High Efficiency Nuclear Power Plants Using Liquid Fluoride Thorium Reactor Technology

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This report contains preliminary findings, subject to revision as analysis proceeds.

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Abstract
An overall system analysis approach is used to propose potential conceptual designs of advanced terrestrial nuclear power plants based on Oak Ridge National Laboratory (ORNL) Molten Salt Reactor (MSR) experience and utilizing Closed Cycle Gas Turbine (CCGT) thermal-to-electric energy conversion technology. In particular conceptual designs for an advanced 1 GWe power plant with turbine reheat and compressor intercooling at a 950 K turbine inlet temperature (TIT), as well as near term 100 MWe demonstration plants with TITs of 950 and 1200 K are presented. Power plant performance data were obtained for TITs ranging from 650 to 1300 K by use of a Closed Brayton Cycle (CBC) systems code which considered the interaction between major sub-systems, including the Liquid Fluoride Thorium Reactor (LFTR), heat source and heat sink heat exchangers, turbo-generator machinery, and an electric power generation and transmission system. Optional off-shore submarine installation of the power plant is a major consideration.

Introduction
In meeting the increasing demand for electrical energy today’s global economies are faced with the dual problem of declining fossil fuel resources (Hubbert 1956) and climate change due to atmospheric accumulation of “greenhouse gases,” principally CO2 and methane. An obvious solution to both issues would be a power generation process that does not require fossil fuels and also does not have any gas emissions. Among the proposed near term “alternative energy” sources, the reliability and capacity factor of traditional nuclear fission power plants has steadily improved over the years to a level of ~ 92 percent, which is more than twice that of “solar” or “wind”. Additional benefits from nuclear power are possible, if investment in “nontraditional” nuclear power generation is undertaken. The inherent advantages of such advanced power generation schemes were recognized by the United States Congress when it passed the “Energy Policy Act of 2005” (U.S. 109th Congress, 2005). Development of advanced nuclear power plants was advocated under “Title VI—Nuclear Matters” and the goals of the “Generation IV Nuclear Energy Systems Initiative” were spelled out under Subtitle C “Next Generation Nuclear Plant Project” (NGNPP). In essence, these goals were to generate electric power for base load energy demands and to produce hydrogen as a new carbon free fuel for vehicular transportation. Furthermore, Generation IV (Gen IV) power plants were to be highly economical, equipped with safety enhancements, have minimal waste, and be proliferation resistant. To meet these objectives a number of “closed cycle gas turbine” (CCGT) energy conversion systems either directly coupled to high temperature gas (cooled) reactors (HTGR), (Richards et al., 2005), or indirectly coupled via intermediate heat exchangers (IHX) to liquid cooled reactors have been proposed. For both configurations the gas turbine working fluid is helium (He), with power plant output ranging from tens of megawatts (MW) to gigawatt (GW) levels.
A good comparison of the performance of power plants using either gas turbine (Brayton), or steam turbine (Rankine) energy conversion systems, in terms of the thermodynamic plant efficiency is shown in Figure 1, which was adapted from the literature (LaBar 2002), except for the abscissa coordinates altered from “F” to “K”. Due to the higher cycle temperature ratios enabled by the higher turbine inlet temperatures for gas turbine systems, a 50 percent increase in CCGT plant efficiency can be realized, when compared to the highest efficiency achievable with the steam cycle. Hence most Gen IV energy conversion systems are based on the CCGT power cycle, also referred to as the Closed Brayton Cycle (CBC). For optimum rotor-dynamic performance vertical orientation of the compressor- turbo-alternator machinery has been proposed (Zhao and Peterson 2005, Kodochigov 2008). Acronyms like High Temperature Gas Reactor (HTGR) or Gas Turbine Modular Helium Reactor (GT-MHR) (Baxi, 2006, 2009) refer to the directly CBC systems, while CBC energy conversion via the Very High Temperature Reactor (VHTR), MSR(Molten Salt Reactor), or LFTR (Liquid Fluoride Thorium Reactor) refer to indirectly heated cycles. Note that, due to the much higher heat transfer capability of liquid (molten) salt or metal, a VHTR can operate at higher outlet temperatures than the HTGR. The drawback is that additional investment in liquid-to-gas heat exchangers and circulating pumps must be made. However, such investment may be warranted if one considers that for a 1000 MWe plant generating power at 5 c/KW-hr ($50/MW-hr) each percent increase in plant efficiency translates into nearly $4.5 M in additional revenue.

