Using Dispersed Modes During Model Correlation

Eric Stewart, Ph.D
Megan Hathcock
Outline

◆ **Introduction**
  • Dispersions
  • Best Model Estimate

◆ **Model Dispersions**
  • Using MAT cards
    – Uncertainty distributions
  • Dispersion-to-test comparison metrics
    – Frequency
    – Mode shapes – MAC vs XOR
  • Pareto front
  • Using Attune

◆ **Examples**
  • TAURUS
  • Ares I-X

◆ **Conclusions/Future Work**
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- Nicholas Mastramico (MSFC/EV31)
- Dave McGhee (MSFC/EV31)
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**Langley Research Center**
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- Mercedes Reaves (LaRC/D322)

**Johnson Space Center**
- George James (JSC/ES611)
- Rodney Rocha (JSC/ES611)
Model correlation procedure typically takes several months to complete for a large modal test
• SLS timeframe only allows for less than 2 weeks

Best Model Estimate (BME) process developed to perform a QUICK model correlation
• Model that most closely represents test data
  1. Prior to test, create thousands of pretest model dispersions
  2. Post test, select one dispersion that best matches the test data → provides a coarse optimization
  3. Using single iteration of Attune (from ATA Engineering), perform optimization to further refine model

BME is a QUICK model tuning effort and not a full model correlation
• Full model correlation will occur after test but will not finish until much later
BME Creation - Dispersions

- Candidate set is created with dispersions of the nominal test configuration model
  - 1000’s of dispersions are created ahead of the test
    - Enables the quick turnaround

- Each of the model dispersions are compared against test data to find the dispersion that most closely matches test
  - Mode frequencies
  - Mode shapes

- The best match is further refined using optimization tool
  - Refined model is BME

- Presentation will show two examples of the BME method
  - TAURUS-T model
  - Ares I-X
Introduction
- Dispersions
- Best Model Estimate

Model Dispersions
- Using MAT cards
  - Uncertainty distributions
- Dispersion-to-test comparison metrics
  - Frequency
  - Mode shapes – MAC vs XOR
- Pareto front
- Using Attune

Examples
- TAUROS
- Ares I-X

Conclusions/Future Work
Model Dispersions – Evaluation

- **Uncertainty factors applied to NASTRAN bulk data file MAT cards (elastic modulus or spring stiffness)**
  - Perform sensitivity work to select variables
- **Create dispersions by perturbing nominal**
  - MAT cards: uniform distribution of ±20%
  - Create many dispersions of the test model to cover the design space
    - Limited by computational resources and time
  - Group parameters to reduce number of variables
- **Dispersions are compared against two objectives**
  - Frequency error
    - Average absolute relative error between selected test modes and analysis modes
  - Mode shape error
    - Norm of XOR/MAC error (accuracy of eigenvectors)
      - Euclidean norm
      - Root sum square of the diagonals (used in future)

\[
g_1 = \frac{1}{N} \sum_{i=1}^{N} W_i \left| \frac{f_{i,\text{test}} - f_{i,\text{model}}}{f_{i,\text{test}}} \right| \times 100
\]

\[
g_2 = \| [I] \{W \} - [MAC] \{W \} \|
\]
Select model dispersions that best matches data
- For this example, “test data” is a dispersed model
- Selection based on two “objectives”
  - Frequency error
  - Mode shape error
- Will likely lead to best set of dispersions → Pareto Front

Pareto front is a natural outcome of multiobjective optimization
No solution is THE optimal solution
Set of non-dominated optimal solutions
Pareto BME discrimination factors
- Closeness to nominal model
- How models do when different weighting factors are used
- Using different mode shape error norm

\[
g_1 = \sum_{i=1}^{f<6Hz} W_i \left| \frac{f_{i\,test} - f_{i\,model}}{f_{i\,test}} \right|
\]
\[
g_2 = \| [I] \{W\} - [MAC]\{W\} \|
\]
After selection of the best single dispersion, Attune is used for further model tuning:

- Use same mode shapes and frequencies as used in the model dispersion process.
- Variables and groups will be the same as used in model dispersions.
- Upper and lower bounds on variables will be closer to nominal than the dispersion uniform distributions.
  - Attune rewards these designs in the objective function.
- Use of cross-orthogonality or MAC built in.
- Can weight or de-weight modes as desired.
  - Will keep same weights as dispersions.
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Examples of BME Process

The BME dispersion process has been implemented on three models of increasing rocket-ness

- ET40 cart model *(in back-up charts)*
  - EV31 intern, Megan Hathcock, built a finite element model, performed modal tests, and correlated the nominal model to the test data
  - Simple, already have test data, have correlated model

