i \} \), to lower and upper depth factors, \( \{ c_i, \bar{c}_i \} \).

Figure 4. Large deformations of tapered cantilever tubular beam (SPAR finite-element model) subjected to upward tip load of \( P = 300 \) lb (or \( P = 600 \) lb); \( l = 300 \) in., \( c_0 = 4 \) in., \( c_n = 1 \) in.
Figure 5. SPAR-generated surface strain curves for tapered cantilever tubular beam subjected to upward tip load of \( P = 300 \text{ lb} \) (or \( P = 600 \text{ lb} \)); \( l = 300 \text{ in.} \), \( c_0 = 4 \text{ in.} \), \( c_n = 1 \text{ in.} \).

Figure 6. Slopes, \( \theta_i \), of tapered cantilever tubular beam calculated from constant domain and variable domain slope equations [(5a), and (13a)] (piecewise linear strain representations); \( l = 300 \text{ in.} \), \( c_0 = 4 \text{ in.} \), \( c_n = 1 \text{ in.} \); \( P = \{300, 600\} \text{ lb.} \)
Figure 7. Slopes, $\theta_{ij}$, of tapered cantilever tubular beam calculated from constant domain and variable domain slope equations [(7a), and (17a)] (piecewise nonlinear strain representations); $l = 300$ in., $c_0 = 4$ in., $c_n = 1$ in.; $P = \{300, 600\}$ lb.

Figure 8. Deflections, $y_{ij}$, of tapered cantilever tubular beam calculated from constant domain and variable domain deflection equations [(5b, 5c), (13b), and (13c)] (piecewise linear strain representations); $l = 300$ in., $c_0 = 4$ in., $c_n = 1$ in.; $P = \{300, 600\}$ lb.
Figure 9. Deflections, $y_i$, of tapered cantilever tubular beam calculated from constant domain and variable domain deflection equations [(7b), (7c), (17b), and (17c)] (piecewise nonlinear strain representations); $l = 300$ in., $c_0 = 4$ in., $c_n = 1$ in.; $P = \{300, 600\}$ lb.

Figure 10. Comparison of slope ($\theta_i$) prediction errors of constant domain and variable domain slope equations [(5a), and (13a)] (piecewise linear strain representations); tapered cantilever tubular beam; $l = 300$ in., $c_0 = 4$ in., $c_n = 1$ in.; $P = 300$ lb.
Figure 11. Comparison of slope ($\theta_i$) prediction errors of constant domain and variable domain slope equations [(5a) and (13a)] (piecewise linear strain representations); tapered cantilever tubular beam; $l = 300$ in, $c_0 = 4$ in., $c_n = 1$ in.; $P = 600$ lb.

Figure 12. Comparison of slope ($\theta_i$) prediction errors of constant domain and variable domain slope equations [(7a) and (17a)] (piecewise nonlinear strain representations); tapered cantilever tubular beam; $l = 300$ in, $c_0 = 4$ in., $c_n = 1$ in.; $P = 300$ lb.
Figure 13. Comparison of slope ($\theta_i$) prediction errors of constant domain and variable domain slope equations [(7a), and (17a)] (piecewise nonlinear strain representations); tapered cantilever tubular beam; $l = 300$ in, $c_0 = 4$ in., $c_n = 1$ in.; $P = 600$ lb.

Figure 14. Comparisons of deflection ($y_i$) prediction errors of constant domain and variable domain deflection equations [(5b), (5c), (13b), and (13c)] (piecewise linear strain representations); tapered cantilever tubular beam; $l = 300$ in, $c_0 = 4$ in., $c_n = 1$ in.; $P = \{300, 600\}$ lb.
Figure 15. Comparisons of deflection ($y_i$) prediction errors of constant domain and variable domain deflection equations [(7b), (7c)), (17b), and (17c)] (piecewise linear strain representations); tapered cantilever tubular beam; $l = 300$ in, $c_0 = 4$ in., $c_n = 1$ in.; $P = \{300, 600\}$ lb.

Figure 16. Comparison of predicted and measured wing deflections along No. 3 strain-sensing line for different loading levels (duplication of figure 11, ref. 14).
Figure 17. Comparison of predicted and measured wing deflections along No. 4 strain-sensing line for different loading levels (duplication of figure 12, ref. 14).
APPENDIX A
DERIVATION OF STRAIN FUNCTION FOR VARIABLE DOMAIN PIECEWISE-NONLINEAR STRAIN REPRESENTATIONS

Let $\xi(=x-x_{i-1})$ be the local axial coordinate measured from the strain-sensing station, $x_{i-1}$ (fig. 2), and let the distribution of strain, $\varepsilon(\xi)$, within the two adjacent domains, $x_{i-1} \leq x \leq x_i$ and $x_i \leq x \leq x_{i+1}$, be presented by the following nonlinear function (quadratic equation) (fig. 2):

$$
\varepsilon(\xi) = a + b\xi + c\xi^2 \quad ; \quad \xi = x - x_{i-1} \quad (A1)
$$

where $a$, $b$, and $c$ are coefficients, which are to be determined in terms of three strains, $\{\varepsilon_{i-1}, \varepsilon_i, \varepsilon_{i+1}\}$ respectively at three strain-sensing stations, $\{x_{i-1}, x_i, x_{i+1}\}$.

At $\xi = 0$ (i.e., at strain-sensing station, $x_{i-1}$), we have $\varepsilon(0) = \varepsilon_{i-1}$, and equation (A1) gives:

$$
a = \varepsilon_{i-1} \quad (A2)
$$

At $\xi = (\Delta l)_i$ (that is, at strain-sensing station, $x_i$), we have $\varepsilon[(\Delta l)_i] = \varepsilon_i$, then in view of equation (A2), equation (A1) can be written as equation (A3):

$$
\varepsilon_i = \varepsilon_{i-1} + b(\Delta l)_i + c(\Delta l)_i^2 
$$

which can be rewritten as equation (A4):

$$
b + c(\Delta l)_i = \frac{\varepsilon_i - \varepsilon_{i-1}}{(\Delta l)_i} \quad (A4)
$$

At $\xi = (\Delta l)_i + (\Delta l)_{i+1}$ (that is, at strain-sensing station, $x_{i+1}$), we have $\varepsilon[(\Delta l)_i + (\Delta l)_{i+1}] = \varepsilon_{i+1}$. Then in view of equation (A2), equation (A1) can be written as equation (A5):

$$
\varepsilon_{i+1} = \varepsilon_{i-1} + b[(\Delta l)_i + (\Delta l)_{i+1}] + c[(\Delta l)_i + (\Delta l)_{i+1}]^2 
$$

which can be rewritten as equation (A6):

$$
b + c[(\Delta l)_i + (\Delta l)_{i+1}] = \frac{\varepsilon_{i+1} - \varepsilon_{i-1}}{(\Delta l)_i + (\Delta l)_{i+1}} \quad (A6)
$$

Subtracting equation (A4) from (A6), one can eliminate $b$ and obtain the expression for $e$ as equation (A7):

$$
e(\Delta l)_{i+1} = \frac{\varepsilon_{i+1} - \varepsilon_{i-1}}{(\Delta l)_i + (\Delta l)_{i+1}} - \frac{\varepsilon_i - \varepsilon_{i-1}}{(\Delta l)_i} 
$$

$$
= \frac{(\Delta l)_i(\varepsilon_{i+1} - \varepsilon_{i-1}) - [(\Delta l)_i + (\Delta l)_{i+1}](\varepsilon_i - \varepsilon_{i-1})}{(\Delta l)_i[(\Delta l)_i + (\Delta l)_{i+1}]}
$$

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From equation (A7), \( e \) can be written as:

\[
e = \frac{(\Delta l)_i\epsilon_{i+1} - [(\Delta l)_i + (\Delta l)_{i+1}]\epsilon_i + [(\Delta l)_i + (\Delta l)_{i+1}]\epsilon_{i-1}}{(\Delta l)_i[(\Delta l)_i + (\Delta l)_{i+1}]}
\]  

(A7)

Substitution of equation (A8) into equation (A4) yields equation (A9):

\[
b = \frac{\epsilon_i - \epsilon_{i-1}}{\Delta l}_i - \frac{[(\Delta l)_i + (\Delta l)_{i+1}]\epsilon_i + (\Delta l)_{i+1}\epsilon_{i+1} - [(\Delta l)_i + (\Delta l)_{i+1}]\epsilon_i + (\Delta l)_{i+1}\epsilon_{i+1}}{(\Delta l)_i[(\Delta l)_i + (\Delta l)_{i+1}]}
\]

\[
= \frac{[(\Delta l)_i + (\Delta l)_{i+1}]\epsilon_i - [(\Delta l)_i + (\Delta l)_{i+1}]\epsilon_{i-1} + (\Delta l)_{i+1}[(\Delta l)_i + (\Delta l)_{i+1}]\epsilon_i - (\Delta l)_{i+1}^2\epsilon_{i+1}}{(\Delta l)_i[(\Delta l)_i + (\Delta l)_{i+1}]}
\]

(A9)

After grouping terms, one obtains the expression for \( b \) as equation (A10):

\[
b = \frac{2[(\Delta l)_i + (\Delta l)_{i+1}]\epsilon_i - [(\Delta l)_i + (\Delta l)_{i+1}]^2\epsilon_i + (\Delta l)_{i+1}^2\epsilon_{i+1}}{(\Delta l)_i[(\Delta l)_i + (\Delta l)_{i+1}]}
\]  

(A10)

In view of equations (A2), (A8), and (A10), equation (A1) can be written as equation (A11):

\[
e(x) = \epsilon_{i-1} - \frac{[(\Delta l)_i + (\Delta l)_{i+1}]^2\epsilon_i + (\Delta l)_{i+1}^2\epsilon_{i+1}}{2(\Delta l)_i[(\Delta l)_i + (\Delta l)_{i+1}]}
\]

\[
+ \frac{3\epsilon_{i-1} - 4\epsilon_i + \epsilon_{i+1}}{2\Delta l} (x - x_{i-1}) + \frac{\epsilon_{i-1} - 2\epsilon_i + \epsilon_{i+1}}{2(\Delta l)^2} (x - x_{i-1})^2
\]  

(A11)

which is equation (11) in the text.

For the constant-domain case, \((\Delta l)_{i-1} = (\Delta l)_i = (\Delta l)_{i+1} = \Delta l\), equation (A11) can be simplified as equation (A12):

\[
e(x) = \epsilon_{i-1} - \frac{3\epsilon_{i-1} - 4\epsilon_i + \epsilon_{i+1}}{2\Delta l} (x - x_{i-1}) + \frac{\epsilon_{i-1} - 2\epsilon_i + \epsilon_{i+1}}{2(\Delta l)^2} (x - x_{i-1})^2
\]  

(A12)

Equation (A12) is identical to equation (4a) in the text.
APPENDIX B
DERIVATION OF VARIABLE-DOMAIN DISPLACEMENT TRANSFER
FUNCTIONS FOR NONUNIFORM EMBEDDED BEAMS

(Based on Piecewise-Linear Strain Representations)

The slope equation (13a) and the deflection equation (13b) for nonuniform beams, based on piecewise-linear representations of both depth factors and surface strains, are duplicated below as equations (B1) and (B2).