Hence the objective of this paper is to examine how gas turbine power systems could use fission reactor heat sources based on the LFTR technology, developed at ORNL (Engel, 1980) during the “Molten Salt Reactor Experiment” (MSRE) program.

The Thorium Fuel Cycle and LFTR Power Plant

The Thorium fuel cycle is based on a series of neutron absorption and beta decay processes initiated by neutron absorption and beta decay reactions starting with naturally occurring thorium-232 as the fertile material and the artificial uranium-233 (\(\alpha_{233} U\)) isotope as the fissile reactor fuel. Table I shows the three essential nuclear reactions (Glasstone and Sesonske, 1967):
The nuclear reaction indicated by “step 1” shows, that a neutron absorbed by thorium-232 will bring about a transmutation to a new isotope, namely thorium-233 and emission of a gamma photon. Note that a logical source for the neutron required for absorption is a power producing fission reactor with the fertile thorium-232 contained in an annulus or blanket enveloping the reactor core. The thorium-233 isotope next (step 2) emits an electron (beta decay) as it rapidly transmutes to protactinium-233. With a half life of only 22.3 min over 99.9 percent of the $^{90}\text{Th}^{233}$ is converted into $^{91}\text{Pa}^{233}$ in 4 hr. In step 3 the protactinium-233 isotope itself undergoes a slow transmutation process by beta decay, with a half life of 27 days, there is a storage requirement or about 10 months for the protactinium-233 to decay to the fissile uranium-233.

**Molten Salt Reactor Technology**

The originators of this ‘fluid fuel reactor’ technology were nuclear researchers at ORNL, under the direction of Alvin Weinberg who served as director of ORNL from 1955 to 1973. The motivation for and the intended first application was in support of the Nuclear Aircraft project in the late 1940s under the ‘Homogeneous Reactor Experiment” (HRE) and the ‘Aircraft Nuclear Propulsion (ANP) project. Reactor outlet temperatures near 1100 K (820 °C) were achieved before the program was discontinued in 1961. However the technology acquired was shifted to a ground based civilian version of a “meltdown proof” reactor, serving as heat source for both a steam power plant and later for a CBC (Engel et al., 1980).

A schematic diagram of the ORNL-MSR Gas turbine power plant is displayed in Figure 2. Shown on the left side of the figure is a graphite matrix moderated MSR reactor with fuel salt mixture ($\text{ThF}_4$-$\text{U}_{233}\text{F}_4$) being circulated by a pump through the core and to a primary (shell-tube) heat exchanger. Note that a parallel loop permits part of the fuel salt to be diverted to a processing plant and reintroduced into the core as ‘purified salt’. - As one of the unique safety features, a melt-plug at the reactor bottom would permit the reactor fluid fuel to be drained into subcritical dump tanks, located in an underground storage facility, should the fuel salt temperature exceed a preset limit. A second pump circulates the liquid heat transfer fluid ($\text{LiF}$-$\text{BeF}_2$) through an intermediary heat exchanger where the helium working fluid is heated to turbine inlet temperature. The high pressure-high temperature He is shown to flow through two parallel turbines which drive two intercooled series compressors and the electric power generator, all mounted on the same shaft. The turbine exhaust flows pass through the hot side of a recuperator where thermal energy is transferred to the high pressure compressor discharge flow before entering the water cooled heat sink heat exchanger (HSHX) which lowers the working fluid temperature to the value required by the LPC (low pressure compressor) inlet condition. The compressor raises both pressure and temperature of the He working fluid before the fluid is cooled back to near inlet temperature by the intercooling He-water heat exchanger. Due to the lower temperature at the inlet of the HPC (high pressure compressor), the compressor work will be reduced significantly, thus allowing more shaft power for the generator and thereby leading to higher plant efficiency. As a final step in completing the circuit, the He working fluid exiting the high pressure compressor enters the cold side of the recuperator where it is preheated by the turbine exhaust stream. The helium then enters the secondary heat exchanger where it is heated to the turbine inlet temperature requirement as explained above.