- JSC TAURUS-T
  - Correlated model and model description from Rodney Rocha
  - Test data provided by Michael Grygier
  - Simple, test data provided
  - Sent a tuned version of the TAURUS-T back to JSC for use in the transient response method

- Ares I-X
  - Actual launch vehicle with test data
  - Nominal model and optimized model provided by Mercedes Reaves at LaRC
  - Test data provided by Dan Lazor (primary curve fitter for I-X)
Example: TAURUS

- **Test Article Unit for Rectified Utility Systems Testing (TAURUS-T)**
  - Unistrut structure 70” wide and 80” in height
  - Bolted to floor (rough approximation of cantilever)
  - Joints modeled with springs since unistrut bar shear centers do not quite align
  - Joint stiffness approximated by modeling the brackets that connect the bars

- **Test modes come from hammer tests**
- **Correlated model provided**
- **Nominal model created from correlated model by changing joint springs**
  - Correlated model falls within the range of uncertainties during dispersion process
Comparing to the first 20 test modes
RSS norm of the mode shape error used as tie-breaker for Pareto front
• Dispersion 372 chosen as starting point for Attune
Dispersions outperform hand-tuned model and the nominal model after one iteration of Attune
- Optimizer makes the model “worse” by one metric while improving it in another metric
- Attune using different metrics than when we started
  - Need to be consistent
**Example: Ares I-X**

**Ares I-X**
- Actual launch vehicle with test data
- Nominal model provided by Mercedes Reaves at LaRC
- Test data provided by Dan Lazor (primary curve fitter for I-X)

**Able to compare BME dispersion process to the optimized Ares I-X model found in the literature**
  - Used same variables as Horta, et al.

**Used first seven test modes in the dispersion comparisons**
- Torsion modes and axial modes difficult to match to test data

**Used XOR instead of MAC**
- Attempting different mode shape metrics
Example: Ares I-X Variables

- Used same variables as Horta, et al.

- Young’s modulus of different parts of the vehicle
  - ±20% uniform distribution

- Upper stage simulator springs
  - ±100% uniform distribution
Using Euclidean norm, the Pareto front outperforms the nominal model and the hand-tuned model.

Using Attune, the Pareto designs get “worse.”

Using the RSS norm of the XOR shows that the optimized model of Horta, et al. is Pareto-optimal and is on par with dispersions + Attune.

Seems to be a limit on how well the Ares I-X model can be tuned.
Since we only target a few modes to select the BME, what is the effect on the non-target modes that may have an effect on the loads analysis?

During BME process, only first 7 test modes are kept for the Ares I-X comparisons, so we compare test modes 8-14
- 5 of 7 modes improve in frequency error, only marginally worse for other two
- 4 of 7 improve XOR diagonal, worst case is decrease from 0.80 to 0.74 (test mode 13)

Need to check for other models, but maybe rising tide raises all boats
- Improve the physics for some modes, improve the physics for all modes?

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◆ Conclusions/Future Work
Conclusions and Future Work

◆ Using model dispersions as a starting point allows us to quickly adjust a model to reflect new test data
  • The analyst does a lot of work before the test to save time post-test
  • Creating 1000s of model dispersions to provide “coarse tuning,” then use Attune to provide the “fine tuning”

◆ Successful model tuning on three structures
  • TAURUS
  • Ares I-X
  • Cart (in backup charts)

◆ Mode weighting factors, matrix norm method, and XOR vs. MAC all play key roles in determining the BME

◆ The BME process will be used on future tests
  • ISPE modal test (ongoing work)
  • SLS modal test (mid 2018)
BACK UP
**Model Dispersions – Overview**

- **Uncertainty factors applied to bulk data file M* cards (Young’s modulus or spring stiffness)**
  - Piggyback off of prior sensitivity work to select variables

- **Create dispersions by applying uncertainty factors**
  - M* cards have uniform distribution of ±20% of nominal values
  - Groups of parameters used to reduce number of variables
  - Create many dispersions for the test
    - Goal is 5000 dispersions for each structure
    - Limited by computational resources and time

- **Future Work**
  - Optimal Latin Hypercube design of experiments used to generate the dispersion designs
  - Investigating how to augment the LHS design space if we get extra time or computational power

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<th>LVSA</th>
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Model Dispersions – Tracking modes

Frequency error calculation
- Frequency error determined from mode shape matching

\[ MAC = \frac{(\phi_t^T \phi_m)^2}{(\phi_t^T \phi_t)(\phi_m^T \phi_m)} \]
\[ XOR = \phi_t^T M_{AA} \phi_m \]