Slope equation:

\[
\tan \theta_i = (\Delta l) \left[ \frac{\epsilon_{i-1} - \epsilon_i}{c_{i-1} - c_i} + \frac{\epsilon_i c_{i-1} - \epsilon_{i-1} c_i}{(c_{i-1} - c_i)^2} \log e \frac{c_i}{c_{i-1}} \right] + \tan \theta_{i-1}
\]  

(B1)  

(i = 1,2,3,.....,n)

Deflection equation:

\[
y_i = (\Delta l)^2 \left\{ \frac{\epsilon_{i-1} - \epsilon_i}{2(c_{i-1} - c_i)} - \frac{\epsilon_i c_{i-1} - \epsilon_{i-1} c_i}{(c_{i-1} - c_i)^3} \left[ c_i \log e \frac{c_i}{c_{i-1}} + (c_{i-1} - c_i) \right] \right\} + y_{i-1} + (\Delta l) \tan \theta_{i-1}
\]  

(B2)  

(i = 1,2,3,.....,n)

Writing out equation (B2) for different indices, i (=1,2,3,.....,n) , and considering the indicial relationships expressed in equations (B1) and (B2), one obtains equation (B3):

For \( i = 1 \):

\[
y_1 = (\Delta l)^2 \left\{ \frac{\epsilon_0 - \epsilon_1}{2(c_0 - c_1)} - \frac{\epsilon_1 c_0 - \epsilon_0 c_1}{(c_0 - c_1)^3} \left[ c_1 \log e \frac{c_1}{c_0} + (c_0 - c_1) \right] \right\} + y_0 + (\Delta l) \tan \theta_0
\]  

(B3)
For $i = 2$:

$$y_2 = (\Delta l)_2^2 \left[ \frac{\varepsilon_1 - \varepsilon_2}{2(c_1 - c_2)} - \frac{\varepsilon_1 c_2 - \varepsilon_2 c_1}{(c_1 - c_2)^3} \left[ c_2 \log_e \frac{c_2}{c_1} + (c_1 - c_2) \right] \right] + y_1 + (\Delta l)_2 \tan \theta_1$$

$$= (\Delta l)_2^2 \left[ \frac{\varepsilon_1 - \varepsilon_2}{2(c_1 - c_2)} - \frac{\varepsilon_1 c_2 - \varepsilon_2 c_1}{(c_1 - c_2)^3} \left[ c_2 \log_e \frac{c_2}{c_1} + (c_1 - c_2) \right] \right]$$

$$+ (\Delta l)_1^2 \left[ \frac{\varepsilon_0 - \varepsilon_1}{2(c_0 - c_1)} - \frac{\varepsilon_0 c_1 - \varepsilon_1 c_0}{(c_0 - c_1)^3} \left[ c_1 \log_e \frac{c_1}{c_0} + (c_0 - c_1) \right] \right] + y_0 + (\Delta l)_1 \tan \theta_0$$

$$+ (\Delta l)_2 \left[ \frac{(\Delta l)_1}{\tan \theta_1} \left[ \frac{\varepsilon_0 \varepsilon_1 - \varepsilon_0 c_1 - \varepsilon_1 c_0}{c_0 - c_1} \log_e \frac{c_1}{c_0} \right] + \tan \theta_0 \right]$$

(B4)

After grouping terms, equation (B4) becomes:

$$y_2 = (\Delta l)_2^2 \left[ \frac{\varepsilon_1 - \varepsilon_2}{2(c_1 - c_2)} - \frac{\varepsilon_1 c_2 - \varepsilon_2 c_1}{(c_1 - c_2)^3} \left[ c_2 \log_e \frac{c_2}{c_1} + (c_1 - c_2) \right] \right]$$

$$+ (\Delta l)_1^2 \left[ \frac{\varepsilon_0 - \varepsilon_1}{2(c_0 - c_1)} - \frac{\varepsilon_0 c_1 - \varepsilon_1 c_0}{(c_0 - c_1)^3} \left[ c_1 \log_e \frac{c_1}{c_0} + (c_0 - c_1) \right] \right]$$

$$+ (\Delta l)_2 \left[ \frac{(\Delta l)_1}{\tan \theta_1} \left[ \frac{\varepsilon_0 \varepsilon_1 - \varepsilon_0 c_1 - \varepsilon_1 c_0}{c_0 - c_1} \log_e \frac{c_1}{c_0} \right] + y_0 + \left\lfloor (\Delta l)_1 + (\Delta l)_2 \right\rfloor \tan \theta_0 \right]$$

(B5)
For \( i = 3 \):

\[
\begin{align*}
y_3 &= (\Delta l)^3 \left\{ \frac{e_2 - e_3}{2(c_2 - c_3)} - \frac{e_2e_3 - e_3c_2}{(c_2 - c_3)^3} \left[ c_3 \log_e \frac{e_3}{c_2} + (c_2 - c_3) \right] \right\} + y_2 + (\Delta l)_3 \tan \theta_2 \\
&= (\Delta l)^3 \left\{ \frac{e_2 - e_3}{2(c_2 - c_1)} - \frac{e_2e_3 - e_3c_2}{(c_2 - c_1)^3} \left[ c_3 \log_e \frac{c_3}{c_2} + (c_2 - c_1) \right] \right\}
\end{align*}
\]

\[
\begin{align*}
&+ (\Delta l)^3 \left\{ \frac{e_0 - e_1}{2(c_0 - c_2)} - \frac{e_0e_1 - e_1c_0}{(c_0 - c_2)^3} \left[ c_0 \log_e \frac{c_0}{c_1} + (c_0 - c_2) \right] \right\} \\
&+ (\Delta l)^3 (\Delta l)_1 \left[ \frac{e_0 - e_1}{c_0 - c_1} + \frac{e_0e_1 - e_1c_0}{(c_0 - c_1)^2} \log_e \frac{c_1}{c_0} \right] + y_0 + [(\Delta l)_1 + (\Delta l)_2] \tan \theta_0 \\
&+ (\Delta l)^3 \left\{ \frac{e_1 - e_2}{c_1 - c_2} + \frac{e_1e_2 - e_2c_1}{(c_1 - c_2)^2} \log_e \frac{c_2}{c_1} \right\} + (\Delta l)_1 \left[ \frac{e_0 - e_1}{c_0 - c_1} + \frac{e_0e_1 - e_1c_0}{(c_0 - c_1)^2} \log_e \frac{c_1}{c_0} \right] + \tan \theta_0 
\end{align*}
\]

After grouping terms, equation (B6) becomes equation (B7):

\[
\begin{align*}
y_3 &= (\Delta l)^3 \left\{ \frac{e_2 - e_3}{2(c_2 - c_3)} - \frac{e_2e_3 - e_3c_2}{(c_2 - c_3)^3} \left[ c_3 \log_e \frac{e_3}{c_2} + (c_2 - c_3) \right] \right\} \\
&+ (\Delta l)^3 \left\{ \frac{e_1 - e_2}{2(c_1 - c_2)} - \frac{e_1e_2 - e_2c_1}{(c_1 - c_2)^3} \left[ c_2 \log_e \frac{e_2}{c_1} + (c_1 - c_2) \right] \right\} \\
&+ (\Delta l)^3 \left\{ \frac{e_0 - e_1}{2(c_0 - c_1)} - \frac{e_0e_1 - e_1c_0}{(c_0 - c_1)^3} \left[ c_0 \log_e \frac{e_1}{c_0} + (c_0 - c_1) \right] \right\} \\
&+ (\Delta l)^3 \left\{ \frac{e_1 - e_2}{c_1 - c_2} + \frac{e_1e_2 - e_2c_1}{(c_1 - c_2)^2} \log_e \frac{c_2}{c_1} \right\} \\
&+ [(\Delta l)_1 + (\Delta l)_2] \left\{ \frac{e_0 - e_1}{c_0 - c_1} + \frac{e_0e_1 - e_1c_0}{(c_0 - c_1)^2} \log_e \frac{c_1}{c_0} \right\} \\
&+ y_0 + [(\Delta l)_1 + (\Delta l)_2 + (\Delta l)_3] \tan \theta_0
\end{align*}
\]
For $i = 4$:

$$
y_4 = (\Delta l)_4 \left\{ \frac{\epsilon_3 - \epsilon_4}{2(c_3 - c_4)} \frac{\epsilon_3 \epsilon_4 - \epsilon_4 \epsilon_3}{(c_3 - c_4)^3} \left[ c_4 \log_e \frac{c_4}{c_3} + (c_3 - c_4) \right] \right\} + y_3 + (\Delta l)_4 \tan \theta_3
$$

$$
= (\Delta l)_4 \left\{ \frac{\epsilon_3 - \epsilon_4}{2(c_3 - c_4)} \frac{\epsilon_3 \epsilon_4 - \epsilon_4 \epsilon_3}{(c_3 - c_4)^3} \left[ c_4 \log_e \frac{c_4}{c_3} + (c_3 - c_4) \right] \right\}
$$

$$
+ (\Delta l)_3 \left\{ \frac{\epsilon_2 - \epsilon_3}{2(c_2 - c_3)} \frac{\epsilon_2 \epsilon_3 - \epsilon_3 \epsilon_2}{(c_2 - c_3)^3} \left[ c_3 \log_e \frac{c_3}{c_2} + (c_2 - c_3) \right] \right\}
$$

$$
+ (\Delta l)_2 \left\{ \frac{\epsilon_1 - \epsilon_2}{2(c_1 - c_2)} \frac{\epsilon_1 \epsilon_2 - \epsilon_2 \epsilon_1}{(c_1 - c_2)^3} \left[ c_2 \log_e \frac{c_2}{c_1} + (c_1 - c_2) \right] \right\}
$$

$$
+ (\Delta l)_1 \left\{ \frac{\epsilon_0 - \epsilon_1}{2(c_0 - c_1)} \frac{\epsilon_0 \epsilon_1 - \epsilon_1 \epsilon_0}{(c_0 - c_1)^3} \left[ c_1 \log_e \frac{c_1}{c_0} + (c_0 - c_1) \right] \right\}
$$

$$
+ y_0 + [(\Delta l)_3 + (\Delta l)_2 + (\Delta l)_1] \tan \theta_0
$$

$$
+ (\Delta l)_4 \left\{ \frac{\epsilon_2 - \epsilon_3}{c_2 - c_3} \frac{\epsilon_2 \epsilon_3 - \epsilon_3 \epsilon_2}{(c_2 - c_3)^2} \log_e \frac{c_3}{c_2} + (\Delta l)_2 \frac{\epsilon_1 - \epsilon_2}{c_1 - c_2} \frac{\epsilon_1 \epsilon_2 - \epsilon_2 \epsilon_1}{(c_1 - c_2)^2} \log_e \frac{c_2}{c_1} \right\}
$$

$$
+ (\Delta l)_1 \left\{ \frac{\epsilon_0 - \epsilon_1}{c_0 - c_1} \frac{\epsilon_0 \epsilon_1 - \epsilon_1 \epsilon_0}{(c_0 - c_1)^2} \log_e \frac{c_1}{c_0} \right\} \tan \theta_0
$$

(B8)
After grouping terms, equation (B8) becomes equation (B9):

\[
y_4 = (\Delta l)^2 \left\{ \frac{\varepsilon_3 - \varepsilon_4}{2(c_3 - c_4)} - \frac{\varepsilon_3 c_4 - \varepsilon_4 c_3}{(c_3 - c_4)^3} \left[ c_4 \log_e \frac{c_4}{c_3} + (c_3 - c_4) \right] \right\} \\
+ (\Delta l)^2 \left\{ \frac{\varepsilon_2 - \varepsilon_3}{2(c_2 - c_3)} - \frac{\varepsilon_2 c_3 - \varepsilon_3 c_2}{(c_2 - c_3)^3} \left[ c_3 \log_e \frac{c_3}{c_2} + (c_2 - c_3) \right] \right\} \\
+ (\Delta l)^2 \left\{ \frac{\varepsilon_1 - \varepsilon_2}{2(c_1 - c_2)} - \frac{\varepsilon_1 c_2 - \varepsilon_2 c_1}{(c_1 - c_2)^3} \left[ c_2 \log_e \frac{c_2}{c_1} + (c_1 - c_2) \right] \right\} \\
+ (\Delta l)^2 \left\{ \frac{\varepsilon_0 - \varepsilon_1}{2(c_0 - c_1)} - \frac{\varepsilon_0 c_1 - \varepsilon_1 c_0}{(c_0 - c_1)^3} \left[ c_1 \log_e \frac{c_1}{c_0} + (c_0 - c_1) \right] \right\} \\
+ \left[ (\Delta l)^2 + (\Delta l)_3 \right] \left\{ (\Delta l)_3 \left[ \frac{\varepsilon_3 - \varepsilon_4}{c_3 - c_4} + \frac{\varepsilon_3 c_4 - \varepsilon_4 c_3}{(c_3 - c_4)^2} \log_e \frac{c_3}{c_2} \right] \right\} \\
+ \left[ (\Delta l)^2 + (\Delta l)_3 + (\Delta l)_2 \right] \left\{ (\Delta l)_2 \left[ \frac{\varepsilon_2 - \varepsilon_3}{c_2 - c_3} + \frac{\varepsilon_2 c_3 - \varepsilon_3 c_2}{(c_2 - c_3)^2} \log_e \frac{c_2}{c_1} \right] \right\} \\
+ \left[ (\Delta l)^2 + (\Delta l)_3 + (\Delta l)_2 + (\Delta l)_4 \right] \tan \theta_0 + y_0 + \left[ (\Delta l)_1 + (\Delta l)_2 + (\Delta l)_3 + (\Delta l)_4 \right] \tan \theta_0
\]

(B9)

Based on the indicial progression patterns in equations (B3), (B5), (B7), and (B9), the generalized form of the deflection equation can be written in dual summation forms (with different summation limits) as:

\[
y_i = \sum_{j=1}^{i} \left\{ (\Delta l)^2 \left[ \frac{\varepsilon_{j-1} - \varepsilon_j}{2(c_{j-1} - c_j)} - \frac{\varepsilon_{j-1} c_j - \varepsilon_j c_{j-1}}{(c_{j-1} - c_j)^3} \left[ c_j \log_e \frac{c_j}{c_{j-1}} + (c_{j-1} - c_j) \right] \right] \right\} + y_0 \\
+ \sum_{j=1}^{i} \left\{ \left[ (\Delta l)_{j-1} + (\Delta l)_{j+1} + \ldots + (\Delta l)_{j} \right] (\Delta l)_j \left[ \frac{\varepsilon_{j-1} - \varepsilon_j}{c_{j-1} - c_j} + \frac{\varepsilon_{j-1} c_j - \varepsilon_j c_{j-1}}{(c_{j-1} - c_j)^2} \log_e \frac{c_j}{c_{j-1}} \right] \right\} + \sum_{j=1}^{i} \left[ (\Delta l)_j \right] \tan \theta_0
\]

(B10)
Equation (B10) can be rewritten in a more compact form as:

\[
y_i = \sum_{j=1}^{i} \left[ \left( \Delta l \right)_j \left[ \frac{\xi_{j-1} - \xi_j}{2(c_{j-1} - c_j)} - \frac{\xi_{j-1} c_j - \xi_j c_{j-1}}{(c_{j-1} - c_j)^3} \left( c_j \log \frac{c_j}{c_{j-1}} + (c_{j-1} - c_j) \right) \right] \right] + y_0
\]

Contributions from deflection terms

\[
+ \sum_{j=1}^{i} \left[ \sum_{k=j+1}^{i} \left( \Delta l \right)_k \left[ \frac{\xi_{j-1} - \xi_j}{c_{j-1} - c_j} + \frac{\xi_{j-1} c_j - \xi_j c_{j-1}}{(c_{j-1} - c_j)^2} \log \frac{c_j}{c_{j-1}} \right] \right] + \left[ \sum_{j=1}^{i} \left( \Delta l \right)_j \right] \tan \theta_0
\]

Contributions from slope terms

\[(i = 1, 2, 3, \ldots, n)\]

Equation (B11) is identical to equation (13c) in the text, and is called the Variable-Domain Nonuniform Displacement Transfer Function in dual summation form for nonuniform embedded beams based on piecewise-linear strain representations.