Although not shown in the schematic, reactor core heat can also be used for $\text{H}_2$ production by processes like high temperature electrolysis of water, or the water gas shift reaction. Thus all of the objectives set forth under the Energy Policy Act of 2005 could be accomplished with advanced nuclear technology as represented by MSR or LFTR.
Technological Advantages of LFTR Power Plants

Compared to traditional nuclear reactors which “burn” the fissile uranium isotope U\textsubscript{235} the LFTR uses fissile U\textsubscript{233} which is derived from Th\textsubscript{232}. But whereas U\textsubscript{235} constitutes only 0.7 percent of mined natural uranium, practically all of the Thorium can be converted to U\textsubscript{233}, and no processing for enrichment is needed. As will be shown in a later section of this paper, at turbine inlet temperatures of 1200 K closed cycle gas turbine thermal energy conversion efficiency, $\eta_t$, of over 50 percent can be attained, as compared to a 30 to 35 percent efficiency for currently operating steam turbines plants with inlet temperatures of approximately 570 K (300 °C). Thus a factor of three hundred times as much output electric power per unit mass of raw fuel ore (uranium oxide (U\textsubscript{3}O\textsubscript{8}) versus Thoria (ThO\textsubscript{2}) can be obtained via the Thorium fuel cycle with closed cycle gas turbine energy conversion. As a result fission fragment waste products are reduced by a commensurate amount, and their radioactivity would decay to background levels in less than 300 years, as contrasted to over 10,000 years for currently used reactors, thus obviating the need for long term storage, such as at Yucca Mountain. The thermal spectrum LFTR concept is inherently safe, with a negative temperature coefficient of reactivity, thus making a “core meltdown” due to loss of coolant impossible. Since the fuel is a pumped liquid solution of LiF-BeF\textsubscript{2}-UF\textsubscript{4}, refueling can be accomplished without reactor shutdown. The fissile fuel can also be made “proliferation resistant” by permitting it to be contaminated (denatured) with small amounts of U\textsubscript{232} to increase its dose rate which would greatly reduce its unshielded exposure time and greatly increase detectability.

With Thorium ores, such as Monazite, being four times more abundant in the earth’s crust than uranium ores, over 60 percent of the world’s resources are located in the following democratically
governed countries: Australia (18 percent), United States (16 percent), India (13 percent), Brazil (9 percent), and Norway (5 percent). Thus future global energy demands could be met by these Thorium sources for over several tens of millennia.

Conceptual Design Modeling of Gas Turbine LFTR Nuclear Power Plants

Having established that Closed Cycle Gas turbine power plants with both directly or indirectly supplied thermal energy from nuclear heat sources would best meet the NGNPP-Gen IV power plant requirements, an author generated CBC code previously used in the modeling of space and planetary surface power systems (Juhasz 2005, 2006, 2007) was modified to meet the modeling requirements of terrestrial nuclear power plants. Special emphasis was placed on incorporating the two series heat exchanger requirements of LFTR–reactors as exemplified by ORNL MSRE technology.

Furthermore the provision for treating CBC compression and turbine expansion processes as composed of separate incremental series steps allowed for realistic modeling of power systems with compressor intercooling and/or turbine reheat options. Since cycle reject heat and intercooling heat transfer is accomplished via gas-water heat exchangers, the space radiator heat rejection sub-routines were bypassed in the modeling computations. Allowing for the working fluid passing through heat exchangers on the cycle hot side, recuperator, compressor intercooler and heat sink, the cycle pressure drop was set at 4 percent. Thus the turbine overall turbine pressure ratio for up to three series machines was 96 percent of the overall pressure ratio produced by the compressors. Provision was added to compute and display local pressure and temperature state points along the system schematic diagrams for moiling simulations for different cycle configurations, TITs and power output levels.