Mode shape error calculation
- Calculation of an initial MAC or XOR comparison
  - Note: Guyan mass matrix pulled out of model using Attune DMAP
  - MAC matrices are sorted to keep highest values on diagonal
    - Use in-house codes sortMAC.m and sortXOR.m
    - Details in back-up charts

\[ g_1 = \frac{1}{N} \sum_{i=1}^{N} \left( W_i \frac{|f_{i\text{test}} - f_{i\text{model}}|}{f_{i\text{test}}} \right) \times 100 \]

<table>
<thead>
<tr>
<th>MAC Original Cart Model Sorted</th>
<th>Model Modes</th>
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<td>MAC Original Cart Model</td>
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<td>Freq (Hz)</td>
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</table>
MAC = \frac{(\varphi_T' \ast \varphi_M)^2}{\varphi_T' \ast \varphi_T \ast \varphi_M' \ast \varphi_M}

MAC matrices are sorted to keep the highest values along the diagonal.

### Test Modes

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<th>Freq (Hz)</th>
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<th>16.1</th>
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### MAC Original Model

MAC matrices are sorted to keep the highest values along the diagonal.
sortMAC.m identifies the maximum value \( i(x) \) in each column, returns the row position, and creates a duplicate MAC matrix.

It then compares row positions and looks for repeats \( (\) \). In this case both 3 and 6 have duplicates.

It compares the duplicate values and zeroes \( (\) \) out the smaller one in the duplicate MAC matrix.

The function then finds the max in each column again and repeats the process until there are no duplicates in the max position array \( (\) \).
Once there are no duplicates, the function creates a sorting array by concatenating the model mode numbers and max position arrays.

The function then sorts the sorting array by max position and then arranges the original MAC matrix and model frequencies by the Model Modes matrix.
Second example for BME process is small cart in ET40

- Initially chosen as a way to test out EV31 operational modal analysis codes
- Model developed and correlated in-house

- Hand-tuning process took 300+ iterations to get final model
- Dispersions + Attune process saves time and effort while achieving similar results
First nominal model mode (7.5 Hz) is wheel rotation mode and not captured in test due to lack of sensors
Similarly, second model mode (9.1 Hz) is wheel mode
Modes 3-10 used to compare dispersions to test data
Data from roving accelerometer test with cart in “grounded” configuration
• Wheels are constrained to prevent motion during testing (duct tape and bolts)

<table>
<thead>
<tr>
<th>Mode #</th>
<th>Nominal Model Modes (Hz)</th>
<th>Hand-Tuned Model Modes (Hz)</th>
<th>Test Modes (Hz)</th>
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**BME Example: ET40 Cart Variables**

### Cart Material Properties

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**Dispersed ± 20%**

### Cart Physical Properties

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<td>in.</td>
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<tr>
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<td>Cyan</td>
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**Dispersed ± 30%**
**BME Example: ET40 Cart Variables**

### Cart Physical Properties

<table>
<thead>
<tr>
<th>Color</th>
<th>Property</th>
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<th>Units</th>
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<tr>
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<td>Red</td>
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**Dispersed ± 30%**

### Top Bar Springs

- K1: 1.00E+02 lb/in.
- K2: 1.00E+02 lb/in.
- K3: 1.00E+02 lb/in.
- K4: 1.00E+02 lb-in/rad
- K5: 1.00E+02 lb-in/rad
- K6: 1.00E+02 lb-in/rad

**Dispersed ± 100%**

### Back Wheel Springs

- K1: 1.00E+03 lb/in.
- K2: 1.00E+03 lb/in.
- K3: 1.00E+03 lb/in.
- K4: 1.00E+04 lb-in/rad
- K5: 1.00E+04 lb-in/rad
- K6: 1.00E+04 lb-in/rad

**Dispersed ± 100%**

### Front Wheel Springs

- K1: 1.00E+03 lb/in.
- K2: 1.00E+03 lb/in.
- K3: 1.00E+03 lb/in.
- K4: 1.00E+03 lb-in/rad
- K5: 1.00E+03 lb-in/rad
- K6: 1.00E+03 lb-in/rad

**Dispersed ± 100%**

---

**www.nasa.gov/sls**
Starting Pareto points chosen by engineering judgment
• Chose points with good MAC to avoid mode swapping

Hand-tuned model performs the best
Pareto points + 1 iteration of Attune (magenta) perform nearly as well as the hand tuned model while saving time
Nominal model with three iterations of Attune is dominated by the Pareto+1 Attune points
The range of joint spring stiffnesses chosen such that the correlated model is a possible dispersion model
3000 dispersions were created by applying an uncertainty factor between 0.05-20 for each of the spring degrees of freedom
Young’s modulus and density uncertainty factors between 0.95-1.05

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Example: Ares I-X XOR vs MAC

Using MAC instead of cross-orthogonality will change the Pareto front.