For constant-domain lengths, \((\Delta l)_1 = (\Delta l)_2 = (\Delta l)_3 = \ldots = \Delta l\), equation (B11) degenerates into:

\[
y_i = (\Delta l)^2 \sum_{j=1}^{i} \left[ \left( \Delta l \right)_j \left[ \frac{\xi_{j-1} - \xi_j}{2(c_{j-1} - c_j)} - \frac{\xi_{j-1} c_j - \xi_j c_{j-1}}{(c_{j-1} - c_j)^3} \left( c_j \log \frac{c_j}{c_{j-1}} + (c_{j-1} - c_j) \right) \right] \right] + y_0
\]

Contributions from deflection terms

\[
+ (\Delta l)^2 \sum_{j=1}^{i} \left[ (i - j) \left[ \frac{\xi_{j-1} - \xi_j}{c_{j-1} - c_j} + \frac{\xi_{j-1} c_j - \xi_j c_{j-1}}{(c_{j-1} - c_j)^2} \log \frac{c_j}{c_{j-1}} \right] \right] + (i) \Delta l \tan \theta_0
\]

Contributions from slope terms

\[(i = 1, 2, 3, \ldots, n)\]

Equation (B12) is identical to equation (5c) in the text, and is called the Constant-Domain Nonuniform Displacement Transfer Function in dual summation form for nonuniform embedded beams based on piecewise-linear strain representations.
APPENDIX C  
DERIVATIONS OF VARIABLE-DOMAIN IMPROVED DISPLACEMENT TRANSFER FUNCTION FOR NONUNIFORM EMBEDDED BEAMS  
(Based on Piecewise-Nonlinear Strain Representations)

Appendix C presents the details of integrations of the beam curvature equation (1) to obtain the improved slope and deflection equations for the formulation of variable-domain Improved Displacement Transfer Functions for nonuniform embedded beams based on piecewise-nonlinear strain representations.

**Improved Slope Equations**

The slope, \( \tan \theta(x) \), of the nonuniform beam in the small domain, \( x_{i-1} \leq x \leq x_i \), between the two adjacent strain-sensing stations, \( \{x_{i-1}, x_i\} \), can be obtained by integrating equation (1) as equation (C1):

\[
\int_{x_{i-1}}^{x} \frac{dy}{dx} \\, dx = \frac{\delta y}{dx} \bigg|_{x_{i-1}}^{x} \equiv \tan \theta(x) - \tan \theta_{i-1} = \int_{x_{i-1}}^{x} \epsilon(x) \, dx ; \quad (x_{i-1} \leq x \leq x_i) \quad (C1)
\]

The distributions of the beam depth factor, \( c(x) \) [eq. (9)], and bending strain, \( \epsilon(x) \) [eq. (11)], in the small domain, \( x_{i-1} \leq x \leq x_i \), between the two adjacent strain-sensing stations, \( \{x_{i-1}, x_i\} \), are respectively duplicated below as equations (C-2) and (C-3).

\[
c(x) = c_{i-1} + (c_i - c_{i-1}) \frac{x - x_{i-1}}{(\Delta l)_i} ; \quad (x_{i-1} \leq x \leq x_i) \quad (C2)
\]

\[
\epsilon(x) = \epsilon_{i-1} - \frac{(\Delta l)_{i+1}[2(\Delta l)_i + (\Delta l)_{i+1}]\epsilon_{i-1} - [(\Delta l)_i + (\Delta l)_{i+1}]^2 \epsilon_i + (\Delta l)_i^2 \epsilon_{i+1}(x - x_{i-1})}{(\Delta l)_i(\Delta l)_{i+1}[(\Delta l)_i + (\Delta l)_{i+1}]} \nonumber \\
+ \frac{(\Delta l)_{i+1}\epsilon_{i-1} - [(\Delta l)_i + (\Delta l)_{i+1}]\epsilon_i + (\Delta l)_i \epsilon_{i+1}(x - x_{i-1})^2}{(\Delta l)_i(\Delta l)_{i+1}[(\Delta l)_i + (\Delta l)_{i+1}]} ; \quad (x_{i-1} \leq x \leq x_i) \quad (C3)
\]

Let \( \{A_i, B_i, C_i, \xi\} \) be defined respectively with equations (C4), (C5), (C6), (C7) as

\[
A_i \equiv -\frac{(\Delta l)_{i+1}[2(\Delta l)_i + (\Delta l)_{i+1}]\epsilon_{i-1} - [(\Delta l)_i + (\Delta l)_{i+1}]^2 \epsilon_i + (\Delta l)_i^2 \epsilon_{i+1}}{(\Delta l)_i(\Delta l)_{i+1}[(\Delta l)_i + (\Delta l)_{i+1}]} \quad (C4)
\]

\[
B_i \equiv \frac{(\Delta l)_{i+1}\epsilon_{i-1} - [(\Delta l)_i + (\Delta l)_{i+1}]\epsilon_i + (\Delta l)_i \epsilon_{i+1}}{(\Delta l)_i(\Delta l)_{i+1}[(\Delta l)_i + (\Delta l)_{i+1}]} \quad (C5)
\]

\[
C_i \equiv \frac{c_i - c_{i-1}}{(\Delta l)_i} \quad (C6)
\]

\[
\xi \equiv (x - x_{i-1}) \quad (C7)
\]
then the last three terms of equation (C1) can be written as:

$$\tan \theta(x) = \int_0^x \frac{e_{i+1} + A_i \xi + B_i \xi^2}{c_i + C_i \xi^3} d\xi + \tan \theta_{i-1} \quad [0 \leq \xi \leq (\Delta_i)] \quad \text{(C8)}$$

After carrying out integration of equation (C8), one obtains (ref. 15):

$$\begin{align*}
\tan \theta(x) &= \left[ \left( \frac{e_{i-1}}{C_i} \right. - A_i \frac{c_{i-1}}{C_i} + B_i \frac{c_{i-1}^2}{C_i} \right) \log_e(c_{i-1} + C_i \xi) + A_i \frac{\xi}{C_i} - A_i \frac{c_{i-1}}{C_i} \log_e(c_{i-1} + C_i \xi) \\
&\quad + \frac{B_i}{C_i^3} \left[ \frac{1}{2} (c_{i-1} + C_i \xi)^2 - 2 c_{i-1} (c_{i-1} + C_i \xi) \right. \\
&\quad + \left. c_{i-1}^2 \log_e(c_{i-1} + C_i \xi) \right] \right] + \tan \theta_{i-1} \\
&= \frac{1}{C_i^3} \left[ C_i c_{i-1} - A_i C_i c_{i-1} + B_i c_{i-1}^2 \right] \left[ \log_e(c_{i-1} + C_i \xi) - \log_e(c_{i-1}) + \frac{A_i}{C_i} \xi \\
&\quad + \frac{B_i}{2 C_i^3} \left[ (c_{i-1} + C_i \xi)^2 - c_{i-1}^2 \right] - 2 B_i C_i \left[ (c_{i-1} + C_i \xi) - c_{i-1} \right] + \tan \theta_{i-1} \right]
\end{align*}$$

\text{(C9)}
After simplifying, equation (C9) becomes:

\[
\tan \theta(x) = \frac{1}{C_i^3} \left[ C_i^2 e_{i-1} - A_i c_i c_{i-1} + B_i c_{i-1}^2 \right] \log_e \left( 1 + \frac{C_i}{c_{i-1}} \xi \right) + \frac{\xi}{C_i} A_i + B_i \left( \frac{\xi}{2} - \frac{c_{i-1}}{C_i} \right) + \tan \theta_{i-1}
\]  

(C10)

At the strain-sensing station, \( x_i \), we have \( \xi = x_i - x_{i-1} = (\Delta l)_i \), then equation (C10) gives the slope \( \tan \theta_i [\equiv \tan \theta(x_i)] \) at \( x_i \) as:

\[
\tan \theta_i = \frac{1}{C_i^3} \left[ C_i^2 e_{i-1} - A_i c_i c_{i-1} + B_i c_{i-1}^2 \right] \log_e \left( 1 + \frac{C_i}{c_{i-1}} (\Delta l)_i \right) + \frac{(\Delta l)_i}{C_i} \left[ A_i + B_i \left( \frac{(\Delta l)_i}{2} - \frac{c_{i-1}}{C_i} \right) \right] + \tan \theta_{i-1}
\]  

(C11)

In view of \( C_i \equiv (c_i - c_{i-1})/(\Delta l)_i \) [eq. (C6)], equation (C11) takes on the form:

\[
\tan \theta_i = \frac{(\Delta l)^3_i}{(c_i - c_{i-1})^3} \left[ (c_i - c_{i-1})^2 e_{i-1} - A_i \frac{c_i - c_{i-1}}{c_{i-1}} c_{i-1} + B_i c_{i-1}^2 \right] \log_e \left( 1 + \frac{c_i - c_{i-1}}{c_{i-1}} \frac{(\Delta l)_i}{(\Delta l)_i} \right) + \frac{(\Delta l)^2_i}{c_i - c_{i-1}} \left[ A_i + B_i \left( \frac{(\Delta l)_i}{2} - \frac{c_{i-1}}{c_i - c_{i-1}} \right) \right] + \tan \theta_{i-1}
\]

\[
= \frac{(\Delta l)^3_i}{(c_i - c_{i-1})^3} \left[ \frac{(c_i - c_{i-1})^2 e_{i-1} - A_i c_i c_{i-1} - B_i c_{i-1}^2}{(\Delta l)_i} \right] \log_e \frac{c_i}{c_{i-1}} + \frac{(\Delta l)^2_i}{c_i - c_{i-1}} \left[ A_i + B_i \left( \frac{3c_i c_{i-1}}{2(c_i - c_{i-1})} \right) \right] + \tan \theta_{i-1}
\]  

(C12)

After grouping terms, equation (C12) becomes:

\[
\tan \theta_i = \frac{(\Delta l)^3_i}{(c_i - c_{i-1})^3} \left[ (c_i - c_{i-1})^2 e_{i-1} - A_i (\Delta l)_i c_{i-1} - B_i (\Delta l)^2_i c_{i-1}^2 \right] \log_e \frac{c_i}{c_{i-1}} + \frac{(\Delta l)^2_i}{2(c_i - c_{i-1})} \left[ 2A_i (\Delta l)_i (c_i - c_{i-1}) + B_i (\Delta l)^2_i (c_i - 3c_{i-1}) \right] + \tan \theta_{i-1}
\]  

(C13)

\( i = 1, 2, 3, \ldots, n \)

Equation (C13) is identical to equation (17a) in the text, the \textit{variable-domain improved slope equation} for nonuniform beams in recursive form based on nonlinear strain representations.
Improved Deflection Equations

The deflection, \( y(x) \), can be obtained by integrating the slope equation (C10) as (ref. 15).