A list of key input values which were kept constant is shown in Table II.

<table>
<thead>
<tr>
<th>TABLE II.—KEY CYCLE INPUT PARAMETERS</th>
</tr>
</thead>
<tbody>
<tr>
<td>Compressor Inlet Temperature (TIC), K ........................................ 300</td>
</tr>
<tr>
<td>Cooling Water Temperature, K ...................................................... 288</td>
</tr>
<tr>
<td>Reactor Heat Loss, percent ............................................................. 1.0</td>
</tr>
<tr>
<td>Polytropic Efficiency—Compress, percent ................................. 86</td>
</tr>
<tr>
<td>Polytropic Efficiency—Turbine, percent ......................................... 92</td>
</tr>
<tr>
<td>Recuperator Effectiveness, percent .................................................. 95</td>
</tr>
<tr>
<td>Intercooler HX Pressure Loss, percent ........................................... 0.5</td>
</tr>
<tr>
<td>Reheat HX Pressure Loss, percent .................................................. 0.8</td>
</tr>
<tr>
<td>Turbine Pressure Ratio Fraction, percent ......................................... 96</td>
</tr>
<tr>
<td>Generator Efficiency, percent ......................................................... 98</td>
</tr>
</tbody>
</table>

Several conceptual power plant cycle configurations were modeled using the code briefly described above. As shown in Figure 3, the first of these is for a 1000 MWe power plant with turbine reheat and compressor intercooling (availability of water cooling reservoirs assumed), with a TIT of 950 K. With three series turbines and compressors, and the required heat exchangers on the hot side and cold side of the cycle, a fairly convoluted cycle schematic was analyzed. Note that the total He mass flowrate was only about 681 kg/s for this three series turbomachine configuration with an overall pressure ratio of 8, with each stage ratio of 2. The specific work parameter of 1468 kJ/kg expresses the ratio of total power output of 1000 MWe = 10^6 kJ/s to 680 kg/s. This flowrate is only a third of the over 2100 kg/s that would have been required for accomplishing the same power output with one large single compressor and turbine and the resulting specific work for this case would be less than 500 kJ/kg. So the system complexity is offset somewhat by much smaller rotating machinery and heat exchanger size. However another drawback is that, due to the cascading pressure levels the turbo-generator speeds (in rpm) optimize at 7200 for the HPT, 5400 for the MPT and 3600 for the LPT. This would require speed reducer transmissions for changing the intermediate and high pressure turboset speeds to 3600 rpm, for generation of 60 Hz electric power via 2 pole alternators.
The reactor thermal power is shown to be 2365 MWt, which indicates a plant thermal efficiency of 42.3 percent. Even after subtracting the approximately 3 MWe for combined pump power requirements the plant efficiency is still above 42 percent for the 950 K TIT, requiring a reactor outlet temperature of under 990 K, assuming high effectiveness heat exchangers. Of course, just like for the ORNL MSR system, the primary reactor fuel-coolant is uranium tetra fluoride (U$_{233}$F$_4$) which may also contain LiF-BeF$_2$ eutectic in solution. The secondary heat exchanger fluid is LiF-BeF$_2$ liquid salt with a melting point of ~ 630 K.

The next cycle analyzed, shown in Figure 4, has been greatly simplified by removing the reheat feature, but keeping the three series intercooled compressors. The TIT is still 950 K, but the output power level is reduced to 100 MWe. Note that the overall pressure ratio for this system is ~ 2.21 (i.e., 2.08 MPa: 0.94 MPa). Even though there is no ‘reheat’, the total turbine expansion work is split into two sections. The HPT (high pressure turbine) work is dedicated to driving the three series compressors with intercooling after the first and the second stage. The output of the low pressure turbine (LPT) at a TIT—834 K is used to drive the 100 MWe generator. Although the turbine speed still optimized at 7200 rpm, higher power output levels with higher machine diameters would lead to optimum turbine speeds near 3600 rpm. But to design a 3600 rpm turbo-generator for a 100 MWe output the operating pressure levels could be reduced, albeit the turbomachine diameters would thereby need increase.
Figure 4.—Schematic of 100 MWe liquid fluoride thorium reactor power plant with 950 K turbine (no reheat) and compressor intercooling. Plant efficiency is ~ 41.3 percent.

Note, that even without reheat the plant thermal efficiency only dropped about one percent. Compared to Figure 3, the number of hot side heat exchangers has been reduced from four to two. Such beneficial results with *intercooling only* were also pointed out in the reference literature (Frutschi, 2005). Note that the ‘*specific work*’ parameter has decreased to about 530 kJ/kg. This is indicated by the relatively high He mass flowrate requirement of 189 kg/s for this 100 MWe power output, when compared to the 681 kg/s mass flow for the 1000 MWe case of Figure 3. The primary and intermediate heat exchanger mass flows were computed on the basis of thermal capacity and density of the respective ‘liquid fuel’ (U$_{233}$F$_4$) and LiF-BeF$_2$ heat transfer fluids.
As shown in Figure 5, the last case analyzed was for a 100 MWe power output with the ‘intercool only’ option. But the TIT was increased to 1200 K, thus providing a cycle temperature ratio of 4. For this higher temperature ratio the plant thermodynamic efficiency increased to 50.5 percent and the overall optimum pressure ratio to 2.5. The specific work parameter almost doubled to 933 kJ/kg. This is also reflected by the reduced He mass flowrate of 107 kg/s. Note also that the high pressure turbine exit temperature, which is also the inlet temperature for the low pressure turbine, increased from 834 K for the case discussed in Figure 4 to 1061 K for this higher TIT.

The higher plant thermal efficiency and specific work values, coupled with lower working fluid mass flowrate requirement, reinforce the fact that higher peak cycle temperatures enabled by advances in high temperature materials technology are the key to achieving economies in lower heat input requirements and lower component sizes. These promising trends augur well for rewards in the future if required investments are made in the present.

The results of a systematic increase in TIT from 650 K to 1300 K for ‘Intercooled Only’ and ‘Intercooled + Reheated’ gas turbine systems are shown in Figures 6 and 7, respectively. The first case, illustrated by the red curves, represents configuration of three series compressors and a single turbine.
Figure 6.—Power plant efficiency as a function of turbine inlet temperature for only intercooled (one turbine) and intercooled + reheated (three turbines)—100 MWe CCGT power plant.

Figure 7.—Power plant mass flowrate as a function of turbine inlet temperature for only intercooled (one turbine) and intercooled + reheated (three turbines)—100 MWe CCGT power plant.
The second case, illustrated by the blue curves, is representative of the three reheated series turbines plus three series intercooled compressor case, as shown in Figure 3. The dramatic increase in power plant thermal efficiency from the low 20’s percent range for a TIT of 650 K, to over 53 percent at a TIT of 1300 K is shown in Figure 6. For the change in turbine inlet temperature shown the optimum pressure ratios increase from about 1.8 to 2.5 for the ‘Intercool only’ case. But for the ‘Intercool + Reheat’ case the optimum pressure ratios increase from about 2.8 to near 8.0 over the same temperature range. Due to the higher pressure ratios, the isentropic compressor efficiencies are lower for the same polytropic value of 86 percent as shown in Table II. This explains why the efficiency values for the ‘Intercooled only’ configuration surpass those for the ‘Intercool + Reheat’ at the high turbine inlet temperatures. But, regarding total helium mass flowrate, Figure 7 shows that while the mass flow decreases by a factor of ~ 6 over the temperature range, the three turbine ‘Intercool + Reheat’ configuration requires less than half the mass flow of the ‘Intercool only’ option. Since turbo-machine and heat exchanger size at the same operating pressure is proportional to working fluid mass flow, having larger sized components is the cost for reduction in system complexity offered by the single turbine ‘Intercool only’ option.