\[
y(x) = \int_{x_{i-1}}^{x_i} \tan(x)\,dx + \frac{y_{i-1}}{\text{Deflection at } x_{i-1}} = \int_0^\xi \tan(\xi)\,d\xi + y_{i-1}
\]

\[
= \left[ \frac{1}{C_i} \left[ C_i^2 e_{i-1}^2 - A_i C_i c_{i-1} + B_i c_{i-1}^2 \right] \log_e \left( 1 + \frac{c_i^2}{c_{i-1}^2} \xi \right) \right]_{\xi=0}^{\xi=\xi_i} + \frac{\xi_i^2}{2C_i} \left[ A_i + B_i \left( \frac{\xi_i^3}{3} - \frac{c_{i-1}^2 \xi_i}{C_i} \right) \right] + y_{i-1} + \xi_i \tan \theta_{i-1}
\]

\[
= \frac{1}{C_i} \left[ C_i^2 e_{i-1}^2 - A_i C_i c_{i-1} + B_i c_{i-1}^2 \right] \log_e \left( 1 + \frac{c_i^2}{c_{i-1}^2} \xi_i \right) \left( 1 + \frac{C_i}{c_{i-1}} \xi_i \right) - \xi_i
\]

\[
+ \frac{\xi_i^2}{2C_i} \left[ A_i + B_i \left( \frac{\xi_i^3}{3} - \frac{c_{i-1}^2 \xi_i}{C_i} \right) \right] + y_{i-1} + \xi_i \tan \theta_{i-1}
\]

\[\text{(C14)}\]

At the strain-sensing station, \( x_i \), we have \( \xi = x_i - x_{i-1} = (\Delta l)_i \), and equation (C14) gives the deflection \( y_i = y(x_i) \) as:

\[
y_i = y(x_i) = \frac{1}{C_i} \left[ C_i^2 e_{i-1}^2 - A_i C_i c_{i-1} + B_i c_{i-1}^2 \right] \log_e \left( 1 + \frac{c_i^2}{c_{i-1}^2} (\Delta l)_i \right) \left( 1 + \frac{C_i}{c_{i-1}} (\Delta l)_i \right) - (\Delta l)_i \]

\[+ \frac{(\Delta l)_i^2}{2C_i} \left[ A_i + B_i \left( \frac{(\Delta l)_i}{3} - \frac{c_{i-1}^2 (\Delta l)_i}{C_i} \right) \right] + y_{i-1} + (\Delta l)_i \tan \theta_{i-1}\]

\[\text{(C15)}\]

In view of \( C_i \equiv (c_i - c_{i-1})/(\Delta l)_i \) [eq. (C6)], equation (C15) becomes:

\[
y_i = \frac{(\Delta l)_i^4}{(c_i - c_{i-1})^4} \left[ \frac{(c_i - c_{i-1})^2}{(\Delta l)_i^2} e_{i-1} - A_i \frac{c_i - c_{i-1}}{(\Delta l)_i} c_{i-1} + B_i c_{i-1}^2 \right] \left( \frac{1}{c_i - c_{i-1}} \right) \left[ c_i \log_e \frac{c_i}{c_{i-1}} - (c_i - c_{i-1}) \right]
\]

\[+ \frac{(\Delta l)_i^3}{2(c_i - c_{i-1})} \left[ A_i + B_i \left( \frac{(\Delta l)_i}{3} - \frac{c_{i-1}^2 (\Delta l)_i}{(c_i - c_{i-1})} \right) \right] + y_{i-1} + (\Delta l)_i \tan \theta_{i-1}
\]

\[\text{(C16)}\]
After grouping terms, equation (C16) becomes:

\[
y_i = \frac{(\Delta l)^2}{(c_i - c_{i+1})^1} \left[ (c_i - c_{i+1})^2 e_{i+1} - A_j(\Delta l_i)(c_i - c_{i+1})e_{i+1} + B_j(\Delta l_i) c_{i+1}^2 \right] c_i \log \frac{c_i}{c_{i+1}} - (c_i - c_{i+1}) \\
+ \frac{(\Delta l)^2}{6(c_i - c_{i+1})^2} \left[ 3A_j(\Delta l_i)(c_i - c_{i+1}) + B_j(\Delta l_i)(c_i - 4c_{i+1}) \right] + y_{i-1} + (\Delta l_i) \tan \theta_{i-1} \\
(i = 1, 2, 3, ..., n) \]  

(C17)

Equation (C17) is identical to equation (17b) in the text, the variable-domain improved deflection equation for nonuniform beams in recursive form based on nonlinear strain representations.

**Improved Displacement Transfer Function**

Substitution of slope equation (C13) into deflection equation (C17), and using similar mathematical steps of Appendix B, one can write the deflection equation in dual summation form as:

\[
y_i = \sum_{j=1}^{i} \left[ \frac{(\Delta l_i)^2}{(c_j - c_{j+1})^1} \left[ (c_j - c_{j+1})^2 e_{j+1} - A_j(\Delta l_i)(c_j - c_{j+1})e_{j+1} + B_j(\Delta l_i) c_{j+1}^2 \right] x_c \log \frac{c_j}{c_{j+1}} - (c_j - c_{j+1}) \\
+ \frac{(\Delta l_i)^2}{6(c_j - c_{j+1})^2} \left[ 3A_j(\Delta l_i)(c_j - c_{j+1}) + B_j(\Delta l_i)(c_j - 4c_{j+1}) \right] \right] + y_0 \\
+ \sum_{j=1}^{i} \left[ \sum_{k=j+1}^{i} (\Delta l_k) \right] \left[ \frac{(\Delta l_k)^2}{(c_j - c_{j+1})^3} \left[ (c_j - c_{j+1})^2 e_{j+1} - A_j(\Delta l_i)(c_j - c_{j+1})e_{j+1} + B_j(\Delta l_i) c_{j+1}^2 \right] \log \frac{c_j}{c_{j+1}} - (c_j - c_{j+1}) \\
+ \frac{(\Delta l_k)^2}{2(c_j - c_{j+1})^2} \left[ 2A_j(\Delta l_i)(c_j - c_{j+1}) + B_j(\Delta l_i)(c_j - 3c_{j+1}) \right] \right] \]

\[
+ \sum_{j=1}^{i} (\Delta l_j) \tan \theta_0 \]

\[(i = 1, 2, 3, ..., n) \]  

(C18)

Equation (C18) is identical to equation (17c) in the text, and is called the Variable-Domain Improved Displacement Transfer Function in dual summation form for nonuniform embedded beams based on piecewise-nonlinear strain representations.
Degeneration into Constant-Domain Case

This section is to prove that when the variable-domain lengths become constant, the variable-domain slope and deflection equations [(C13), (C17)] will degenerate respectively into the constant-domain slope and deflection equations [(5a), (5b)].

For the constant-domain case, \((\Delta l)_1 = (\Delta l)_2 = (\Delta l)_3 = \ldots = \Delta l\), the coefficients, \(A_i\) [eq. (C4)] and \(B_i\) [eq. (C5)] become equations (C19) and (C20) respectively:

\[
A_i = -\frac{(\Delta l)_i + (\Delta l)_{i+1} - (\Delta l)_{i+1}}{2(\Delta l)} \left[\frac{1}{(\Delta l)_{i+1}} + (\Delta l)_{i+1}\right] = -\frac{1}{2\Delta l} - \epsilon_{i-1} + \epsilon_{i+1} \tag{C19} \\
B_i = \frac{(\Delta l)_i + (\Delta l)_{i+1} - (\Delta l)_{i+1}}{2(\Delta l)} \left[\frac{1}{(\Delta l)_{i+1}} + (\Delta l)_{i+1}\right] = \frac{2\epsilon_i + \epsilon_{i+1}}{2(\Delta l)^2} \tag{C20}
\]

Slope Equations

In view of equations [(C19), (C20)], the slope equation (C13) can be written as:

\[
\tan \theta_i = \frac{(\Delta l)_i}{c_i - c_{i-1}} \left[ (c_i - c_{i-1})^2 \epsilon_{i-1} - A_i (\Delta l) c_{i-1} (c_i - c_{i-1}) + B_i (\Delta l)^2 c_{i-1}^2 \right] \log_e \frac{c_i}{c_{i-1}} \\
+ \frac{(\Delta l)_i}{2(c_i - c_{i-1})^2} \left[ 2A_i (\Delta l) (c_i - c_{i-1}) + B_i (\Delta l)^2 (c_i - 3c_{i-1}) \right] + \tan \theta_{i-1}
\]

\[
= \frac{\Delta l}{2(c_i - c_{i-1})^3} \left[ (c_i - c_{i-1})^2 \epsilon_{i-1} - \left( \frac{3 \epsilon_{i-1} - 4 \epsilon_i + \epsilon_{i+1}}{2A_i (\Delta l)} \right) c_{i-1}^2 (c_i - c_{i-1}) + \left( \frac{\epsilon_{i-1} - 2 \epsilon_i + \epsilon_{i+1}}{2B_i (\Delta l)^2} \right) c_{i-1}^2 \right] \log_e \frac{c_i}{c_{i-1}} \\
+ \frac{\Delta l}{2(c_i - c_{i-1})^2} \left[ 2(c_i - c_{i-1})^2 \epsilon_{i-1} + (3 \epsilon_{i-1} - 4 \epsilon_i + \epsilon_{i+1}) (c_i - c_{i-1}) c_{i-1} \right] + \tan \theta_{i-1}
\]

\[
\tan \theta_i = -\frac{2(3 \epsilon_{i-1} - 4 \epsilon_i + \epsilon_{i+1}) (c_i - c_{i-1}) + (\epsilon_{i-1} - 2 \epsilon_i + \epsilon_{i+1}) (c_i - 3c_{i-1})}{2(c_i - c_{i-1})^2} \log_e \frac{c_i}{c_{i-1}} \\
+ \frac{\Delta l}{4(c_i - c_{i-1})^3} \left[ (6 \epsilon_{i-1} - 8 \epsilon_i + 2 \epsilon_{i+1}) (c_i - c_{i-1}) + (\epsilon_{i-1} - 2 \epsilon_i + \epsilon_{i+1}) (c_i - 3c_{i-1}) \right] + \tan \theta_{i-1}
\]
In view of equations [(C19), (C20)], the deflection equation (C17) can be written as:

\[
\frac{\Delta l}{2(c_i - c_{i-1})^3} \left[ (2c_i^2 - 4c_i c_{i-1} + 2c_{i-1}^2) + (3c_i - 3c_{i-1})c_{i-1} + c_{i-1}^2 \right] \varepsilon_{i-1} \log \frac{c_i}{c_{i-1}} + \frac{\Delta l}{4(c_i - c_{i-1})^2} \left[ -(6c_i - 6c_{i-1}) + (c_i - 3c_{i-1}) \right] \varepsilon_{i-1} + \frac{\Delta l}{4(c_i - c_{i-1})^2} (8c_i - 8c_{i-1} - 2c_i - 6c_{i-1}) \varepsilon_{i} + \tan \theta_{i-1} \\
+ \frac{\Delta l}{4(c_i - c_{i-1})^2} \left[ -2(2c_i - 2c_{i-1}) + (c_i - 3c_{i-1}) \right] \varepsilon_{i-1} + \frac{\Delta l}{4(c_i - c_{i-1})^2} \left[ -2(2c_i - 2c_{i-1}) + (c_i - 3c_{i-1}) \right] \varepsilon_{i} + \tan \theta_{i-1} \\
= \frac{\Delta l}{2(c_i - c_{i-1})^3} \left[ (2c_i^2 - 4c_i c_{i-1} + 2c_{i-1}^2) e_{i-1} \right] + \frac{\Delta l}{4(c_i - c_{i-1})^2} \left[ -5c_i + 3c_{i-1} \right] e_{i-1} + \frac{\Delta l}{4(c_i - c_{i-1})^2} \left[ -5c_i + 3c_{i-1} \right] e_{i} + \tan \theta_{i-1} \\
+ \frac{\Delta l}{4(c_i - c_{i-1})^2} \left[ -5c_i + 3c_{i-1} \right] e_{i-1} + 2(3c_i - c_{i-1}) e_i - (c_i + c_{i-1}) e_{i-1} + \tan \theta_{i-1} \tag{C21}
\]

After grouping terms, equation (C21) becomes:

\[
\tan \theta_i = \frac{\Delta l}{2(c_i - c_{i-1})^3} \left[ (2c_i - c_{i-1})(c_i e_{i-1} - 2c_{i-1} e_i) + c_i e_{i-1} e_{i+1} \right] \log \frac{c_i}{c_{i-1}} \\
- \frac{\Delta l}{4(c_i - c_{i-1})^2} \left[ 5c_i - 3c_{i-1} \right] e_{i-1} - 2(3c_i - c_{i-1}) e_i - (c_i + c_{i-1}) e_{i-1} + \tan \theta_{i-1} \tag{C22}
\]

Equation (C22) is identical to the improved slope equation (7a) for the constant-domain case.

**Deflection Equations**

In view of equations [(C19), (C20)], the deflection equation (C17) can be written as:

\[
y_i = \frac{(\Delta l)^2}{(c_i - c_{i-1})^4} \left[ (c_i - c_{i-1})^2 e_{i-1} - A_i (\Delta l)(c_i - c_{i-1})c_{i-1} + B_i (\Delta l)^2 c_{i-1}^2 \right] \left[ c_i \log \frac{c_i}{c_{i-1}} - (c_i - c_{i-1}) \right] \\
+ \frac{(\Delta l)^2}{6(c_i - c_{i-1})^3} \left[ 3A_i (\Delta l)(c_i - c_{i-1}) + B_i (\Delta l)^2 (c_i - 4c_{i-1}) \right] + y_{i-1} + (\Delta l) \tan \theta_{i-1}
\]
\[
\begin{align*}
&\frac{(\Delta l)^2}{(c_i - c_{i-1})^4} \left[ (c_i - c_{i-1})^2 \varepsilon_{i-1} - \left( -\frac{3\varepsilon_{i-1} - 4\varepsilon_i + \varepsilon_{i+1}}{2} \right) c_i - c_{i-1} + \frac{\varepsilon_{i-1} - 2\varepsilon_i + \varepsilon_{i+1}}{2} c_i'^2 \right] \\
&\times \left[ c_i \log_e \left( \frac{c_i}{c_{i-1}} \right) - (c_i - c_{i-1}) \right] \\
&+ \frac{(\Delta l)^2}{2(c_i - c_{i-1})} \left[ \left( -\frac{3\varepsilon_{i-1} - 4\varepsilon_i + \varepsilon_{i+1}}{2} \right) c_i - c_{i-1} + \frac{\varepsilon_{i-1} - 2\varepsilon_i + \varepsilon_{i+1}}{2} c_i \right] c_i' + \left( \Delta l \right) \tan \theta_{i-1} \\
&= \frac{(\Delta l)^2}{2(c_i - c_{i-1})^4} \left[ 2(c_i - c_{i-1})^2 \varepsilon_{i-1} + (3\varepsilon_{i-1} - 4\varepsilon_i + \varepsilon_{i+1}) c_i (c_i - c_{i-1}) \right] \\
&+ \frac{(\Delta l)^2}{12(c_i - c_{i-1})^2} \left[ -3(3\varepsilon_{i-1} - 4\varepsilon_i + \varepsilon_{i+1}) c_i + (c_i - c_{i-1}) \right] + \left( \Delta l \right) \tan \theta_{i-1} \\
&= \frac{(\Delta l)^2}{2(c_i - c_{i-1})^4} \left[ (2c_i^2 - 4c_i c_{i-1} + c_{i-1}^2) + 3(c_i - c_{i-1}) c_i + c_i'^2 \right] \varepsilon_{i-1} \\
&+ \left( (c_i - c_{i-1}) c_i' + c_i'^2 \right) \varepsilon_{i-1} \\
&+ \frac{(\Delta l)^2}{12(c_i - c_{i-1})^2} \left[ -9(c_i - c_{i-1}) + (c_i - 4c_i) \right] c_i'^2 \\
&+ \left( 12(c_i - c_{i-1}) - 2(c_i - 4c_i) \right) c_i' + \left( \Delta l \right) \tan \theta_{i-1} \\
&= \frac{(\Delta l)^2}{2(c_i - c_{i-1})^4} \left[ (2c_i^2 - c_i c_{i-1}) c_{i-1} - 2(2c_i c_{i-1}^2 - c_i^2) c_i + c_i c_{i-1} c_{i+1} \left[ c_i \log_e \left( \frac{c_i}{c_{i-1}} \right) - (c_i - c_{i-1}) \right] \\
&+ \frac{(\Delta l)^2}{12(c_i - c_{i-1})^2} \left[ -(8c_i - 5c_{i-1}) c_i c_{i-1} + 2(5c_i - 2c_{i-1}) c_i - (2c_i + c_{i-1}) c_{i+1} \right] + (\Delta l) \tan \theta_{i-1}\right] \\
\end{align*}
\]

After grouping terms, equation (C23) becomes:

\[
y_i = \frac{(\Delta l)^2}{2(c_i - c_{i-1})^4} \left[ (2c_i - c_{i-1})(c_i c_{i-1} - 2c_{i-1} c_i) + c_i c_{i-1} c_{i+1} \right] \left[ c_i \log_e \left( \frac{c_i}{c_{i-1}} \right) - (c_i - c_{i-1}) \right] \\
- \frac{(\Delta l)^2}{12(c_i - c_{i-1})^2} \left[ -(8c_i - 5c_{i-1}) c_i c_{i-1} - 2(5c_i - 2c_{i-1}) c_i + (2c_i + c_{i-1}) c_{i+1} \right] + (\Delta l) \tan \theta_{i-1} \\
\]

Equation (C24) is identical to the improved deflection equation (7b) for the constant-domain case.
Displacement Transfer Function

Combining equations (C22), (C24), and after grouping terms, one obtains the deflection equation in dual summation form as:

\[
y_i = (\Delta l)^2 \sum_{j=1}^{n} \left\{ \frac{1}{2(c_j - c_{j-1})} \left[ (2c_j - c_{j-1})(c_j e_{j-1} - 2c_{j-1} e_j) + c_j e_{j-1} e_j \right] \times \right. \\
\times \left[ c_j \log \frac{c_j}{c_{j-1}} - (c_j - c_{j-1}) \right] \\
\left. \frac{1}{12(c_j - c_{j-1})^2} \left[ (8c_j - 5c_{j-1}) e_{j-1} - 2(5c_j - 2c_{j-1}) e_j + (2c_j + c_{j-1}) e_{j+1} \right] \right\} + y_0 \\
+ (\Delta l)^2 \sum_{j=1}^{n} (i - j) \left\{ \frac{1}{2(c_j - c_{j-1})} \left[ (2c_j - c_{j-1})(c_j e_{j-1} - 2c_{j-1} e_j) + c_j e_{j-1} e_j \right] \log \frac{c_j}{c_{j-1}} \\
\frac{1}{4(c_j - c_{j-1})^2} \left[ (5c_j - 3c_{j-1}) e_{j-1} - 2(3c_j - c_{j-1}) e_j + (c_j + c_{j-1}) e_{j+1} \right] \right\} \\
+ (i)\Delta l \tan \theta_0 \right\} \\
(i = 1,2,3,\ldots,n)
\]

Equation (C25) is identical to equation (7c), the Constant-Domain Improved Displacement Transfer Function in dual summation form for nonuniform beams based on piecewise-nonlinear strain representations.
APPENDIX D
DERIVATIONS OF THE VARIABLE-DOMAIN LOG-EXPANDED
DISPLACEMENT TRANSFER FUNCTION FOR SLIGHTLY
NONUNIFORM EMBEDDED BEAMS

(Based on Piecewise-Nonlinear Strain Representations)

Appendix D presents the mathematical derivations of the slope equation (18a), deflection equation (18b), and Displacement Transfer Function (18c) for slightly nonuniform beams through expansions of logarithmic functions.

Log-Expanded Slope Equations

The slope equation (17a) is duplicated below as equation (D1).

\[
\tan \theta = \frac{(\Delta l)_i}{(c_i - c_{i-1})^3} \left[ (c_i - c_{i-1})^2 \varepsilon_{i-1} - A_i(\Delta l)_i c_{i-1} + B_i(\Delta l)_i^2 c_{i-1} \right] \log_e \frac{c_i}{c_{i-1}} \\
+ \frac{(\Delta l)_i}{2(c_i - c_{i-1})^2} \left[ 2A_i(\Delta l)_i (c_i - c_{i-1}) + B_i(\Delta l)_i^2 (c_i - 3c_{i-1}) \right] + \tan \theta_{i-1}
\]

(D1)

For slowly changing \( c(x) \) [that is, \( (c_i / c_{i-1}) \rightarrow 1 \)], the logarithm term, \( \log_e (c_i / c_{i-1}) \), in the slope equation (D1) can be expanded in series form up to the third order term as (ref. 15):

\[
\log_e \frac{c_i}{c_{i-1}} = \left( \frac{c_i}{c_{i-1}} - 1 \right) - \frac{1}{2} \left( \frac{c_i}{c_{i-1}} - 1 \right)^2 + \frac{1}{3} \left( \frac{c_i}{c_{i-1}} - 1 \right)^3 - \ldots
\]

\[
= \frac{c_i - c_{i-1}}{c_{i-1}} \left[ 1 - \frac{c_i - c_{i-1}}{2c_{i-1}} + \frac{(c_i - c_{i-1})^2}{3c_{i-1}^2} - \ldots \right]
\]

\[
= \frac{c_i - c_{i-1}}{6c_{i-1}} \left[ 6c_{i-1}^2 - 3c_{i-1}(c_i - c_{i-1}) + 2(c_i - c_{i-1})^2 - \ldots \right]
\]

\[
= \frac{c_i - c_{i-1}}{6c_{i-1}} \left[ 6c_{i-1}^2 - (c_i - c_{i-1})(3c_{i-1} - 2(c_i - c_{i-1})) - \ldots \right]
\]

\[
= \frac{c_i - c_{i-1}}{6c_{i-1}} \left[ 6c_{i-1}^2 + (c_i - c_{i-1})(2c_i - 5c_{i-1}) - \ldots \right]
\]

(D2)
In view of equation (D2), equation (D1) can be written as:

\[
\tan \theta = \frac{(\Delta l)}{(c_i - c_{i-1})} \left[ (c_i - c_{i-1})^2 \varepsilon_{i-1} - A_i(\Delta l,c_{i-1})(c_i - c_{i-1}) + B_i(\Delta l)^2 c_{i-1}^2 \right] \times \\
\frac{c_i - c_{i-1}}{6c_{i-1}^3} \left[ 6c_{i-1}^2 + (c_i - c_{i-1})(2c_i - 5c_{i-1}) - ... \right] \\
+ \frac{(\Delta l)}{2(c_i - c_{i-1})^3} \left[ 2A_i(\Delta l)(c_i - c_{i-1}) + B_i(\Delta l)^2 (c_i - 3c_{i-1}) \right] + \tan \theta_{i-1}
\]

\[
= \frac{(\Delta l)}{6c_{i-1}^3(c_i - c_{i-1})^2} \left[ (c_i - c_{i-1})^2 \left[ 6c_{i-1}^2 + (c_i - c_{i-1})(2c_i - 5c_{i-1}) - ... \right] \varepsilon_{i-1} \\
- A_i(\Delta l) c_{i-1} (c_i - c_{i-1}) \left[ 6c_{i-1}^2 + (c_i - c_{i-1})(2c_i - 5c_{i-1}) - ... \right] \\
+ B_i(\Delta l)^2 c_{i-1}^2 \left[ 6c_{i-1}^2 + (c_i - c_{i-1})(2c_i - 5c_{i-1}) - ... \right] \\
- A_i(\Delta l) c_{i-1}^3 (c_i - c_{i-1}) + 3B_i(\Delta l)^2 c_{i-1}^3 (c_i - 3c_{i-1}) \right] + \tan \theta_{i-1}
\]

\[
= \frac{(\Delta l)}{6c_{i-1}^3(c_i - c_{i-1})^2} \left[ (c_i - c_{i-1})^2 \left[ 6c_{i-1}^2 + (c_i - c_{i-1})(2c_i - 5c_{i-1}) - ... \right] \varepsilon_{i-1} \\
- 6A_i(\Delta l) c_{i-1}^3 (c_i - c_{i-1}) - A_i(\Delta l) c_{i-1} (c_i - c_{i-1})^2 (2c_i - 5c_{i-1}) - ... \\
- 6A_i(\Delta l) c_{i-1}^3 (c_i - c_{i-1}) + 3B_i(\Delta l)^2 c_{i-1}^3 (c_i - 3c_{i-1}) \right] + \tan \theta_{i-1}
\]

\[
= \frac{(\Delta l)}{6c_{i-1}^3(c_i - c_{i-1})^2} \left[ (c_i - c_{i-1})^2 \left[ 6c_{i-1}^2 + (c_i - c_{i-1})(2c_i - 5c_{i-1}) - ... \right] \varepsilon_{i-1} \\
- A_i(\Delta l) c_{i-1} (c_i - c_{i-1})^2 (2c_i - 5c_{i-1}) - ... \\
+ 2B_i(\Delta l)^2 c_{i-1}^2 (c_i - c_{i-1})^2 \right] + \tan \theta_{i-1} \tag{D3}
\]

After cancelling \((c_i - c_{i-1})^2\) terms, equation (D3) becomes:

\[
\tan \theta = \frac{(\Delta l)}{6c_{i-1}^3} \left[ 6c_{i-1}^2 + (c_i - c_{i-1})(2c_i - 5c_{i-1}) \right] \varepsilon_{i-1} \\
- A_i(\Delta l) c_{i-1} (2c_i - 5c_{i-1}) + 2B_i(\Delta l)^2 c_{i-1}^2 \right] + \tan \theta_{i-1} \tag{D4}
\]

Equation (D4) is the Log-expanded slope equation (18a) in the text.
For uniform beam \((c_{i-1} = c_i = c)\), equation (D4) becomes:

\[
\tan \theta_i = \frac{(\Delta l)}{c} \left[ e_{i-1} + \frac{A}{2} (\Delta l), + \frac{B}{3} (\Delta l)^2 \right] + \tan \theta_{i-1}
\]  

(D5)

which is identical to equation (E8) in Appendix E.