**Plant Sub-Marine Basing and High Voltage dc Power**

Although proposed here only in conceptual form with detailed designs to be generated at some future time, an LFTR-CBC power plant could be based off shore in a large submarine pressure vessel and the three phase ac power generated could be transformed to high voltage before rectification-conversion to HVDC and transmission via submarine cables to users in coastal regions. A preliminary electrical wiring scheme for power output from two parallel turbo-generators is shown in Figure 8.

The generator employed for the conversion process is an ac synchronous generator. The generated ac voltage is converted into dc voltage using a rectifier. A converter transformer is employed to step up the generated voltage to the necessary transmission voltage level. The converter transformer has two secondary windings, a delta and a Y-winding. This construction facilitates a 12-pulse rectification in the

![Figure 8.—Possible wiring for HVDC power transmission between source and load.](image_url)
rectifiers. The construction of the converter transformer also suppresses the fifth and the seventh harmonic in the system. The dc voltage output from the ac-dc rectifier has very low ripple and hence low losses. Smoothing reactors are employed to reduce ripples from the system in conjunction with dc. The layout of a HVDC transmission system is shown in figure below. The mechanical energy filters. This improves the power quality of the transmitted power. The smoothing reactor, dc filter also act as protective devices and reduce the current surges in the system, in case of a fault. The smooth, filtered dc voltage is transmitted via submarine HVDC transmission cable. The transmitted dc has to be converted back to ac voltage for transmission and distribution purposes. The dc is converted into ac by employing a low loss, high efficiency multilevel converter. The inverted voltage is stepped up or down depending on the voltage on the terrestrial grid. The layout shows the other protective devices in the system. Some obvious advantages of a HVDC over HVAC transmission system are:

1. Economically cheaper—two cables only
2. Efficiency as the system losses are lower
3. Reliability—underground cable location
4. Security—buried or underground cables are less prone to sabotage

The offshore basing of nuclear power plants would also make them more acceptable for location within a few miles of metropolitan centers.

Concluding Remarks

Conceptual designs for ground based gas turbine energy conversion power plants with advanced nuclear fission reactor heat input were analyzed using an author generated code with the capability to model gas turbine power systems with compressor intercooling and/or turbine reheat provisions. It was shown that, given high quality heat exchanger and turbomachine technology with 1200 K inlet temperature, power plants with a thermodynamic efficiency of 50 percent could be constructed.

In particular a nontraditional nuclear fuel, namely uranium-233, derived from natural thorium, nuclear power plants using a Liquid Fluoride Thorium Reactor (LFTR) would offer great benefits for ensuring future energy supplies, reduction of adverse climate effects due to greenhouse gas emissions, and invigoration of the world wide economy. With the inherently higher proliferation resistance of the Thorium fuel cycle LFTR’s meet the requirements of the Gen IV nuclear power plants as spelled out in the Energy Policy Act of 2005.

As confirmed by an author generated CBC code, even without the complexity of turbine reheat cycles, using ‘intercooled only’ option, at least 50 percent of the thermal energy from (LFTRs) could be converted by gas turbine driven generators (operating at ~1200 K turbine inlet temperature) for electric power production during peak demand periods. Both thermal and electrical energy would be available during “off peak” periods for hydrogen production by elevated temperature electrolysis of water or chemical processes such as the water gas shift reaction. This approach would both supply electric power by using environmentally clean nuclear heat which does not generate green house gases, and it would also provide a clean fuel for the future, when, due to increased global demand and the decline in discovering new deposits, our supply of liquid fossil fuels will have been used up within the next 30 to 50 years, as predicted by the Hubbert model and confirmed by other global energy consumption prognoses.

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