**Log-Expanded Deflection Equations**

The deflection equation (17b) is duplicated below as equation (D6).

\[
y_i = \frac{(\Delta l)^2}{(c_i - c_{i-1})^4} \left[ (c_i - c_{i-1})^2 e_{i-1} - A_i (\Delta l), (c_i - c_{i-1}) c_{i-1}^2 + B_i (\Delta l)^2 c_{i-1}^2 \right] c_i \log \frac{c_i}{c_{i-1}} - (c_i - c_{i-1})
\]  

\[
+ \frac{(\Delta l)^2}{6(c_i - c_{i-1})^2} \left[ 3A_i (\Delta l), (c_i - c_{i-1}) + B_i (\Delta l)^2 (c_i - 4c_{i-1}) \right] + y_{i-1} + (\Delta l) \tan \theta_{i-1}
\]

(D6)

For slowly changing \(c(x)\) [that is, \((c_i/c_{i-1}) \to 1\)], the logarithm term, \(\log \frac{c_i}{c_{i-1}}\), in the deflection equation (18b) can be expanded in series form up to the fourth order term as equation (D7) (ref. 15):

\[
\log \frac{c_i}{c_{i-1}} = \left( \frac{c_i}{c_{i-1}} - 1 \right) - \frac{1}{2} \left( \frac{c_i}{c_{i-1}} - 1 \right)^2 + \frac{1}{3} \left( \frac{c_i}{c_{i-1}} - 1 \right)^3 - \frac{1}{4} \left( \frac{c_i}{c_{i-1}} - 1 \right)^4 + ....
\]

\[
= \frac{c_i - c_{i-1}}{12c_{i-1}^4} \left[ 12c_{i-1}^3 - 6c_{i-1}^2 (c_i - c_{i-1}) + 4c_{i-1} (c_i - c_{i-1})^2 - 3(c_i - c_{i-1})^3 + .... \right]
\]

(D7)

Then the term, \(c_i \log \frac{c_i}{c_{i-1}} - (c_i - c_{i-1})\), in equation (D6) can then be written as:

\[
c_i \log \frac{c_i}{c_{i-1}} - (c_i - c_{i-1}) = \frac{c_i (c_i - c_{i-1})}{12c_{i-1}^4} \left[ 12c_{i-1}^3 - 6c_{i-1}^2 (c_i - c_{i-1}) + 4c_{i-1} (c_i - c_{i-1})^2 - 3(c_i - c_{i-1})^3 + .... \right] - (c_i - c_{i-1})
\]

\[
= \frac{c_i (c_i - c_{i-1})}{12c_{i-1}^4} \left[ 12c_{i-1}^3 - 6c_{i-1}^2 (c_i - c_{i-1}) + 4c_{i-1} (c_i - c_{i-1})^2 - 3(c_i - c_{i-1})^3 \right] - \frac{12c_{i-1}^4}{c_i}
\]

\[
= \frac{c_i (c_i - c_{i-1})}{12c_{i-1}^4} \left[ 12c_{i-1}^3 - 6c_{i-1}^2 (c_i - c_{i-1}) + 4c_{i-1} (c_i - c_{i-1})^2 - 3(c_i - c_{i-1})^3 \right]
\]

\[
= \frac{c_i (c_i - c_{i-1})}{12c_{i-1}^4} \left[ 12c_{i-1}^3 - 6c_{i-1}^2 (c_i - c_{i-1}) + 4c_{i-1} (c_i - c_{i-1})^2 - 3c_i (c_i - c_{i-1})^2 \right]
\]

\[
= \frac{(c_i - c_{i-1})^2}{12c_{i-1}^4} \left[ 12c_{i-1}^3 - 6c_{i-1}^2 (c_i - c_{i-1}) + 4c_{i-1} (c_i - c_{i-1}) - 3c_i (c_i - c_{i-1})^2 \right]
\]

\[
= \frac{(c_i - c_{i-1})^2}{12c_{i-1}^4} \left[ 6c_{i-1}^3 - 6c_{i-1}^2 (c_i - c_{i-1}) + 4c_{i-1} (c_i - c_{i-1}) - 3c_i (c_i - c_{i-1})^2 \right]
\]

\[
= \frac{(c_i - c_{i-1})^2}{12c_{i-1}^4} \left[ 6c_{i-1}^3 + (4c_{i-1} - 6c_{i-1}^2) (c_i - c_{i-1}) - 3c_i (c_i - c_{i-1})^2 \right]
\]

(D8)
In view of equation (D8), equation (D6) can be rewritten as:

\[
y_i = \frac{(\Delta l)^2}{(c_i - c_{i-1})^4} \left[ (c_i - c_{i-1})^2 e_{i-1} - A_i (\Delta l) c_{i-1}(c_i - c_{i-1}) + B_i (\Delta l)^2 c_{i-1}^2 \right] \times \]
\[
\times \frac{(c_i - c_{i-1})^2}{12 e_{i-1}^4} \left[ 6c_{i-1}^3 + c_{i-1}(4c_i - 6c_{i-1})(c_i - c_{i-1}) - 3c_i(c_i - c_{i-1})^2 \right] \]
\[
+ \frac{(\Delta l)^2}{6(c_i - c_{i-1})^2} \left[ 3A_i (\Delta l)(c_i - c_{i-1}) + B_i (\Delta l)^2(c_i - 3c_{i-1}) \right] + y_{i-1} + (\Delta l)_i \tan \theta_{i-1} \]
\[
= \frac{(\Delta l)^2}{12 c_{i-1}(c_i - c_{i-1})^4} \left[ (c_i - c_{i-1})^2 \left[ 6c_{i-1}^3 + c_{i-1}(4c_i - 6c_{i-1})(c_i - c_{i-1}) - 3c_i(c_i - c_{i-1})^2 \right] e_{i-1} \right] \]
\[
- A_i (\Delta l)_i c_{i-1}(c_i - c_{i-1}) \left[ 6c_{i-1}^3 + c_{i-1}(4c_i - 6c_{i-1})(c_i - c_{i-1}) - 3c_i(c_i - c_{i-1})^2 \right] \]
\[
+ B_i (\Delta l)_i^2 c_{i-1}^2 \left[ 6c_{i-1}^3 + c_{i-1}(4c_i - 6c_{i-1})(c_i - c_{i-1}) - 3c_i(c_i - c_{i-1})^2 \right] \]
\[
+ y_{i-1} + (\Delta l)_i \tan \theta_{i-1} \]
\[
= \frac{(\Delta l)^2}{12 c_{i-1}(c_i - c_{i-1})^4} \left[ (c_i - c_{i-1})^2 \left[ 6c_{i-1}^3 + c_{i-1}(4c_i - 6c_{i-1})(c_i - c_{i-1}) - 3c_i(c_i - c_{i-1})^2 \right] e_{i-1} \right] \]
\[
- A_i (\Delta l)_i c_{i-1}(c_i - c_{i-1}) \left[ 6c_{i-1}^3 + c_{i-1}(4c_i - 6c_{i-1})(c_i - c_{i-1}) - 3c_i(c_i - c_{i-1})^2 \right] \]
\[
+ B_i (\Delta l)_i^2 c_{i-1}^2 \left[ 6c_{i-1}^3 + c_{i-1}(4c_i - 6c_{i-1})(c_i - c_{i-1}) - 3c_i(c_i - c_{i-1})^2 \right] \]
\[
+ y_{i-1} + (\Delta l)_i \tan \theta_{i-1} \]
\[
= \frac{(\Delta l)^2}{12 c_{i-1}(c_i - c_{i-1})^4} \left[ \left[ 6c_{i-1}^3 + c_{i-1}(4c_i - 6c_{i-1})(c_i - c_{i-1}) - 3c_i(c_i - c_{i-1})^2 \right] e_{i-1} \right] \]
\[
- A_i (\Delta l)_i c_{i-1} \left[ 6c_{i-1}^3 + c_{i-1}(4c_i - 6c_{i-1})(c_i - c_{i-1}) - 3c_i(c_i - c_{i-1})^2 \right] \]
\[
+ B_i (\Delta l)_i^2 c_{i-1}^2 \left[ 6c_{i-1}^3 + c_{i-1}(4c_i - 6c_{i-1})(c_i - c_{i-1}) - 3c_i(c_i - c_{i-1})^2 \right] \]
\[
+ y_{i-1} + (\Delta l)_i \tan \theta_{i-1} \]
\[
\frac{(\Delta l)^2}{12c_{i-1}^4} \left[ 6c_{i-1}^3 + c_{i-1}(4c_i - 6c_{i-1})(c_i - c_{i-1}) - 3c_i(c_i - c_{i-1})^2 \right] e_{i-1} \\
+ A_i(\Delta l) c_{i-1} \left[ c_{i-1}(4c_i - 4c_{i-1} - 2c_i) - 3c_i(c_i - c_{i-1}) \right] \\
+ B_i(\Delta l)^2 c_{i-1}^2 (4c_{i-1} - 3c_i) \\
= \frac{(\Delta l)^2}{12c_{i-1}^4} \left[ 6c_{i-1}^3 + c_{i-1}(4c_i - 6c_{i-1})(c_i - c_{i-1}) - 3c_i(c_i - c_{i-1})^2 \right] e_{i-1} \\
+ A_i(\Delta l) c_{i-1} \left[ 4c_{i-1}(c_i - c_{i-1}) - 2c_{i-1}^2 - 3c_i(c_i - c_{i-1}) \right] \\
+ B_i(\Delta l)^2 c_{i-1}^2 (4c_{i-1} - 3c_i) \\
= \frac{(\Delta l)^2}{12c_{i-1}^4} \left[ 6c_{i-1}^3 + c_{i-1}(4c_i - 6c_{i-1})(c_i - c_{i-1}) - 3c_i(c_i - c_{i-1})^2 \right] e_{i-1} \\
+ A_i(\Delta l) c_{i-1} \left[ 2c_{i-1}^2 + (3c_i - 4c_{i-1})(c_i - c_{i-1}) \right] \\
+ B_i(\Delta l)^2 c_{i-1}^2 (4c_{i-1} - 3c_i) \\
\] (D9)

Term L in equation (D9) can be rewritten as:

\[
L = \left[ 6c_{i-1}^3 + c_{i-1}(4c_i - 6c_{i-1})(c_i - c_{i-1}) - 3c_i(c_i - c_{i-1})^2 \right] \\
= \left[ 6c_{i-1}^3 + (4c_i c_{i-1} - 6c_{i-1}^2 - 3c_i)(c_i - c_{i-1}) \right] \\
= \left[ 6c_{i-1}^3 + (4c_i c_{i-1} - 6c_{i-1}^2 - 3c_i^2 + 3c_i c_{i-1}) \right] \\
= \left[ 6c_{i-1}^3 + (7c_i c_{i-1} - 6c_{i-1}^2 - 3c_i^2) \right] \\
= \left[ 6c_{i-1}^3 + (7c_i c_{i-1} - 6c_{i-1}^2 - 3c_i^2 - 7c_i c_{i-1} + 6c_{i-1}^2 + 3c_i^2 c_{i-1}) \right] \\
= \left[ 6c_{i-1}^3 + (10c_i^2 c_{i-1} - 13c_i c_{i-1}^2 - 3c_i^3 + 3c_i^3 + 3c_i^3) \right] \\
= \left[ 6c_{i-1}^3 + (10c_i^2 c_{i-1} - 13c_i c_{i-1}^2 + 3c_i^3 - 3c_i^3 - 3c_i^3) \right] \\
= \left[ 6c_{i-1}^3 + (10c_i^2 c_{i-1} - 13c_i c_{i-1}^2 + 3c_i^3 - 3c_i^3 - 3c_i^3) \right] \\
= \left[ 6c_{i-1}^3 + (10c_i^2 c_{i-1} - 10c_i c_{i-1}^2 - 3c_i c_{i-1}^2 + 3c_i^3 - 3c_i^3 - 3c_i^3) \right] \\
= \left[ 6c_{i-1}^3 + (10c_i c_{i-1}(c_i - c_{i-1}) - 3c_i^2 c_{i-1}^2 - 3(c_i^3 - c_{i-1}^3)) \right] \\
= \left[ 6c_{i-1}^3 + c_{i-1}(10c_i - 3c_{i-1})(c_i - c_{i-1}) - 3(c_i^3 - c_{i-1}^3) \right] \\
\] (D10)
In light of equation (D10), equation (D9) takes on the final form as:

\[
y_i = \frac{1}{12c_i^4} \left[ 6c_i^3 + c_i + 10c_i - 3c_i(c_j - c_{j-1}) - 3(c_j - c_{j-1})^3 \right] e_{i-1} + \left( \frac{\Delta l}{\Delta t} \right)^2 \left[ 2c_i^2 + 3c_i(c_j - c_{j-1}) \right] e_{i-1} + (\Delta l) \tan \theta_{i-1}
\]

Equation (D11) is the Log-expanded deflection equation (18b) in the text. For uniform beam \((c_{j-1} = c_j = c)\), equation (D11) becomes equation (D12):

\[
y_i = \frac{(\Delta l)^2}{2c} \left[ e_{i-1} + \frac{A_i}{3} (\Delta l) \right] + (\Delta l) \tan \theta_{i-1}
\]

which is identical to equation (E11) in Appendix E.

**Log-Expanded Displacement Transfer Function**

Combining equations (D4) and (D11) and using similar mathematical steps to Appendix B, one can write:

\[
y_i = \frac{1}{12} \sum_{j=1}^{i} \left( \frac{(\Delta l)^2_j}{c_j^4} \right) \left[ 6c_j^3 + c_j + 10c_j - 3c_j(c_j - c_{j-1}) - 3(c_j - c_{j-1})^3 \right] e_{j-1} + \frac{1}{6} \sum_{j=1}^{i} \left( \frac{(\Delta l)^2_j}{c_j^4} \right) \left[ 6c_j^3 + c_j + 10c_j - 3c_j(c_j - c_{j-1}) - 3(c_j - c_{j-1})^3 \right] e_{j-1} + y_0
\]

\[
+ \left[ \sum_{j=1}^{i} \left( \frac{(\Delta l)^2_j}{c_j^4} \right) \right] \tan \theta_j
\]

Equation (D13) is equation (18c) in the text, the Variable-Domain Log-expanded Displacement Transfer Functions in dual summation form for slightly nonuniform (including uniform) embedded beams based on piecewise-nonlinear strain representations.

**Degeneration into Constant Domain Cases**

For constant-domain length, \((\Delta l)_1 = (\Delta l)_2 = (\Delta l)_3 = \ldots = \Delta l\), coefficients \(A_i\) [eq. (C4)] and \(B_i\) [eq. (C5)] become equations (D14) and (D15):

\[
A_i \equiv \frac{-3e_{i-1} - 4e_j + e_{i+1}}{2(\Delta l)}
\]

\[
B_i \equiv \frac{e_{i-1} - (\Delta l) e_{i+1}}{2(\Delta l)}
\]
**Slope Equation** \([(\Delta l)_i \rightarrow \Delta l]\)

In view of equations (D14) and (D15), slope equation (D4) can be written as:

\[
\tan \theta = \frac{(\Delta l)_i}{12c_{i-1}^3} \left[ 12c_{i-1}^2 \varepsilon_{i-1} + 2(c_i - c_{i-1})(2c_i - 5c_{i-1})\varepsilon_{i-1} + 6A_i(c_i - c_{i-1})^2 \frac{\Delta l}{c_{i-1}^3} - 4A_i (\Delta l) c_{i-1}(c_i - c_{i-1}) + 4B_i(c_{i-1}^3)(\Delta l)^2 \right] + \tan \theta_{i-1}
\]

\[
= \frac{(\Delta l)_i}{c_{i-1}} \left[ \varepsilon_{i-1} + \frac{(c_i - c_{i-1})}{6c_{i-1}^2}(2c_i - 5c_{i-1})\varepsilon_{i-1} + \frac{A_i}{2}(\Delta l) - \frac{A_i}{3c_{i-1}}(c_i - c_{i-1}) + B_i (\Delta l)^2 \right] + \tan \theta_{i-1}
\]

\[
= \frac{\Delta l}{c_{i-1}} \left[ \varepsilon_{i-1} + \frac{(c_i - c_{i-1})}{6c_{i-1}^2}(2c_i - 2c_{i-1} - 3c_{i-1})\varepsilon_{i-1} - \frac{3}{4} \varepsilon_{i-1} + 4 \varepsilon_i + 3 \varepsilon_{i+1} - 4 \varepsilon_i + \varepsilon_{i+1}(c_i - c_{i-1}) + 2 \varepsilon_i + \varepsilon_{i+1} \right] + \tan \theta_{i-1}
\]

\[
= \frac{\Delta l}{c_{i-1}} \left[ \varepsilon_{i-1} - \frac{3}{4} \varepsilon_{i-1} + 3 \varepsilon_i - 4 \varepsilon_i + \varepsilon_{i+1}(c_i - c_{i-1}) \right] + \tan \theta_{i-1}
\]

\[
= \frac{\Delta l}{c_{i-1}} \left[ \varepsilon_{i-1} - \frac{2}{6} \varepsilon_i + \varepsilon_{i+1} \right] + \tan \theta_{i-1}
\]

\[
= \frac{\Delta l}{c_{i-1}} \left[ \varepsilon_{i-1} - \frac{2}{6} \varepsilon_i + \varepsilon_{i+1} \right] + \tan \theta_{i-1}
\]

After grouping terms, equation (D16) becomes:

\[
\tan \theta = \frac{\Delta l}{12c_{i-1}^3} \left[ 5c_{i-1}^2 + 4(c_i - c_{i-1})^2 \right] \varepsilon_{i-1} - 8c_{i-1}(c_i - 2c_{i-1})\varepsilon_i + c_{i-1}(2c_i - 3c_{i-1})\varepsilon_{i+1} \right] + \tan \theta_{i-1}
\]

Equation (D17) is identical to equation (8a) in the text, and thus confirming the mathematical accuracy of the Log-expanded slope equation (D4).
Deflection Equation \([(\Delta l)_i \rightarrow \Delta l]\)

In view of equations (D14) and (D15), deflection equation (D11) can be written as:

\[
y_i = \frac{(\Delta l)^2}{12c^4_{i-1}} \left[ 6c^3_{i-1} + (10c_i c_{i-1} - 3c^2_{i-1})(c_i - c_{i-1}) - 3(c^3_{i} - c^3_{i-1}) \right] + A_i (\Delta l)_i c_{i-1} \left[ 2c^2_{i-1} + (3c_i - 4c_{i-1})(c_i - c_{i-1}) \right] + B_i (\Delta l)^2 c^2_{i-1} \left[ c_{i-1} - 3(c_i - c_{i-1}) \right] + y_{i-1} + (\Delta l)_i \tan \theta_{i-1}
\]

\[
y_i = \frac{(\Delta l)^2}{12c^4_{i-1}} \left[ 6c^3_{i-1} + (10c_i c_{i-1} - 3c^2_{i-1})(c_i - c_{i-1}) - 3(c^3_{i} - c^3_{i-1}) \right] + \left( \frac{3e_{i-1} - 4\varepsilon_i + \varepsilon_{i-1}}{2} \right) c_{i-1} \left[ 2c^2_{i-1} + (3c_i - 4c_{i-1})(c_i - c_{i-1}) \right] + \left( \frac{\varepsilon_{i-1} - 2\varepsilon_i + \varepsilon_{i-1}}{2} \right) c^2_{i-1} \left[ c_{i-1} - 3(c_i - c_{i-1}) \right] + y_{i-1} + (\Delta l)_i \tan \theta_{i-1}
\]

\[
y_i = \frac{(\Delta l)^2}{12c^4_{i-1}} \left[ 12c^3_{i-1} + (20c_i c_{i-1} - 6c^2_{i-1})(c_i - c_{i-1}) - 6(c^3_{i} - c^3_{i-1}) \right] + A_i (\Delta l)_i c_{i-1} \left[ 2c^2_{i-1} + (3c_i - 4c_{i-1})(c_i - c_{i-1}) \right] + B_i (\Delta l)^2 c^2_{i-1} \left[ c_{i-1} - 3(c_i - c_{i-1}) \right] + y_{i-1} + (\Delta l)_i \tan \theta_{i-1}
\]
Substitutions of equations (D19), (D20), (D21) into equation (D18) yields:

\[
\frac{\Delta l^2}{24c_i^4} \left\{ \frac{7c_i^3 + c_i (11c_i + 3c_i) (c_i - c_{t-1}) - 6(c_i^3 - c_{i-1}^3)}{\Delta l} \right\} e_i + y_{i-1} + (\Delta l) \tan \theta_{i-1} + y_{i+1} + (\Delta l) \tan \theta_{i+1}
\]

Ther terms \(F_1, F_2, F_3\) in equation (D18) can be rewritten as:

\[
F_1 = \frac{\Delta l^2}{24c_i^4} \left\{ \frac{7c_i^3 + c_i (11c_i + 3c_i) (c_i - c_{t-1}) - 6(c_i^3 - c_{i-1}^3)}{\Delta l} \right\} e_i + y_{i-1} + (\Delta l) \tan \theta_{i-1} + y_{i+1} + (\Delta l) \tan \theta_{i+1}
\]

\[
F_2 = c_{i-1} \left[ 6c_i^2 + (12c_i - 10c_{i-1}) (c_i - c_{t-1}) \right]
\]

\[
F_3 = -c_{i-1} \left[ c_i^2 + (3c_i - c_{i-1}) (c_i - c_{t-1}) \right]
\]

Substitutions of equations (D19), (D20), (D21) into equation (D18) yields:

\[
y_i = \frac{\Delta l^2}{24c_i^4} \left\{ \frac{7c_i^3 + c_i (11c_i + 3c_i) (c_i - c_{t-1}) - 6(c_i^3 - c_{i-1}^3)}{\Delta l} \right\} e_i + y_{i-1} + (\Delta l) \tan \theta_{i-1} + y_{i+1} + (\Delta l) \tan \theta_{i+1}
\]
Equation (D22) is identical to equation (8b) in the text, and thus confirming the mathematical accuracy of the Log expanded deflection equation (D11).

**Displacement Transfer Function** \([((\Delta l), \rightarrow \Delta l)]\)

Combining equations (D17) and (D22), and after grouping terms, one obtains the displacement transfer function in dual summation form as:

\[
y_i = \frac{(\Delta l)^2}{24} \sum_{j=1}^{i} \frac{1}{\epsilon_{j+1}} \left[ \frac{7c_j^3 + (c_j - c_{j+1})(8c_j + 3c_j^3 - 3(3c_j^3 - c_{j+1}^3))}{e_{j+1}} \right] + y_0
\]

\[
+ \frac{(\Delta l)^2}{12} \sum_{j=1}^{i} \frac{1}{\epsilon_{j+1}} \left[ \frac{5c_j^2 + 4(c_j - c_{j+1})^2}{e_{j+1}} - 8c_{j+1}(c_j - 2c_{j+1})e_j \right] + (i)\Delta l \tan \theta_0
\]

Equation (D23) is identical to equation (8c) in the text, the Constant-Domain Log-expanded Displacement Transfer Functions in dual summation form for slightly nonuniform (including uniform) embedded beams based on piecewise-nonlinear strain representations.
APPENDIX E
DERIVATIONS OF VARIABLE-DOMAIN IMPROVED DISPLACE TRANSFER FUNCTIONS FOR UNIFORM BEAMS
(Based on Piecewise-Nonlinear Strain Representations)

Appendix E presents the derivations of the slope and deflection equations for the special case of the uniform beam based on piecewise-nonlinear strain representations.

\textbf{Slope Equations} \([c(x) = c]\)

The slope, \(\tan \theta(x)\), of the uniform beam \([c(x) = c]\) in the small domain, \(x_{i-1} \leq x \leq x_i\), between the two adjacent strain-sensing stations, \(\{x_{i-1}, x_i\}\), can be obtained by integrating both sides of equation (E1):

\[\int_{x_{i-1}}^{x_i} \frac{d^2 y}{dx^2} = \frac{dy}{dx} \left( \frac{d\theta}{dx} \right)_{i-1} = \int_{x_{i-1}}^{x_i} \frac{\varepsilon(x)}{c} \, dx ; \quad (x_{i-1} \leq x \leq x_i) \quad (E1)\]

The piecewise-nonlinear equation (16) representing the bending strain, \(\varepsilon(x)\), within the small domain, \(x_{i-1} \leq x \leq x_i\), between the two adjacent strain-sensing stations, \(\{x_{i-1}, x_i\}\), is duplicated below as equations (E2) – (E5):

\[\varepsilon(x) = \varepsilon_{i-1} - \frac{(\Delta l)_{i+1} [2(\Delta l)_i + (\Delta l)_{i+1}] \varepsilon_{i-1} - [(\Delta l)_i + (\Delta l)_{i+1}]^2 \varepsilon_i + (\Delta l)^2 \varepsilon_{i+1}(x - x_{i-1})}{(\Delta l)_i(\Delta l)_{i+1}[(\Delta l)_i + (\Delta l)_{i+1}]} + \frac{(\Delta l)_{i+1} \varepsilon_{i-1} - [(\Delta l)_i + (\Delta l)_{i+1}] \varepsilon_i + (\Delta l)_i \varepsilon_{i+1}(x - x_{i-1})^2}{(\Delta l)_i(\Delta l)_{i+1}[(\Delta l)_i + (\Delta l)_{i+1}]} \quad (E2)\]

\[(x_{i-1} \leq x \leq x_i)\]

Let,

\[A_i = -\frac{(\Delta l)_{i+1} [2(\Delta l)_i + (\Delta l)_{i+1}] \varepsilon_{i-1} - [(\Delta l)_i + (\Delta l)_{i+1}]^2 \varepsilon_i + (\Delta l)_i \varepsilon_{i+1}}{(\Delta l)_i(\Delta l)_{i+1}[(\Delta l)_i + (\Delta l)_{i+1}]} \quad (E3)\]

\[B_i = \frac{(\Delta l)_{i+1} \varepsilon_{i-1} - [(\Delta l)_i + (\Delta l)_{i+1}] \varepsilon_i + (\Delta l)_i \varepsilon_{i+1}}{(\Delta l)_i(\Delta l)_{i+1}[(\Delta l)_i + (\Delta l)_{i+1}]} \quad (E4)\]

\[\xi = (x - x_{i-1}) \quad (E5)\]
In view of equation (E2), (E3), (E4), and (E5), the last three terms of equation (E1) can be written as:

\[
\tan \theta(x) = \frac{1}{c} \int_0^x (A_i \xi + B_i \xi^2) d\xi + \tan \theta_{i-1} \quad ; \quad [0 \leq \xi \leq (\Delta l)_i]
\]  

(E6)

After carrying out integration of equation (E6), one obtains (ref. 15):

\[
\tan \theta(x) = \frac{\xi}{c} \left[ \frac{A_i}{2} \xi + \frac{B_i}{3} \xi^2 \right] + \tan \theta_{i-1} \quad ; \quad [0 \leq \xi \leq (\Delta l)_i]
\]  

(E7)

At the strain-sensing station, \( x_i \), we have \( \xi = x_i - x_{i-1} = (\Delta l)_i \), then equation (E7) gives the slope, \( \tan \theta_i = \tan \theta(x_i) \) at \( x_i \) as:

\[
\tan \theta_i = \frac{(\Delta l)_i}{c} \left[ \frac{A_i}{2} (\Delta l)_i + \frac{B_i}{3} (\Delta l)_i^2 \right] + \tan \theta_{i-1} \quad ; \quad (i = 1, 2, 3, \ldots, n)
\]  

(E8)

Equation (E8) is the slope equation in recursion form.

**Deflection Equation** \([c(x) = c]\)

Deflection can be obtained by integrating the slope equation (E7) as:

\[
y(x) = \int_0^x \tan \theta(\xi) d\xi + y_{i-1} = \frac{1}{c} \int_0^x \left[ \frac{A_i}{2} \xi + \frac{B_i}{3} \xi^2 \right] d\xi + \tan \theta_{i-1} \quad ; \quad [0 \leq \xi \leq (\Delta l)_i]
\]  

(E9)

Carrying out integration of equation (E9), one obtains:

\[
y(x) = \frac{\xi^2}{2c} \left[ \frac{A_i}{3} \xi + \frac{B_i}{6} \xi^2 \right] + y_{i-1} + \xi \tan \theta_{i-1} \quad ; \quad [0 \leq \xi \leq (\Delta l)_i]
\]  

(E10)

At the strain-sensing station, \( x_i \), we have \( \xi = x_i - x_{i-1} = (\Delta l)_i \), and the deflection, \( y_i = y(x_i) \) at \( x_i \), is given by equation (E10) in the form:

\[
y_i = \frac{(\Delta l)_i^2}{2c} \left[ \frac{A_i}{3} (\Delta l)_i + \frac{B_i}{6} (\Delta l)_i^2 \right] + y_{i-1} + (\Delta l)_i \tan \theta_{i-1} \quad ; \quad (i = 1, 2, 3, \ldots, n)
\]  

(E11)

Equation (E11) is the deflection equation in recursion form.

**Displacement Transfer Function** \([c(x) = c]\)

Substitution of slope equation (E8) into deflection equation (E11), and following the mathematical steps presented in Appendix B, one obtains the deflection equation written in dual summation form as:
Equation (E12) is called the variable-domain Improved Displacement Transfer Function in dual summation form for uniform beams based on piecewise-nonlinear strain representations.

**Degeneration into Constant-Domain Cases**

For constant-domain length, \( (\Delta l)_1 = (\Delta l)_2 = (\Delta l)_3 = \ldots = \Delta l \), coefficients \( A_i \) [eq. (E3)] and \( B_i \) [eq. (E4)] become equations (E13) and (E14):

\[
A_i = -\frac{(\Delta l)_{i+1} [2(\Delta l)_i + (\Delta l)_{i+1}] e_i - [(\Delta l)_i + (\Delta l)_{i+1}]^2 e_i + (\Delta l)^2 e_{i+1}}{(\Delta l)_i [((\Delta l)_i + (\Delta l)_{i+1})^2 - 3e_i - 4e_i + e_{i+1}]} \quad (E13)
\]

\[
B_i = \frac{(\Delta l)_{i+1} e_i - [(\Delta l)_i + (\Delta l)_{i+1}] e_i + (\Delta l) e_{i+1}}{(\Delta l)_i [((\Delta l)_i + (\Delta l)_{i+1})^2 - 2e_i + e_{i+1}]} \quad (E14)
\]

**Slope equation \([c(x) = c]\):**

In view of equations (E13) and (E14), slope equation (E8) can be written as:

\[
\tan \theta_i = \frac{\Delta l}{c} \left[ e_{i-1} + \frac{A_i}{2} (\Delta l)_i + \frac{B_i}{3} (\Delta l)^2_i \right] + \tan \theta_{i-1}
\]

\[
= \frac{\Delta l}{c} \left[ e_{i-1} + \left( \frac{3e_i - 4e_i + e_{i+1}}{2\Delta l} \right) \frac{\Delta l}{2} + \left( \frac{e_i - 2e_i + e_{i+1}}{2(\Delta l)^2} \right) \frac{(\Delta l)^2}{3} \right] + \tan \theta_{i-1}
\]

\[
= \frac{\Delta l}{12c} \left[ 12e_{i-1} - 3(3e_i - 4e_i + e_{i+1}) + 2(e_i - 2e_i + e_{i+1}) \right] + \tan \theta_{i-1}
\]

\[
= \frac{\Delta l}{12c} \left[ 12e_{i-1} - 9e_{i-1} + 12e_i - 3e_{i+1} + 2e_{i-1} - 4e_i + 2e_{i+1} \right] + \tan \theta_{i-1} \quad (E15)
\]

After grouping terms, equation (E15) becomes:

\[
\tan \theta_i = \frac{\Delta l}{12c} (5e_{i-1} + 8e_i - e_{i+1}) + \tan \theta_{i-1} \quad ; \quad (i = 1, 2, 3, \ldots, n) \quad (E16)
\]

Equation (E16) is the degenerated form of equation (8a) in the text for \( c_{i-1} = c_i = c \).
Deflection Equation \([c(x) = c]\)

In view of equations (E13) and (E14), deflection equation (E11) becomes:

\[
y_i = \frac{(\Delta l)^2}{2c} \left[ e_{i-1} + \frac{A_i}{3} (\Delta l)_i + \frac{6}{B_i} (\Delta l)^2 \right] + y_{i-1} + (\Delta l) \tan \theta_{i-1}
\]

\[
= \frac{(\Delta l)^2}{2c} \left[ e_{i-1} + \left( -\frac{3e_{i-1} - 4e_i + e_{i+1}}{2\Delta l} \right) \frac{\Delta l}{A_i} + \left( \frac{e_{i-1} - 2e_i + e_{i+1}}{2(\Delta l)^2} \right) \frac{(\Delta l)^2}{B_i} \right] + y_{i-1} + \Delta l \tan \theta_{i-1}
\]

\[
= \frac{(\Delta l)^2}{24c} \left[ 12e_{i-1} - 2(3e_{i-1} - 4e_i + e_{i+1}) + (e_{i-1} - 2e_i + e_{i+1}) \right] + y_{i-1} + \Delta l \tan \theta_{i-1}
\]

\[
= \frac{(\Delta l)^2}{24c} \left[ 12e_{i-1} - 6e_{i-1} + 8e_i - 2e_{i+1} + e_{i-1} - 2e_i + e_{i+1} \right] + y_{i-1} + \Delta l \tan \theta_{i-1}
\]

(E17)

After grouping terms, equation (E17) becomes:

\[
y_i = \frac{(\Delta l)^2}{24c} (7e_{i-1} + 6e_i - e_{i+1}) + y_{i-1} + \Delta l \tan \theta_{i-1} \quad ; \quad (i = 1, 2, 3, \ldots, n)
\]

(E18)

Equation (E18) is the degenerated form of equation (8b) in the text for \(c_{i-1} = c_i = c\).

Displacement Transfer Function \([c(x) = c]\)

Combining equations (E16) and (E18) into one equation, and after grouping terms, one obtains:

\[
y_i = \frac{(\Delta l)^2}{24c} \sum_{j=1}^{i} (7e_{j-1} + 6e_j - e_{j+1}) + y_0 + \frac{(\Delta l)^2}{12c} \sum_{j=1}^{i-1} [(i-j)(5e_{j-1} + 8e_j - e_{j+1}) + (i)\Delta l \tan \theta_b]
\]

(E19) \hspace{1cm} (Contributions from deflection terms) \hspace{1cm} (Contributions from slope terms)

\(i = 1, 2, 3, \ldots, n\)

Equation (E19) is the degenerated form of equation (8c) in the text for \(c_{i-1} = c_i = c\).

Special Case

For a special case when the strain distribution along the uniform beam is linear, one can write equation (E20),

\[
e_{i+1} - e_i = e_i - e_{i-1}
\]

(E20)

or equation (E21),

\[
e_{i+1} = 2e_i - e_{i-1}
\]

(E21)
In light of equation (E21), the slope equation (E16) becomes:

$$\tan \theta_i = \frac{\Delta l}{12c} (5\varepsilon_{j-1} + 8\varepsilon_i - \varepsilon_{i+1}) + \tan \theta_{i-1}$$

$$= \frac{\Delta l}{12c} [5\varepsilon_{j-1} + 8\varepsilon_i - (2\varepsilon_j - \varepsilon_{j+1})] + \tan \theta_{i-1}$$  \hspace{1cm} (E22)

After grouping terms, equation (E22) becomes:

$$\tan \theta_i = \frac{\Delta l}{2c} (\varepsilon_{i-1} + \varepsilon_i) + \tan \theta_{i-1} \hspace{0.5cm}; \hspace{0.5cm} (i = 1, 2, 3, \ldots, n)$$  \hspace{1cm} (E23)

Equation (E23) is the degenerated form of equation (6a) for $c_{i-1} = c_i = c$.

Also, in view of equation (E21), the deflection equation (E18) becomes:

$$y_i = \frac{(\Delta l)^2}{24c} (7\varepsilon_{j-1} + 6\varepsilon_i - \varepsilon_{i+1}) + y_{i-1} + \Delta l \tan \theta_{i-1}$$

$$= \frac{(\Delta l)^2}{24c} [7\varepsilon_{j-1} + 6\varepsilon_i - (2\varepsilon_j - \varepsilon_{j+1})] + y_{i-1} + \Delta l \tan \theta_{i-1}$$  \hspace{1cm} (E24)

After grouping terms, equation (E24) becomes:

$$y_j = \frac{(\Delta l)^2}{6c} (2\varepsilon_{i-1} + \varepsilon_i) + y_{i-1} + \Delta l \tan \theta_{i-1} \hspace{0.5cm}; \hspace{0.5cm} (i = 1, 2, 3, \ldots, n)$$  \hspace{1cm} (E25)

Equation (E25) is the degenerated form of equation (6b) for $c_{i-1} = c_i = c$.

Finally, in view of equation (E21), the deflection equation (E19) becomes:

$$y_i = \frac{(\Delta l)^2}{24c} \sum_{j=1}^{i} (7\varepsilon_{j-1} + 6\varepsilon_j - \varepsilon_{j+1}) + y_0 + \frac{(\Delta l)^2}{12c} \sum_{j=1}^{i-1} [(i-j)(5\varepsilon_{j-1} + 8\varepsilon_j - \varepsilon_{j+1})] + (i)\Delta l \tan \theta_0$$

$$= \frac{(\Delta l)^2}{24c} \sum_{j=1}^{i} [7\varepsilon_{j-1} + 6\varepsilon_j - (2\varepsilon_j - \varepsilon_{j+1})] + y_0$$

$$+ \frac{(\Delta l)^2}{12c} \sum_{j=1}^{i-1} [(i-j)(5\varepsilon_{j-1} + 8\varepsilon_j - (2\varepsilon_j - \varepsilon_{j+1})] + (i)\Delta l \tan \theta_0 \hspace{1cm} (E26)$$
After grouping terms, equation (E26) becomes:

\[
y_i = \frac{(\Delta l)^2}{6c} \sum_{j=1}^{i} (2\varepsilon_{j-1} + \varepsilon_j) + y_0 + \frac{(\Delta l)^2}{2c} \sum_{j=1}^{i} [(i-j)(\varepsilon_{j-1} + \varepsilon_j)] + (i)\Delta l \tan \theta_0 \]

(E27)

\[
(i = 1,2,3,\ldots,n)
\]

Equation (E27) is the degenerated form of equation (6c) for \( c_{i-1} = c_i = c \).
REFERENCES


